# THEORETICAL — EXPERIMENTAL INVESTIGATION OF SAW — TOUTH FOLDED PLATE STRUCTURES

MSc. THESIS

BY
AMIN SALEH ALY
B. Sc. CIVIL ENGINEERING 1970

FACULTY OF ENGINEERING
CAIRO UNIVERSITY

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### SYNOPSIS

Both theoretical and experimental investigations on the structural behavious of the saw-tooth folded plate are presented in this thesis. Two models are used to check the theoretical results. The first model is an aluminum model. The strains have been measured by S.R.4 strain gages at different points. The second model is reinforced concrete one. Its proportions are six times that of the aluminum model. It is subjected to both symmetrical and unsymmetrical loadings.

Investigations show that Gaafar's method which is recommended in 1961 by the ASCE Committe on folded plate structures agrees well with the experimental results. Also it is found that for some special loadings where the resultant of loads coinside with the shear center of the cross section; Gaafar and the beam methods give the same results.

A more general solution which is recommended when solving a certain problem for different cases of loading is given. The solution is based on Gaafar's method, with a proposed simplification in the calculations.

The finite element method is also used, and a general computer program is published. The maximum core size of the available computer being small, it is not sufficient to obtain more steady results for space structures specially when these structures are unsymmetrical. The method is clearly explained and the aluminum model is taken as an example. The data as well as the results associated to the example are printed to facilitate the analytical procedure.

### NOTATIONS

```
Cross sectional area of plate "ab"
           = Subscripts indicating the name of the fold lines
           = Plate depth
           = Modulus of elasticity
E
           = The algebraic difference between the edge
H
             stresses at a joint.
           = Stress distribution factor.
K
L
           = Longitudinal span.
           = Transverse Bending Moment.
P
           = Harmonic joint load due to relative joint
             displacement.
             Thickness of plate
             Moment of inertia.
I
             Normal stress.
Sab
          = Component of displacement of plate ab in
            its plane.
          = Relative displacement between two successive
            joints
                                 transvers direction.
          = slab moment in the
m
W
          Total Jack load.
          Free edge plate moment at the mid span section
Mo
0
          = Angle of inclination of the principal axes.
```

Other notations are given where they are used.

### CHAPTER

I

FOLDES PLATE STRUCTURES

OF

HOMOGENOUS MATERIALS

### I - INTRODUCTION

A "hipped plate" structure or "folded plate" structure is a three dimensional assembly of plates. These plates are arranged as to produce a stable structure capable of carrying loads. The essential feature of such structure is that the individual component element is flat, not curved, so being of considerable simple framework, it has been widely used in various types of constructions such as, long span roofs, saw tooth roofs, bridges, towers, channels, silos, bunkers, etc.

According to their external shape, hipped plate structures may be distinguished as being prismatic, pyramidal, prismoidal and curved on plane. The prismatic hipped plate structures are composed of rectangular plates connected along their edges, and having diphragms perpendicular to the longitudinal axis at supports. Pyramidal hipped plate structures occur as pavilion roofs. Prismoidal hipped plate structures are an intermediate form of construction between prismatic and pyremidal roofs. Also, if only two plates intersected at a junction, the structure is termed a "simple" hipped plate. And if more than two plates intersected at a junction, the structure is termed a "multiple" hipped plate.

II.

Considerable saving are obtained by this method of construction as compared with standard beam-and-girder design. In addition to the very definite saving of materials, the smooth surface, uninterrupted by beams, ribs and other protections, make for simple framework and erection and convenience in use. In conventional construction, beams would be provided at all junctions of adjoining slabs. It is simple to show that such beams are not only superfluous but also ineffective provided the angle between adjacent plates is not too close to 180°.

The simplest form of folded plates consists of the inclined plates in V-shape. However, this cross section has a disadvantage, in that the area of concrete to resist the compressive flexural force or to permit placement of the reinforcing steel may be inadequate. In Addition, at the end span, the legs of the V act as long contilevers in transmerse bending. A more general form is developed by adding horizontal plates at the top and bottom junctions of the inclined plates.

Floded plate structures are well suited to the application of prestressing. Prestressed folded plate structures

ely, reduced possibility of cracking, smaller deflections and smaller hight to span ratios. Also the
precasting of folded plates effer the possibility of
vast new field of applications and the prestressing
can be done on the precasting bed with the possibility
of additional economic advantages.

A very useful application of the folded plate is to north-light or saw-tooth roof construction, the window area being incorporated into the effective slab to provide greater height to span ratio. This system suits well the big covered halls where a uniform distribution of natural light, that is not possible form windows in the outside walls, is required. Any reinforced concrete saw-tooth roof lies under one of the following two headings:

1 — Slabs(solid or hollow), beams and girder constructions 2 — Beamless roofs hipped as conoids, straight as folded

Comparing between the curved saw-tooth slab, of the shell form and the folded saw-tooth slab, it can be concluded that the curved slab is more convenient for the

plates.

better distribution of stresses as the compression zone is bigger and this increases the stability of such roofs by reducing the compressive stresses. While in the folded slab the compressive stresses concentrate at the upper and lower corners of the fold. From the constructional point of view, the curved slab usually needs high standard of supervision beside higher cost for the shuttering.

### 2- ASSUMPTIONS

The following general assumptions are oftenly used in analyzing a folded plate structure:

- 1- The material is homogeneous, uncrecked and elastic
- 2- Longitudinal edge joints are fully monolithic and contineous.
- 3- The principle of super position holds, ie. the structure may be analyzed for the effects of its redundants as well as for the various external loadings. The results in both cases will then be super posed.
- 4- Individual plates posses negligible tersional resistance
- 5- The function of the supporting diaphragms is to supply the end reactions for the plate action. They are assumed incapable of providing restraint against rotation of the plate ends in their own planes.
- 6- The longitudinal strain, due to plate action varies linearely across the width of each plate.

# 3- PRINCIPIES OF ANALYSIS

Loads applied to the structure are resolved into their components perpendicular and parallel to the respective surfaces on which they act. The perpendicular components result in slab action whereas the parallel

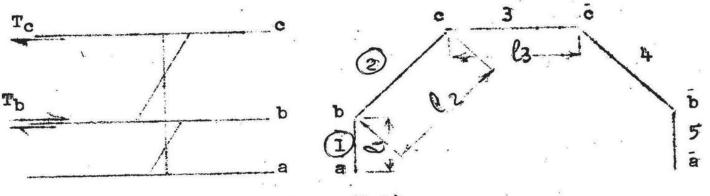
components are resisted by plate action of the respective members. With respect to the slab action, the structure is analyzed as a contineous, one way slab supported at the edges. The deformation of the edges of plate structure, where the plates are joined, is possible only if the plates are deformed in their planes. These edges can therefore be considered to be stiff and immobile supports for the transfer of loads in the cross direction.

OP

The reactions of the contineous slab at each edge support are resolved into components which lie in the planes of the neibouring plates. These components represents slab loads which subject the plates to bending. The bending moments and shear forces in each plate are determined as for deep beams. Thus the same structural members are first designed as slabsfor bending out their planes, and then as paltes for bending in their planes. The monolithic action along the common joints of adjacent plates prevent relative displacements there. Shear forces

Hince, each of the plates, in addition to its normal load P., causing bending moments M., is acted upon along

its two edges, by shear forces"T". For the same reason of continuity, the stresses in two neighbouring plates must be equal along their common edge, e.g. 25 = 1b f2b f2b



$$f_{2b} = \frac{6M_{02}}{t_2 f_2^2} - (\frac{4T_b + 2T_c}{t_2 f_2}) \cdots (1)$$

$$f_{1b} = \frac{6M_{01}}{t_1} + \frac{t_1}{t_1^2} + \frac{t_1}{t_1} \frac{\mathbf{f}_b}{\mathbf{f}_1}$$
 (2)

from (1) & (2) we have

$$\frac{1}{2} \left( \frac{M_0^2}{z_1} + \frac{M_0^1}{z_1} \right) = 2 T_b \left( \frac{1}{z_1} + \frac{1}{z_1} \right) + T_c \left( \frac{1}{z_1} \right)$$

$$\frac{1}{z_1} \left( \frac{M_0^2}{z_1} + \frac{M_0^1}{z_1} \right) = 2 T_b \left( \frac{1}{z_1} + \frac{1}{z_1} \right) + T_c \left( \frac{1}{z_1} \right)$$
(3)

Equation (3) is similar to three moment theorem for contineous beams. Such an equation can be derived for each edge, and the shear forces "T" are determined from

the total set of equations. When the moments M. and the shear forces Ta, Tb, etc., are determined for all plates of the structure, the first part of the analysis is considered to be completed. The second part of the analysis is concerned with the effect of the relative displacements of the joints, this relative displacements the creates secondary loads which the structure should be analysed for it again. The consideration of the relative displacements of the joints may in many cases affect the results seriously.

# 4- STRESS DISTRIBUTION METHOD

Equation (3) is similar to the three moment equation. Winter & Pie had modified it to a stress distribution method which is similar to the moment distribution process developed by Hardy cross.

In the moment distribution method, an immaginary locking is assumed at each suport, which when released the balancing moment is distributed over the adjacent spans; similarly in the stress distribution process, immaginary non-sliding joints are supplied at the common edges of each plate element. The correction of such case

is obtained by the application of a balancing edge shearing force T, the stresses of which are distributed over the adjacant plates .

ent in the Hardy Cross method is that the angles of deflections for the two adjacent moment should be the same over the common support. Again the condition of stress distribution in the Winter & Pie method is that the unit strains should be the same for the two adjacent plates along their common edge.

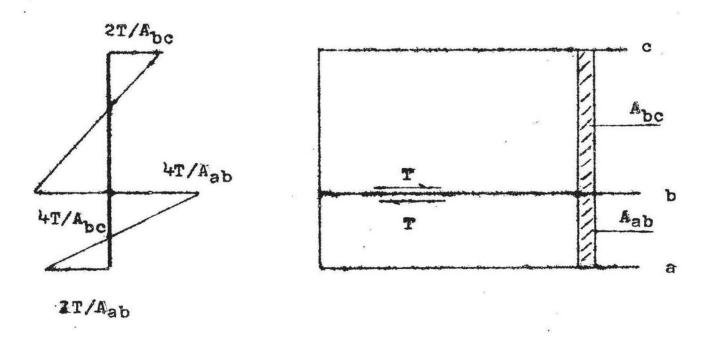


Fig. (1.2)

Fig.(1-2) shows the effect of two equal and opposite corces T on the adjacent plates. If the algebraic difference between the unequal free edge stresses at any joint "b" say, is equal to "", therefore the balancing shear applied to remove the sliding joint is equal to "H", and the part of this value transmitted to plate a-b is equal to:

$$fab = \frac{4T}{Aab} = + \frac{(4T/Abg)}{(4T/Aab + 4T/Abc)}$$

= 
$$\frac{A_{bc}}{A_{bc} + A_{ab}}$$
 = Kabx(-H)

Similarly, the stress transmitted to plate b-c is equal to:

$$\mathbf{f}_{abc} = \frac{A_{ab}}{A_{ab} + A_{bc}} = Kbcx(-H)$$

where K is called the "stress distribution factor"

Again it is clear from the same fig. that the development of a stress fab at edge "b" of the plate a-b creates at the other for edge "a" a corresponding stress equals to  $-\frac{1}{2}$  fab for rectangular cross sections.

calling the value - 1/2 as the carry over factor, the steps adopted for the stress distribution process are carried out as follows

- 1- Calculate the free-edge stresses due to the plate loads alone as equal to # Mo/Z.
- 2- Calculate the balancing stress "-H" at each joint, and distribute this stress over the two adjacent plates according to the distribution factor K.
- 3- Transfer the carry-over stresses to: the far edges of the plates .
- Repeat the above process using the transferred carryover stresses as inetial free edge stresses untill
  satisfactory convergence is obtained.

# 5- HISTORICAL REVIEW

The principle of hipped plate construction was first developed by G. Ehlers in Germany in 1924. He wrote the first technical paper on this subject in 1930. In his method of analysis he considered the various plate elements as beams supporting at the cross and end diaphragms.

Along the longitudinal edges, the plates were assumed to be connected by hinged joints that do not slide longitudinally and that are considered capable of transferring only

Such connections neglect entirely the connecting moments transmitted between the plates due to the rigidity of the joints. The uniform loads on the plates were transformed to the line loads "P" acting at the joints. These loads "P" were resolved into two components, Pcc and Pcb, parallel to the two adjacent plates. The plates, acting as beams between the diaphragms, carried the loads P.

At the same time, the shear stresses T were created to maintain equal longitudinal strains along the common edges.

This strain condition at each joint was used to determine the magnitude and distribution of the edge shear stress T.

b b a a

In 1932, E. Gruber published a paper in which he considered the effect of the rigiditly of the joints, the connecting moments acting along the common edges of the plates, and the effect of the relative displacements between the joints. As a first approximation, the hipped roof was assumed to be hinged along the joints. By Using this assumed hinged structure as a basic system, he developed his solution in the form of simultaneous differential equations of the fourth order, which could be

solved by rapidly converging series. For a hipped roof of r+1 plates, r being the number of joints, the total number of unknowns is 7r\*2, for a roof of five plates, this will involve thirty unknown. The solution is complicated even if trigonometric series are used. In this solution, Mr. Gruber showed that the maximum longitudinal stresses on a cross section and the maximum deflections for a roof with hinged plates were about twice as great as those for the rigidly connected plates. Consequently he concluded that the influence of the rigid connections ought not to be neglected as it had been according to the usual practice, at that time.

Vlassow in 1936 determined the values of the stresses at the critical section by the solution of a set simultaneous linear algebraic equations that were established on the basis of equilibrium at the critical section and on the basis of continuity of joints transversely.

Longitudinal variation of applied loads of transverse moments were approximated on the basis of a fourier series. The inaccuracies in the results, especially at sections other than the critical section depend on the number of terms employed in the fourier series.

The number of simultaneous equations involved were, in general 2n-2 in which n is the number of plates.

Sheek

As many as eight unknowns may be present in each equation and a fair degree of computational effort is required for solution.

In 1947, Messrs Winter and Pei of Cornell University at Ithaca, N.Y., U.S.A. published paper on hipped plate construction in which they transformed the algebraic solution into a stress distribution method, which has the advantage of numerical simplicity over the other procedures. However, they also made the same simplifying assumption of neglecting the effect of the relative deflections of the joints.

In 1948, Mr. Pei presented a method of analysis considering joint displacements. The method requires the solution of 6n+1 simultaneous algebraic equations in which n is the number of plates. For a roof of five plates, the number of equations is thirty-one.

Non of the mathematical investigations previously mentioned gives any experimental evidence to substantiate the assumptions and verify the analytical procedures that were used.

In 1949, Prof. Gaafar of the Cairo University, Cairo, Bgypt presented his thesis" The Analysis of Hipped Plate Str. considering the Relative Displacements of Joints".

He proved analytically, by his method, and experimentally that the relative displacements of joints strongly affect the normal stresses. He concluded from his work that:

- 1. When the relative transverse displacements of the joints are not taken into consideration, the stress values are so different from the experimental values (if the stresses are computed by Winter and Peimethod).
- 2. On the other hand, when the relative displacements of the joints are considered, the values of the longitudinal stresses agree much better with the experimental results.
- 3. The Winter and Pie method may not even predict the right sence of the longitudinal normal stresses.
- 4. The structure under loading does not behave as a unit like that of the beam theory.

Prof. Gaafar has chosen the relative transverse displacement between any two consecutive joints, as an unknown, and is treated as an additional load on the roof. These displacements will affect the slab moments, shears, and consequently, the plate loads P, and the plate deflections, S. For a loading that is symmetrical about the

maximum at the middle of the span and are zero at the end diaphragms. The general form of the curve is known but its exact equation is not known. He considered a sine curve elastic line as an approximation for the actual one, and proved that the error in that assumption does not exceed 2%. Such use of the sine curve greatly simplifies the analytical treatments. In the case of multispan roofs, or roofs on which the loads are for from being symmetrical about the middle of the span, this sine wave treatment can not accurately be used, and the elastic line of the corresponding loaded beam would have to be adopted.

The steps used by Gaafar are as follows:

1. The first step in the analysis is the computation of the forces and of the transverse and longitudinal stresses acting at the edges of each plate element, neglecting the effect of the relative displacement of the joints. The analytical procedure used by Messrs. Winter and Pei provides a convenient solution for this problem. In this procedure the roof in the transverse direction is considered as a continuous one-way slab supported on rigid supports at the joints, and thus the shear forces Q are readily obtained.

The Q-forces at each joint are resolved into two components P-forces parallel to the continuous plates. The plates, acting as beams between the end diaphragms, carry the P-loads (plate action). At the same time, edge shear stresses (T) are created along the edges to maintain equal longitudinal strains along the common edges. The longitudinal plate stresses at sections of the roof, caused by the P-forces only, are corrected by the stress-distribution method to include the effect of the edge shear stresses T.

effect of the relative transverse displacement of the joints on the transverse and longitudinal stresses. This operation is most easily accomplished by choosing the relative transvrse displacement (\$\triansled{\sigma}\$) between each pair of consecutive joints as unknown, determining the corresponding transverse (slab) fixedend moments in terms of the \$\triansled{\sigma}\$ values, and correcting for rotation at the ends of the slab strips by the moment-distribution method. After the end moments have been determined, the Q-forces and P-forces are computed in the same manner as in step 1. This operation must be repeated for each different value.

- 3. After the Q-forces are found, the corresponding longitudinal (plate) stresses are computed in the same manner as in step I.
- 4. From the values of the longitudinal stresses yielded in steps I and 3, the plate deflections \$\frac{1}{2}, and \$\fra
- 5. After the Δ-values have been computed, the slab moments and shears, and consequently longitudinal (plate) stresses producted by these transverse displacements, are known because they have been previously (steps 2 &3) expressed in terms of the Δ-values. These values are added algebraically to the corresponding values in step 1 to give the final results.

Later Mr. Brielmaier published a paper about the enalysis of folded plate structures, he suggested a solution for such roofs by successive approximations. He concluded that "The change in the slab mometnts due to the plates" deflections may be large and should always be considered. This large change in the slab moments may result in an appraciable change in the slab reactions and hence in the plates stresses which in turn will change the plates deflections.

The computations for the longitudinal stresses and the transverse deformations, may be carried out by successive approximations starting in each step with the stresses and deformations yielded in the preceding one. In many cases; according to Brielmaie, the first correction for the plates deflections is sufficient. In some other cases, Several corrections are required, and in some the corrections are large and oscillating or converging slowly. He stated that when the change is not evaluated adequately by one or two corrections, it might be advisable to correct by using equations, which permit the direct solutions for final edge moments.

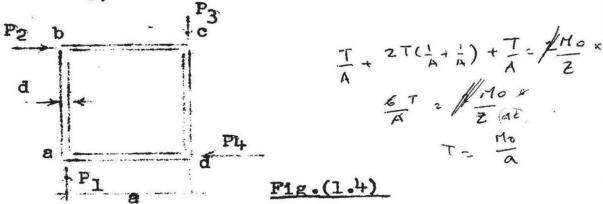
In the lest few years, there has been some published papers about the use of the electronic digital computer to facilitate the analysis of the folded plate structure. The computer is used to solve numerous simple equations which are handeled by the matrix algebra. The digital computer method makes the designer tied and dependent on the computing center.

## 6- SPECIAL TOPICS

A-Theory of Hipped plate Edrustures and Torsion

The theory of hipped-plate structures provides a mean of taking various problems associated with the torsion of hollow sections ("box"sections), L-sections, T-sections, etc. A few relevant examples will be given.

Cosider the prismatic hipped-plate structure represented in cross-section in Fig. 1.4. Its paltes are assumed to be similar in respect of their conditions of support and loading, so that all the Mo diagrams



are affine. The structure is subjected to four point loads  $P_1 = P_2 = P_3 = P_4 = P$  which produce a torsional moment  $M_D = 2P$ .a. The section modulus Z is assumed to be the same for all the plates. The shears T acting at the junctions A,B,C and D will then all be equal in respect of magnitade and direction. They can be computed from the equation:

$$T(\frac{1}{A}) + 2T(\frac{1}{A} + \frac{1}{A}) + T(\frac{1}{A}) = \frac{1}{2} \left(\frac{7}{2}, \frac{7}{4}\right)$$

$$\frac{6T}{\alpha t} = \frac{67}{t \alpha^{2}}$$

$$\frac{6T}{2} = \frac{17}{4}$$

$$\frac{17}{2} = \frac{17}{4}$$

$$\frac{17}{2} = \frac{17}{4}$$

$$\frac{17}{4} = \frac{17}{4}$$

$$\frac{17}{4} = \frac{17}{4}$$

$$\frac{17}{4} = \frac{17}{4}$$

The relieving moment developed by the shear forces T, which acts in opossition to the moment Mo, will then be:

and the final moment will be

$$M = M_0 - M_T = 0$$

Hence in this case the structure is subjected to pure tossion, the stresses that occur in the component plates of the structure are shear stresses, not flextural stresses.

Another loading case is represented in Fig.1.5. The forces P2 and P4 acting on the structure will likewise produce a torsional moment, equal P.a.

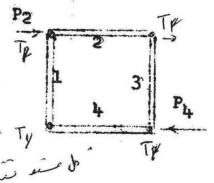


Fig.(1.5)

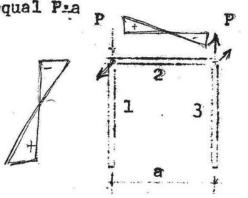


Fig. (1.6)

The equation of three shears gives in this case

$$6T = \frac{A}{Z} \cdot \frac{M_0}{2} = \frac{M_0}{2K} = \frac{M_0 \times 6}{2 \times a}$$

$$T = \frac{M_0}{2 \times a}$$

The relieving moment is:

$$M_T = T.a = \frac{M_0}{2}$$

This moment is only half as large as the applied moment Mo. So the structure is subjected to doubly antisymmetrical state of stress.

The next example is represented by Fig.1.6. The junction shear is

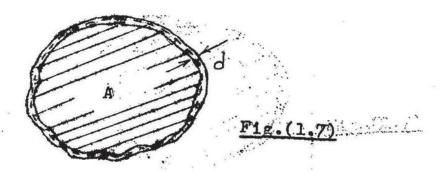
$$5 T = \frac{A}{Z} \times \frac{M_0}{2}$$
 .  $T = \frac{6}{10} \frac{M_0}{a}$ 

therefore

$$M_{T2} = \frac{6}{10} M_0$$
 $M_2 = -\frac{6}{10} M_0$ 
 $M_{T1,3} = \frac{3}{10} M_0$ 
 $M_{1,3} = \frac{7}{10} M_0$ 

Again flextural stresses occur in addition to shear stresses.

The magnitude of the shear stresses can also be determined using theory of the hipped plate structure. Before doing this it will however, be interested to determine them with the aid of torsional theory for hallow bodies.



Applying Bredt's theorem on the shown Fig. we get:

$$\frac{Md}{AS} = \frac{Md}{AS}$$

where "Md" is the applied torque

"A" is the area inclused by the center line of the section of the wall of the hallow body.

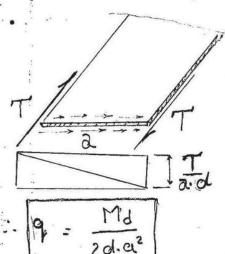
For the case of ex. I;

$$d = \frac{5xqxa_{5}}{Mq}$$

Applying hipped plate theory :

· . :

$$T = \frac{M_d}{2.a}$$



ssuming this force to be uniformly distributed over he length a then:

q = Md ie the same as obtained before

Also it can be proved that the longitudinal shear stresses have the same magnitude of the

transverse shears. As shown in

Fig.(1:8)

$$P = \frac{Md}{2a}$$

$$P.X = \frac{Md}{2a} = x$$

 $T = \frac{Mo}{a} = \frac{Md}{2a^2} \times \text{(from Eq. of 3 shears)}$ 

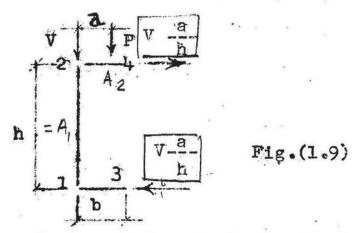
But 
$$dT_n = d \cdot q_n \cdot dx$$
 therefore  $-\frac{dT_n}{dx} = d \cdot q_n$ 

Thus we obtain 
$$q_n = \frac{1}{d} \frac{dT_n}{dx}$$

Thus we obtain  $q_n = \frac{M_d}{2d \cdot a2}$ 

The same as transverse shear stresses.

ie. the longitudinal shear stresses are of the same magnitude as the transverse shear stresses.



Consider the channel-shaped hipped -plate structure

shown in Fig(1.9) loaded by a vertical load P which produces the lateral forces:

$$T = \frac{M}{h} \frac{1 - \frac{a}{b} \times \frac{A_1}{A_2}}{1 + \frac{2}{3} \cdot \frac{A_1}{A_2}}$$

$$f_1 = -f_2 = \frac{M}{h} - \frac{2 + 3 - \frac{3}{h}}{\frac{A_1}{3} + \frac{A_2}{2}}$$

$$f_3 = -f_4 = -\frac{M}{h} \frac{1 + \frac{3a}{b} + \frac{a}{b} - \frac{A_1}{A_2}}{\frac{A_1}{3} + \frac{A_2}{2}}$$

For a = 0

$$f_1 = -f_2 - \frac{2M}{h(-\frac{A_1}{3} + \frac{A_2}{2})}$$
 and  $f_2 = -f_4 = -\frac{M}{h(-\frac{A_1}{3} + \frac{A_2}{2})}$ 

This shows that the stresses are not proportion to the distance from the neutral axis.

To satisfy this assumption, the load P must be applied in such a way what we obtain:

$$f_1 = f_4 = -f_2 = -f_3$$

The corresponding distance ao is determined from the equation:

$$2 + 3 \frac{ao}{b} = - (1 + \frac{3}{b} + \frac{aoA1}{bA2})$$

ao = 
$$-\frac{3b}{6+\frac{A_1}{A_2}}$$

When the load passes through this point-which is located on the horizontal axis of symmetry and is called the shear center - the stresses are

$$f_1 = f_4 = -f_2 = -f_3 = \frac{M_1}{h(A_2 + \frac{A_1}{6})}$$

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# B. Stairs as Mipped-Plate Structures

Design of stairs as hipped plate is characterised by the saving of materials. It is most advantageous in cases
where the walls of the staircase are of reinforced concrete
which is often adopted in industrial and tall buildings.

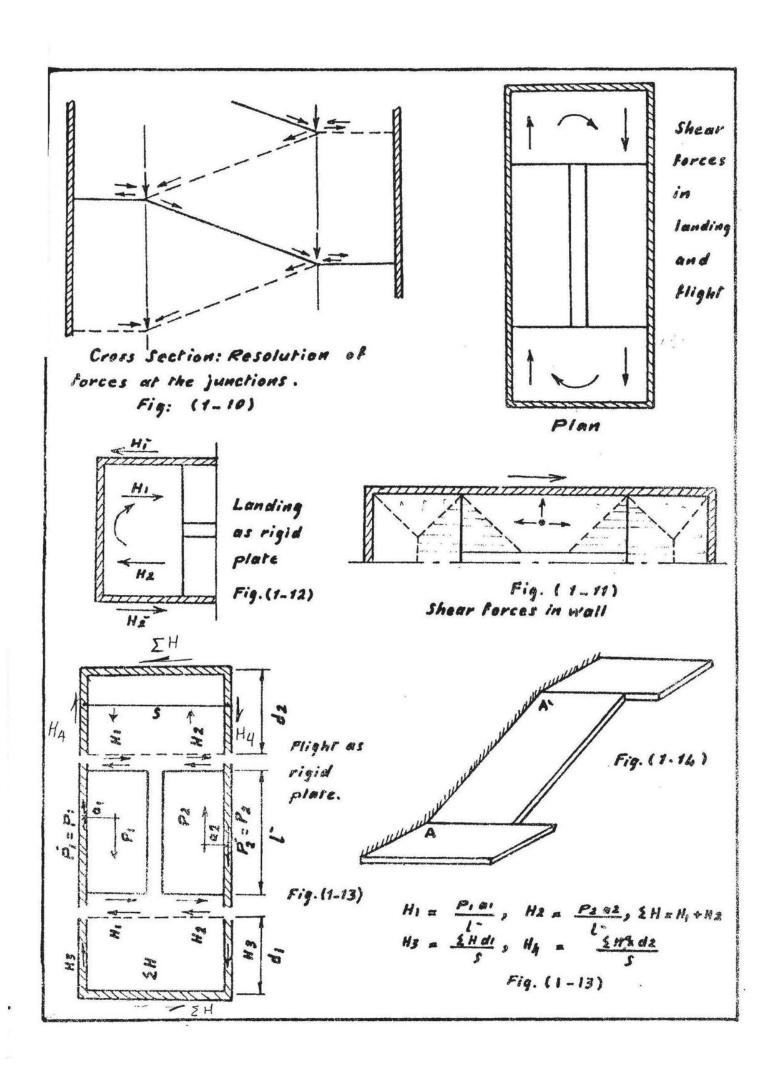
As shown in Fig(1:19)the loads are considered as being transmitted to the junctions of the plates, where they are resolved into forces directed in the planes of the plates intessecting there.

The horizontal shear forces drising from the upper and from the lower flight act upon the landing in opposite difections and consequently exercise a twisting moment upon it. In this connection the landing may be regarded as a plate supported on three sides and is capable of resisting this moment.

A very favourable arrangement is to support the flight of steps on the wall of the staircase or, better still, to build it into the latter. In this case only the loads corresponding to the shaded areas in Fig.(1.11) will be transmitted to the junctions of the hipped-plate

structure. Consequently the horisontal shear in the landing will be smaller in magnitude than if the flight were not supported on the walls. The flight also will act as a plate supported on three sides. The force P acting in the plane of the flight is in equilibrium with the shear force P in the staircase wall. The moment developed by the couple formed by P and P is in turn resisted by the couple H.H. These forces H and H produce a moment that acts upon the landing and reduces the above mentioned twisting moment due to the horisontal shears.

The wall enclosing the staircese must be able to take up the shear stresses that occur. In case of the absence of this wall, the load P in the plane of the flight will be transmitted in equal portions to corner points A and A of the junction between the flight and the upper and lower landing respectively. Whence the forces are transmitted directly to the wall by the built-in landings. For structureal reasons it will then be necessary to provide special reinforcement at the corners A and A (especially shear reinforcement).



CHAPTER

I

THE ALUMINUM MODEL

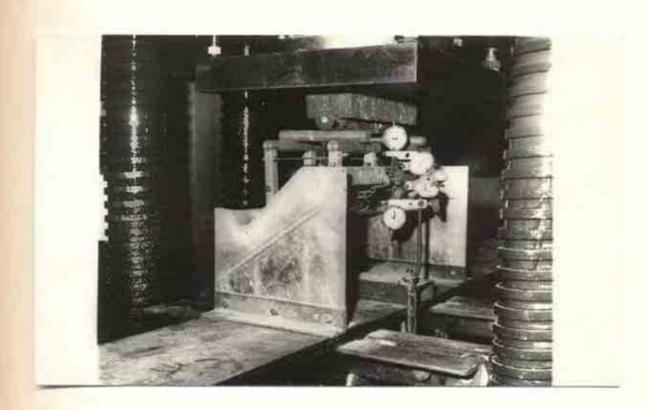
### 1- DESCRIPTIVE OF THE MODEL

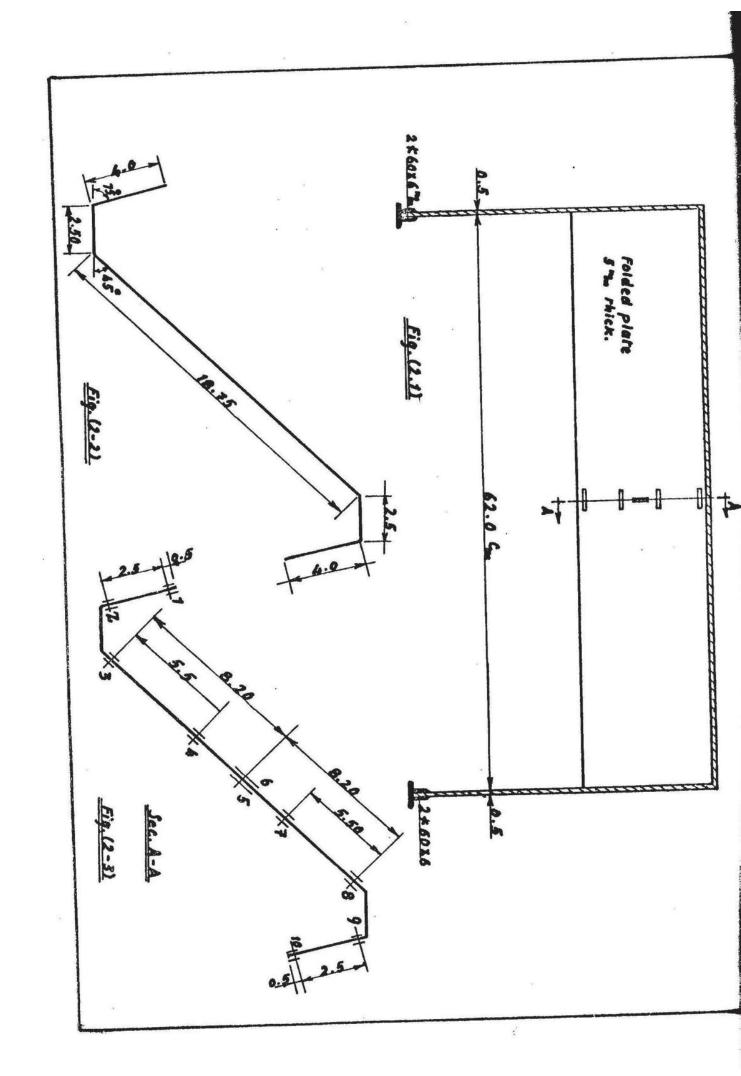
An aluminum model is used for the experimental investigation of strains in a saw-tooth folded plate structure. The dimensions of the model are shown in Fig.(2-1) and Fig.(2-2). The loads are applied at sixteen paints on the model Fig.(2-4) and Fig.(2-5). S.R.4 resistance gages are placed to measure the longitudinal strains at the middle section Fig.(2-3). There gages are placed on both sides of each plate. and connected together in series to give the average strain. The transverse strains are measured at one point on both sides by means of S.R.4 resistance gage. Also the vertical and horizontal displacements of joints are measured by means of Dial Gages Fig.(2-6).

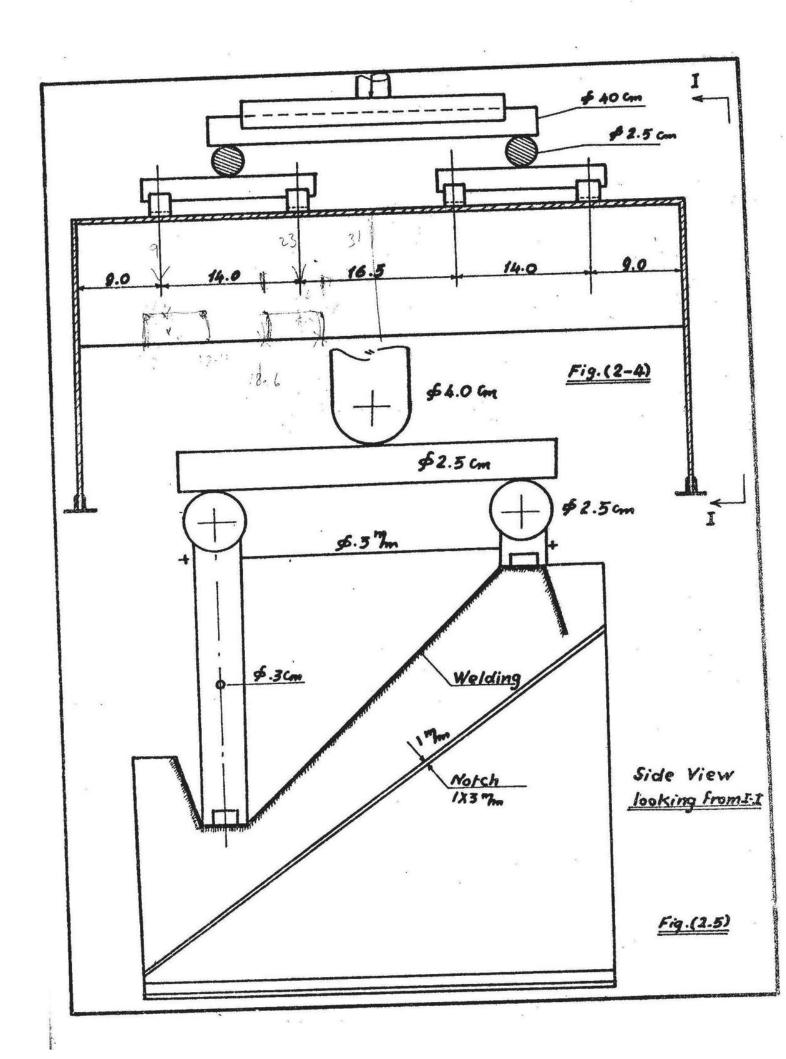
Fig. (2-5) and Fig. (2-7) show that the model is provided by two end diaphragms welded to the superstructure. The diaphragms are made also from the same material of the poof construction and having the same thickness. Each diaphragm is connected to the tested machine during the test to add an extra mean of stability.

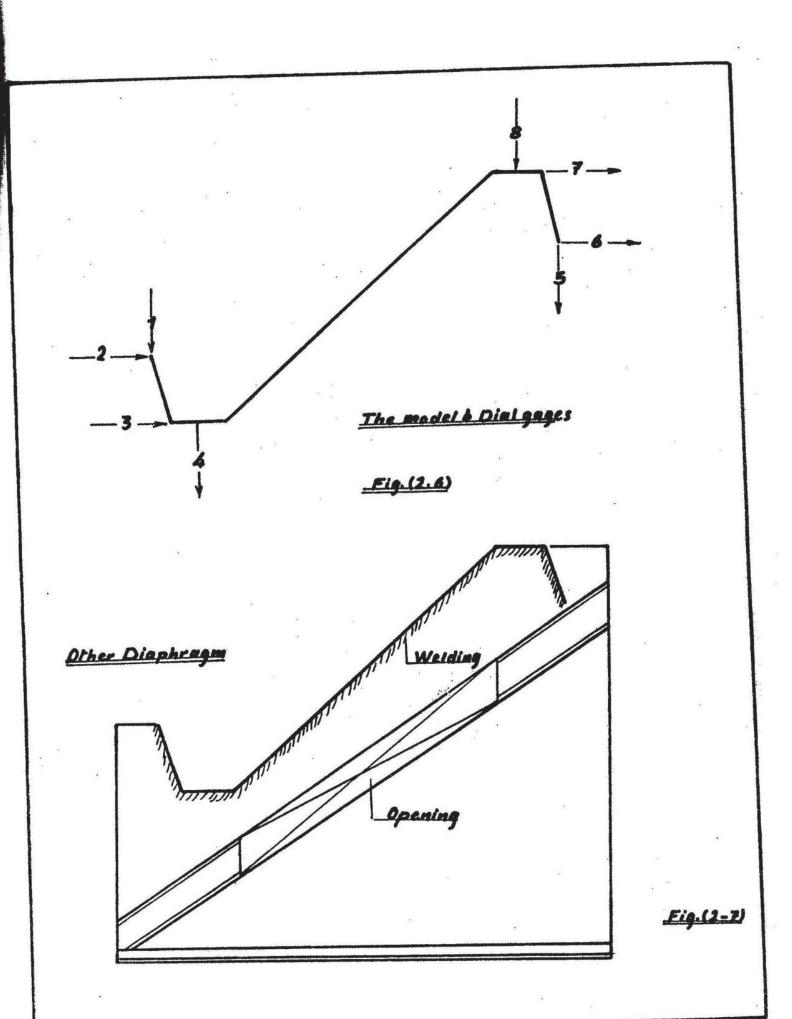
In order to have a truely simply supported condition one of the two diaphragms are provided with a notch parallol nearly to the expected neutral axis.







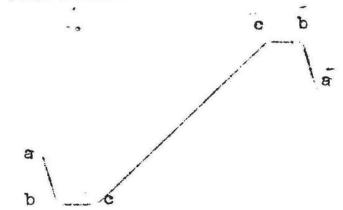




### 2- ANALYSIS OF THE MODEL (GAAFAR'S METHOD)

### a. Elementary Analysis:

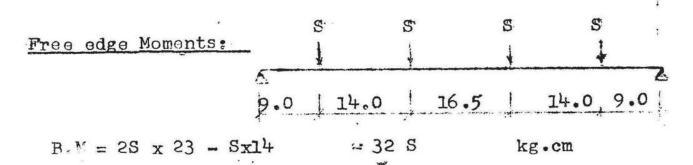
- Properties of the Diff rent Plates



$$L = 62.5$$
 cm

$$t = 0.50$$
 cm

Plate	Dim. Cm.	Area Cm <sup>2</sup>	$Z = \frac{12 \times t^3}{6} \text{ cm}^3$
AB, AB	4 x 0.5	2.0	1.33
BC,ĒĈ	2.5x.5	1.25	0.52
cc	18.75x.5	9.375	29.2



### - Plate Loads, bending moments and

the corresponding stresses at middle section:

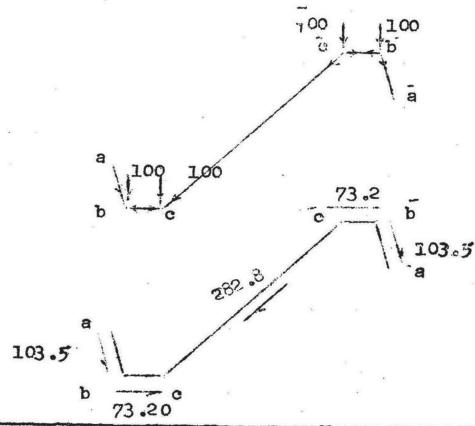


Plate	loads	B.M=325	Z	$\sigma = \frac{M}{Z}$
AB, AB	103.5	3340	1.33	2510
BC,BC	73.2	2340	0.52	4500
с <del>с</del>	282.8	9050	29.2	310

### Stress Distribution Process

Distribution factors at point "B"

$$k_{ba} = \frac{A_{bc}}{A_{bc} + A_{ba}} = \frac{1.25}{1.25+2} = 0.385$$
  $k_{bc} = 0.615$ 

at point "C"
$$k_{cb} = \frac{9.375}{9.375 + 1.25} = 0.88$$

$$k_{cb} = 0.12$$

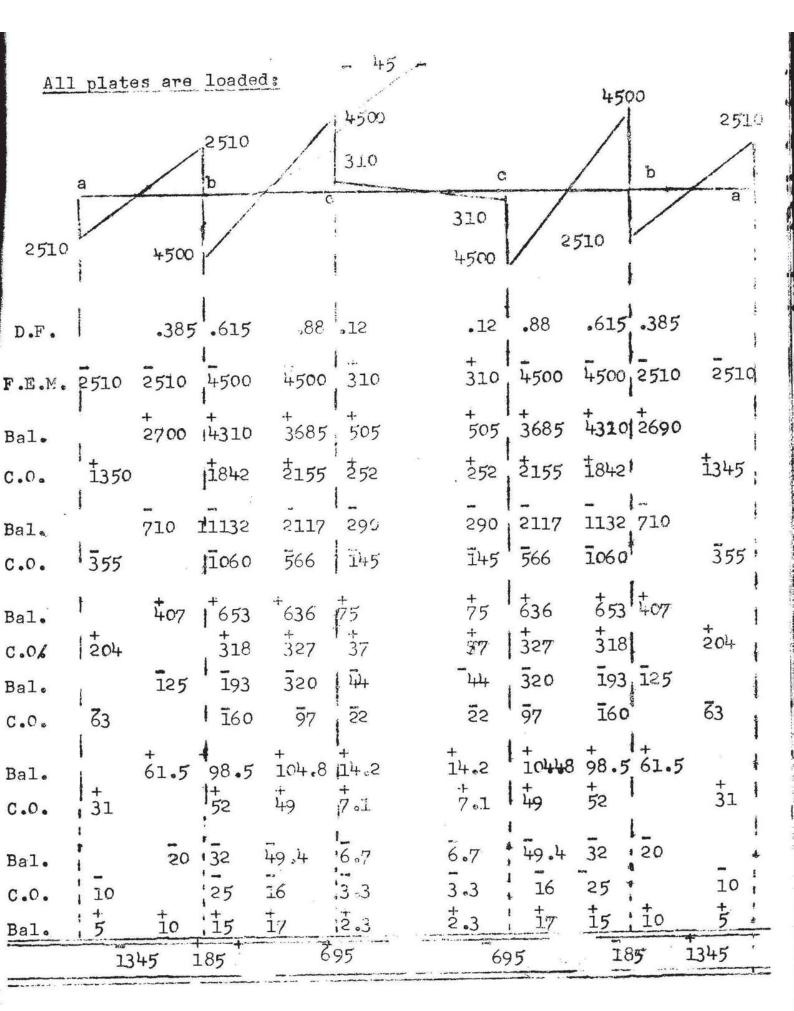
A-B Loaded alone:

		2510			-		-	ã
Ð		b	<u> </u>		c		ъ	- 20 T
	2510							
D.F.	.385	.615	.88	•15	115	.88	.615	•385
E.M.	-2510 <b>-</b> 2510							
Bal.	+970	+1540						
C.O.	+485		+770		Ť			
					1			
Bal.			6 80	-90				
C.O.		-340			-45			
Bal.	+131	+209			+5.4	+39.6		
C.O.		,	+105	+2.7			+19.8	
0.0.	10)		. 107	,			,	ŀ
Bal.	ŧ	-	94.8	-12.9	•		-12.1	<b>-7.</b> 7.
C.O.		-47.4			-6.5	-6.1		-3.9
					- 1			· ·
Bal.	+18.2	+29.2			+1.6	+11.		
C.O.	+9.1	+	+14.6	+.8			+5.5	
Bala			-13.6	-1.8			-3.4	-2.1
	-1950.9 +1	390.8	-10	5	+44	.1	-9.8	+ 3.9

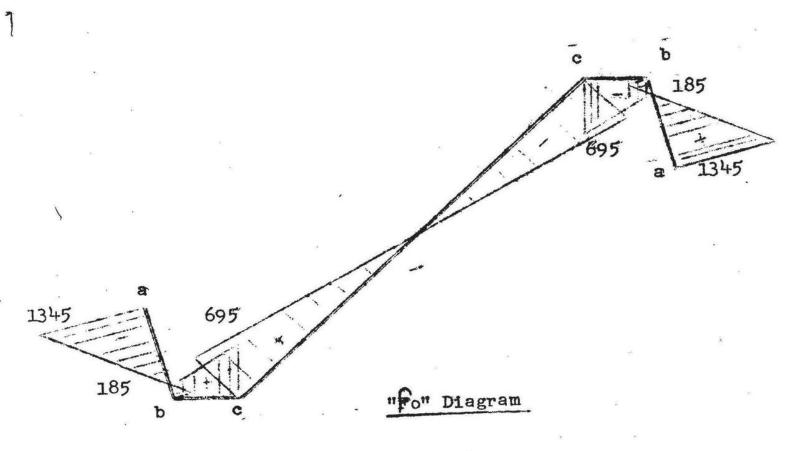
lone:

B-C Los	ded alone:				
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	14500	4500	i		Ŷ
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870	1980	1300			1
al.	760 1220	1215   165	32.5 237.5	1	
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c.o. 1380	608	610   16	83	119	•
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1. 7	15   25	24.5 3.5	2 17	7 : 4	2
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. 1 27	53 84	16.7 2.3	2.3 16.7 1.2 42	84 53	27
). 1.6	3.2,5.2	38 5.2	5.2 38 2.6 2.6	5.2   3.2	1.6
1. 1.	7.2 111.8	4.6 0.6	· 5 1 4.6	+	3.6
1.	1	5.2 .7	.\$ 15.2	- ! -	
29	56	+ 245	245	+ 56	29 I

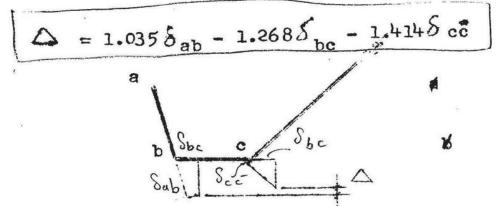


# - Stresses after distribution neglecting The effect of joint displacements

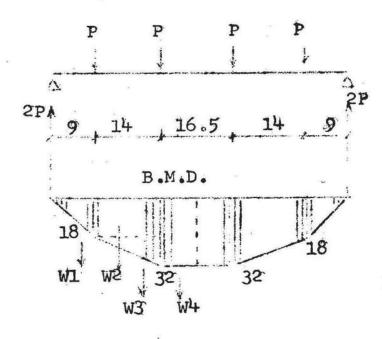


## b . Correction Analysis:

- Geometrical Relation between  $\triangle$  and  $\delta$ :



# Displacement Parallel to plate element & i

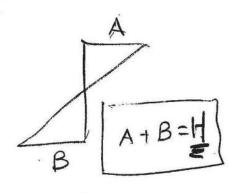


$$W1 = 18 \times 9 = 81$$

$$W^2 = 18x14 = 252$$

$$w3 = \frac{14 \times 14}{2} = 98$$

$$\begin{cases} = 695x31;25-( ) \\ = 21700-(264x-\frac{8.25}{2}+98(8.25+\frac{14}{3}) \end{cases}$$



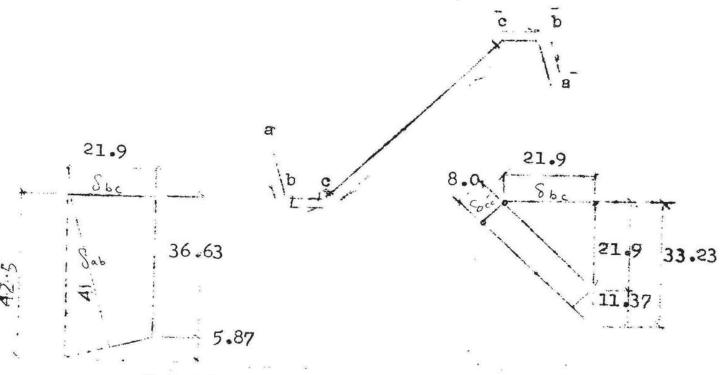
$$H = \underbrace{\frac{1}{M}}_{H} \times \frac{s}{q} = \underbrace{\frac{1}{M}}_{M} q = \underbrace{\frac{1}{35h}}_{q} q$$

$$P = \frac{IH}{32d}$$

$$...S = \frac{13450}{x} = \frac{IH}{x} = \frac{L^2}{4L^2} = \frac{HL^2}{9.3Ed}$$

# Relative displacement " 4 "between edges \$ & C or \$ & C:

$$bc = \frac{510}{9.3x^2.5} = 21.9 - \frac{L^2}{E}$$



Point b

$$\triangle = 3.4 - \frac{L^2}{B}$$

Point c

- Plat	e loads due to tra	113 V 01 D 0	Č		- b
	P .915	.085	Š	P	P
D.F.	+ <sub>M</sub>	0			
bal	•915M	0.085			
inal M	O <sub>g</sub> e8 5M	0.085M			

### - Stiffnesses factors:

$$kcb = \frac{3}{4} \times \frac{1}{2.5} = \frac{3}{10} \qquad kcc = \frac{1}{18775} \times \frac{1}{2} \quad due \text{ to symmetry.}$$

$$\frac{3}{10} + \frac{1}{37.5} = \frac{37.5 \times 3}{37.5 \times 3 + 10} = .915$$

$$M = \frac{3EI}{12} - \Delta$$

$$\tilde{P} = \frac{0.085M}{1}$$

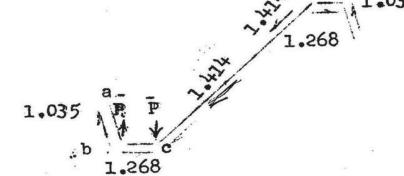
$$10 \ P = \frac{0.085 \ x3x1x.5x.5x.5}{12x2.5x2.5x2.5} = \frac{2 \ kg/cm}{12x2.5x2.5x2.5}$$

If 
$$\emptyset = \frac{P}{12} \times 10^{-6}$$
 kg.cm.

$$0.085x3x1x.5x;5x.5x62-5x62.5 \times 10^{6} = 0672x10^{6} E\Delta$$

$$12 \times 2.5x2.5x2.5 \times 10^{6} = 0672x10^{6} E\Delta$$

$$0.0672x10^{6} E\Delta$$

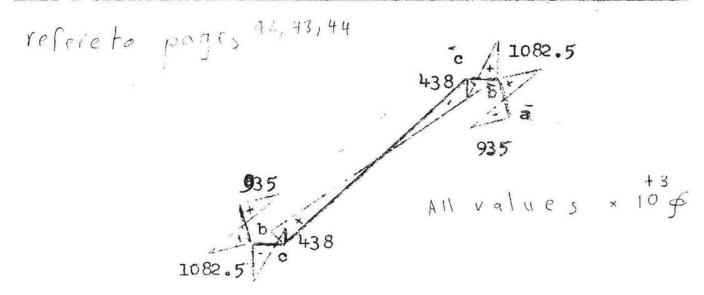


### Bending moments & stresses due to transverse deformations:

Palte	Ld.	B. M	Stresses
AB.	1.035 P	1.035 x10 <sup>6</sup> \$	0.778 x106 \$
BC	1.268 P	1.268 x10 <sup>6</sup> ±	2.435x10 <sup>6</sup> \$
cc	2*8558b	2.828x10 <sup>6</sup> \$	0.097x10 <sup>6</sup> #

Stresses by supermostilions.

plate	a <sub>x1</sub> t <sub>0</sub> 3	b <sub>x1</sub> 03	c <sub>x10</sub> 3	x163	b <sub>xlo</sub> <sup>+3</sup>	$\tilde{a}_{x10}^{\dagger}3$
AB Loaded	6 <b>5</b> 8	430	31.5	13.7	3.05	1.2
BC Loaded	305	610	225	100	55 +	11
CC Loaded	9.1	17.5	76.5	76.5	17.5	9.1
5	935	1082.5	1438	43 <b>8</b>	1082.5	935



-Substituting the values of & in the geometrical reletion

$$\delta ab = (51 - \frac{2017}{4 \pi^2} - \phi \times 10) - \frac{L^2}{E} = (41 - 52 \times 10^3 \phi) - \frac{L^2}{E}$$

$$\delta bc = (21.9 + \frac{1520}{2.5 \pi^2}) - \frac{L^2}{E} = (21.9 + 61.5 \phi \times 10^3) - \frac{L^2}{E}$$

$$\delta cc = (8 + \frac{2 \times 438}{18.75 \pi^2}) - \frac{L^2}{E} = (8 + 4.73 \phi \times 10^3) - \frac{L^2}{E}$$

but we have:

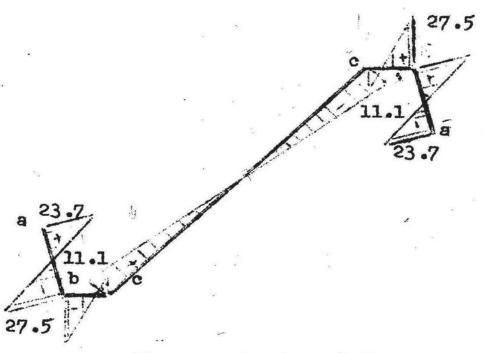
$$\Delta = (\delta_{ab}/\cos 15 - \delta_{bc} + 15) - (\delta_{bc} + \sqrt{2} \delta_{cc})$$

$$1e \Delta = 1.035 \delta_{ab} - 1.2688 \delta_{bc} - 1.414 \delta_{cc}$$

$$= 3.6 \frac{L^2}{E} - 138.0 \times 10^3 f_{-E}^{-L^2}$$

$$= 3.6 \frac{L^2}{E} - 36.3 \Delta \qquad \Delta = 0.0967 \frac{L^2}{E}$$

$$\therefore \phi = 25.4 \times 10^6$$

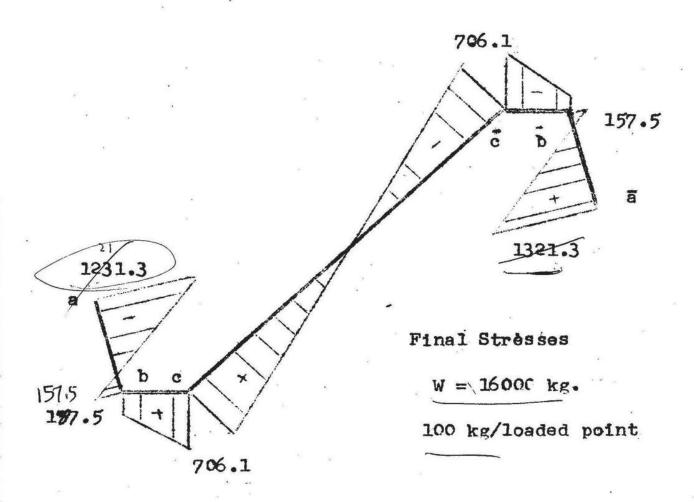


Stresses due to relative displacement.

#### c- Final Results

#### - Longitudinal Stresses.

Stresses due to transverse defromations are added algebraically to the corresponding values obtained in the elementary analysis to get the final stresses.

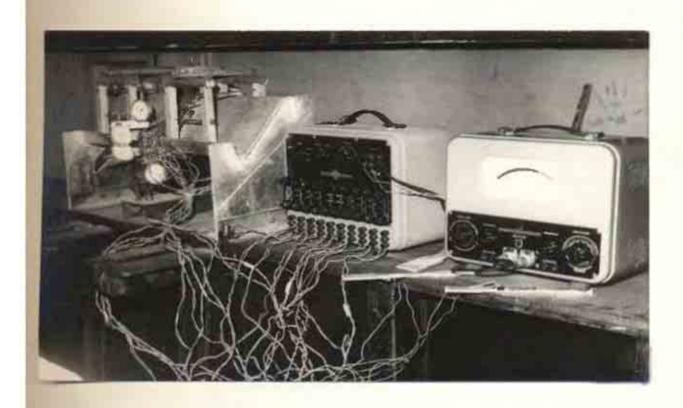


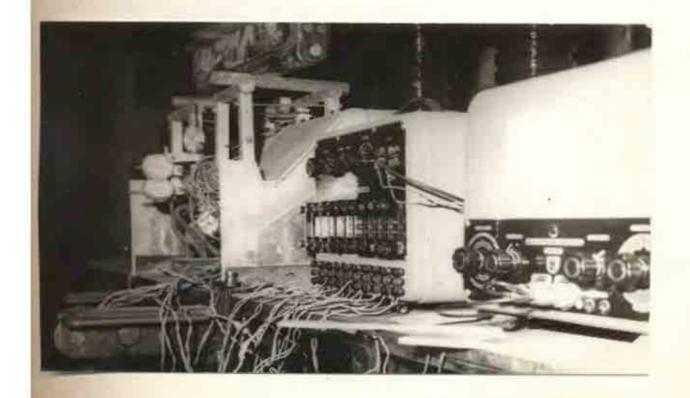
100 × 4 = 8 kg -Transferse Stresses  $M = \frac{0.085 \times 3 \times .5 \times .5 \times .5}{12 \times 2.5 \times 2.5} \times \frac{62.5 \times 62.5}{1} \times \frac{50.0967}{100.0967} \times \frac{62.5 \times 62.5}{100.0967} \times \frac{62.5$ M diagram 6M 6x 0.16 = 3.84 kg/cm<sup>2</sup> 17 = 1 N Diagram Displacements = 0.232 = 0.225 cm  $\delta_{\text{bc}} = \frac{(695-185)}{9.3 \times 2.5} \quad \frac{L^2}{E} \quad \frac{(275+11.1) \times L^2}{2.5 \times \pi^2} \quad E$ = 0.1250 + 0.009 =-134 cm  $\delta_{cc} = \frac{695 \text{ x}^2}{9.3 \text{ x} 18.75} \qquad \frac{L_2}{E} + \frac{11.1 \text{ x}^2}{18.75 \text{ m}^2} \times \frac{L^2}{E}$ = 0.04580 +0.00068 = .04648 cm Abv = Sab/eos15 - Sbc tan15 = 0.2350 - 0.0347 0.2003 cm Dev = Sbe tan46 + Sec/cos 450

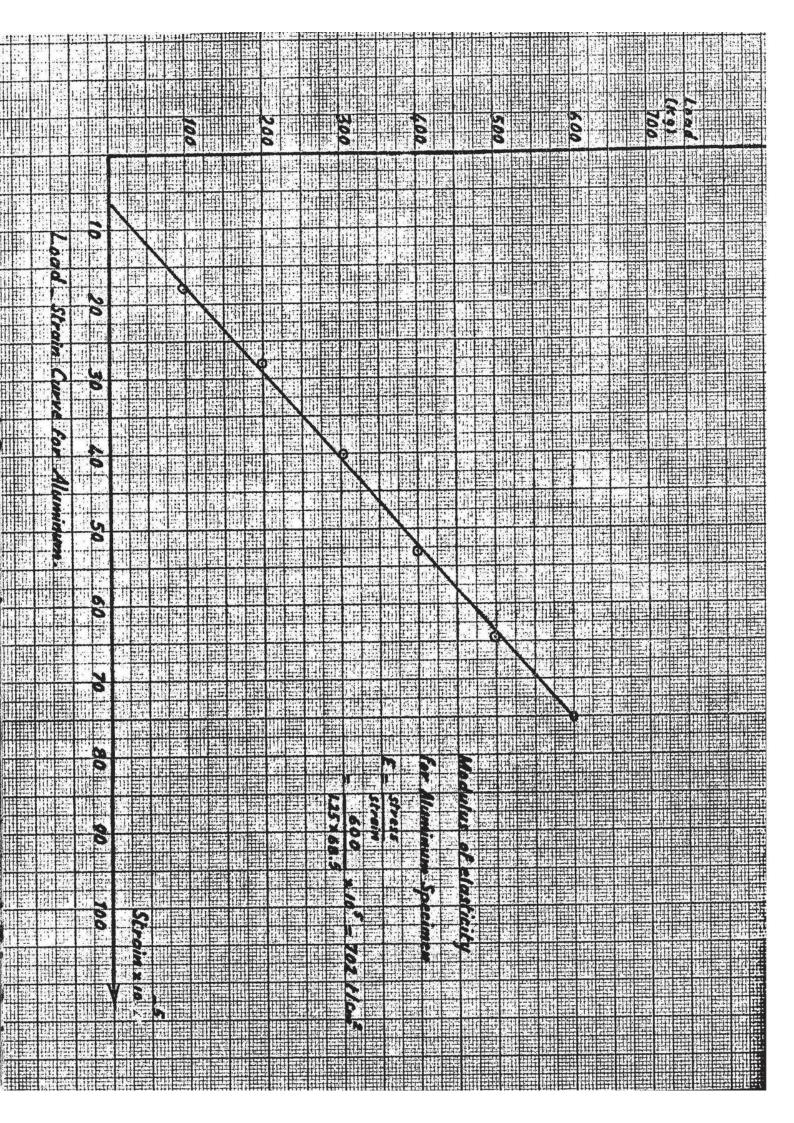
= 0.1996 CM

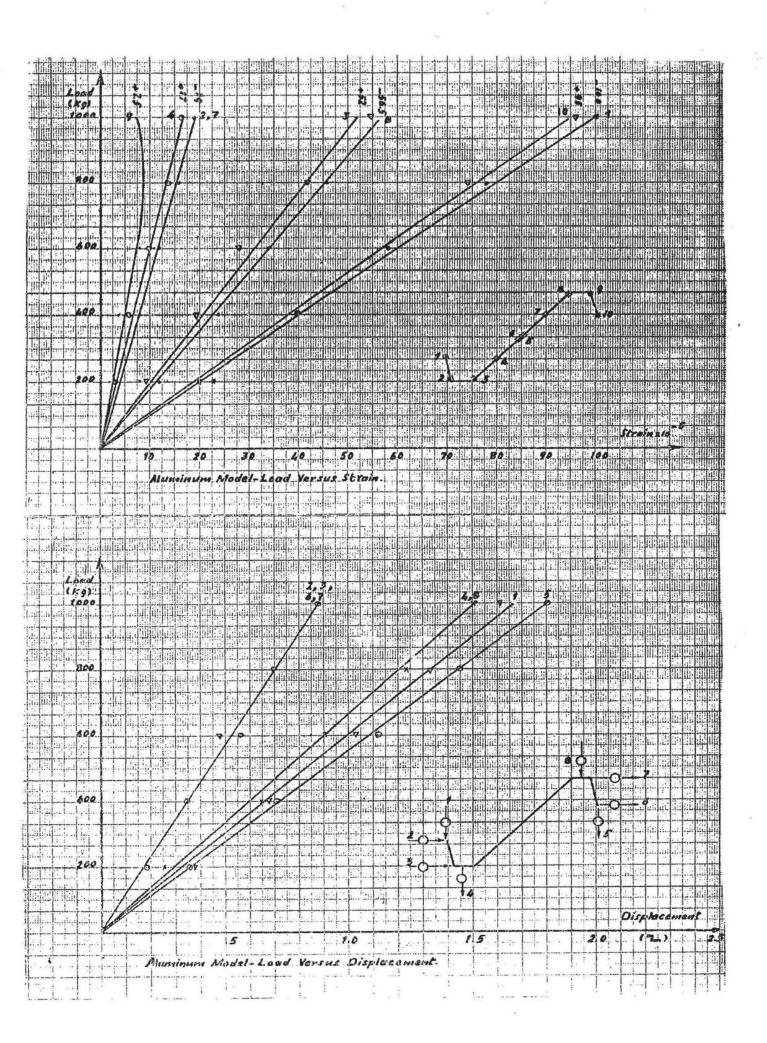
= 0.1340 + 0.0656

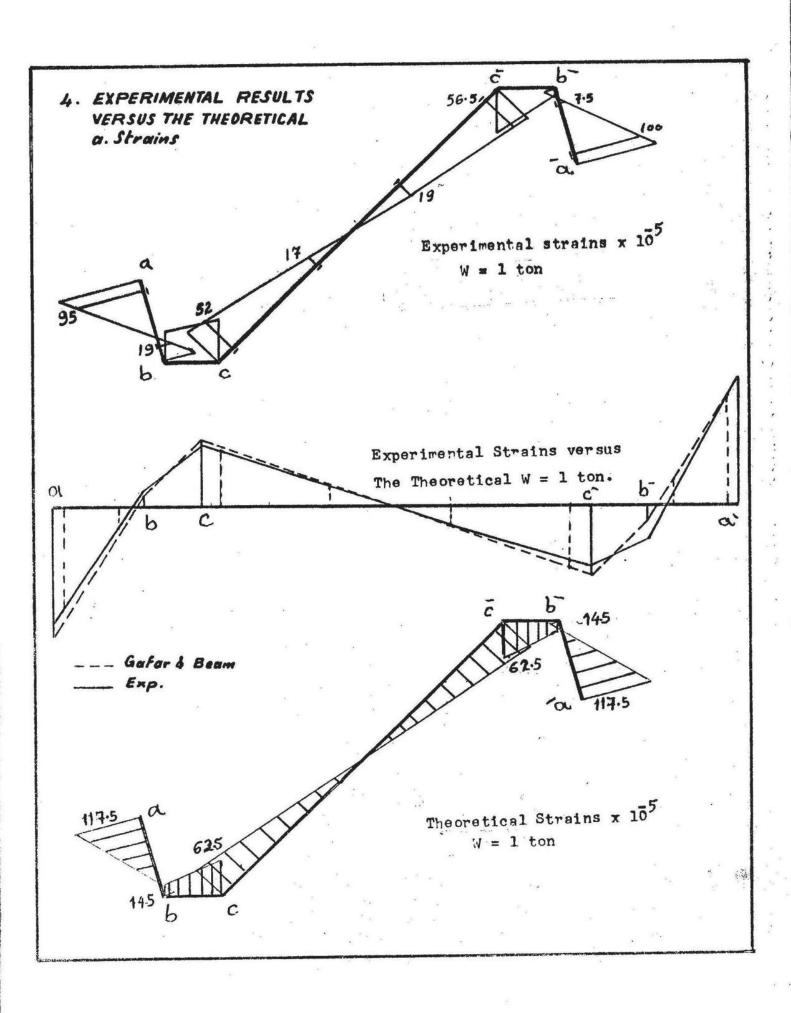
### 3 - EXPERIMENTAL RESULTS

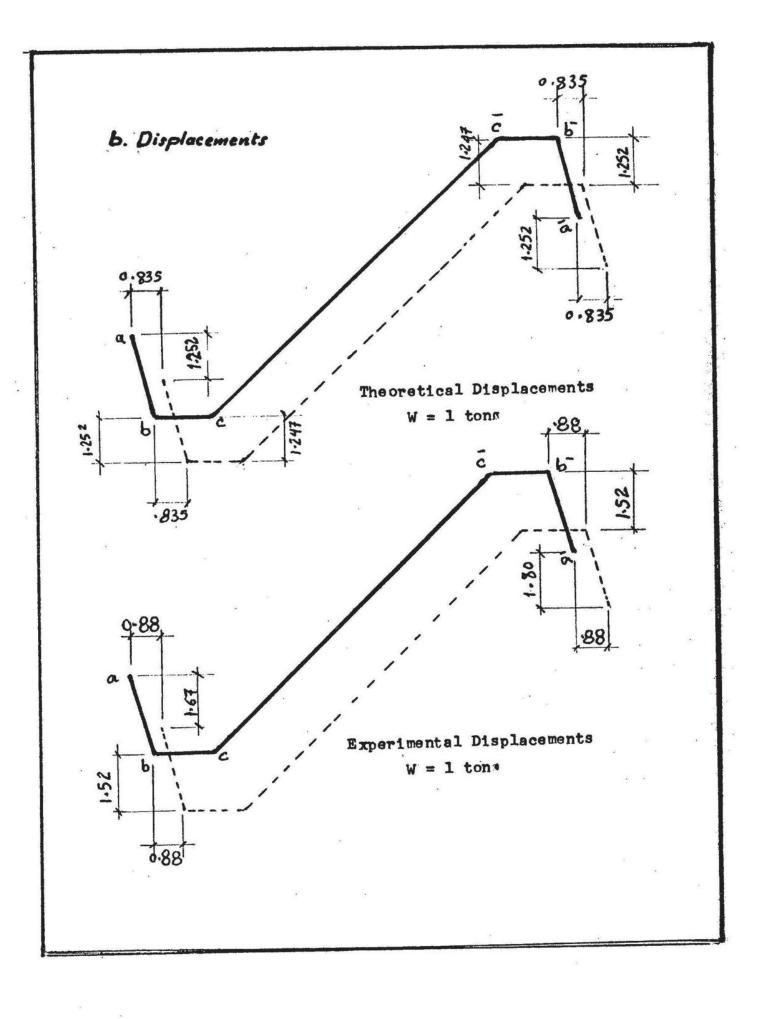












## CHAPTER

III

THE REINFORCED CONCRETE MODEL

### 1- DESCRIPTION OF THE MODEL

The mix was made according to the following ratio per cubic meter of finished concrete:

Cement Sand Gravel W/C 400 kg 580 kg 1160 kg 0.5

The course agregate has a maximum nominal size of 6.0mm and the fine aggregate is the Giza aggregate. Also the coment used was of the rapid hardening portland cement of Helwan. The curing was made by covering the roof with a layer of sand witted twice a day for the first two weeks and then left in the room temperature till date of test which was after four weeks. The properties of the hardened concrete and the steel used are shown in tables (3-1) & (3-2).

Fig.(3-7) Shows the leading system where the leads from the jack are transmitted to a sixteen leaded points

through a system of group of simple beams similar to that of the aluminum model. Also the position of the longitudinal and transverse stain gages as well as the dial gages are shown in Fig.(3-8) and Fig.(3-9).

The model is supported at two end diaphragms having a thicknesses of 6.0 cm and of concrete dimensions and details of reinforcement as shown in Fig.(3-6).

This model was subjected to three cases of loading namely:

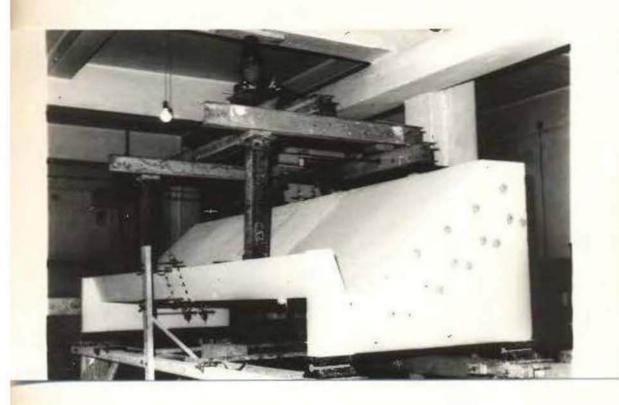
- a Symmotrical loading .
- b- Unsymmetrical leading where leads were applied at the upper joints.
- c Unsymmetrical leading where leads were applied at the lower joints .

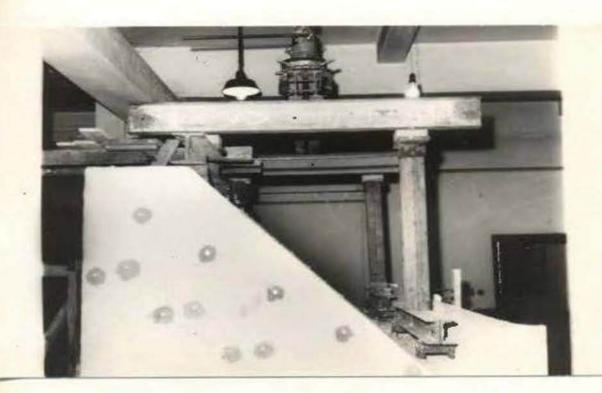
Table (2-1): Properties of Hardened Concrete

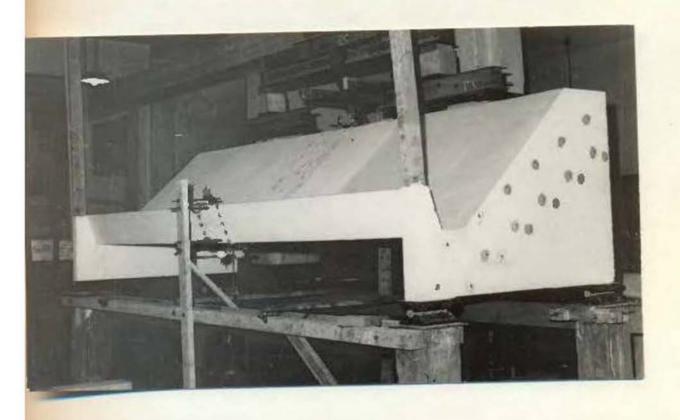
	St	Strength Ke		E _ For plain
Piece No	Cube	Prism	Cylinder	Concrete Prism
1	31S	280	240	
2	300	280	5,4,4	PASSE OF SERVICE
3	320	288	252	
Average	311	283	245	300 x 10 <sup>3</sup> kg/cm <sup>2</sup>

Tagle (3-2) Properties of the Steel Barsused

Properties	ø 2mm•	ø 6mm.	ø 13 mm.
Area of C.S cm <sup>2</sup>	0.0314	0.279	1.33
Yield load kg.	83	7≸0	3750
Ult. lead kg.	109	1050	4900
Yield stress kg/cm	2640	2690	2810
Olt. stross kg/cm <sup>2</sup>		3760	3670

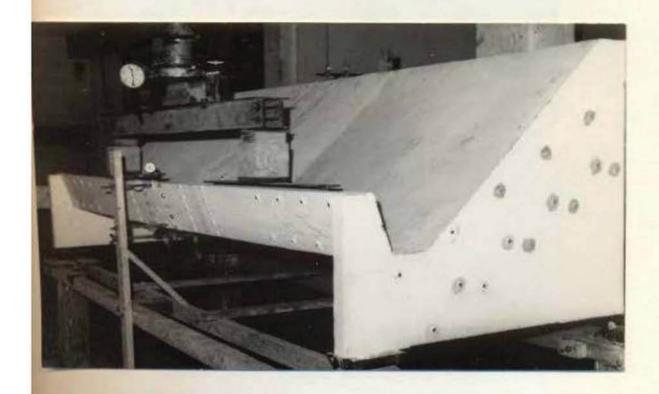


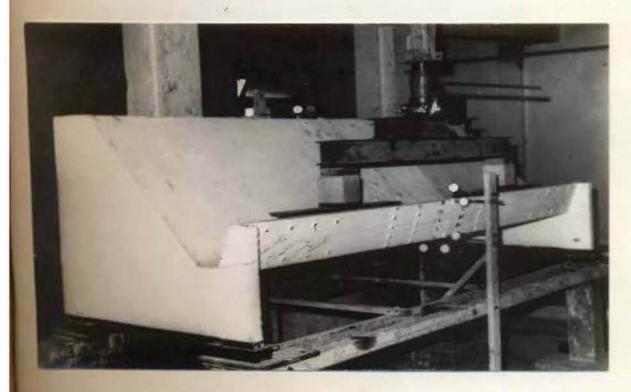


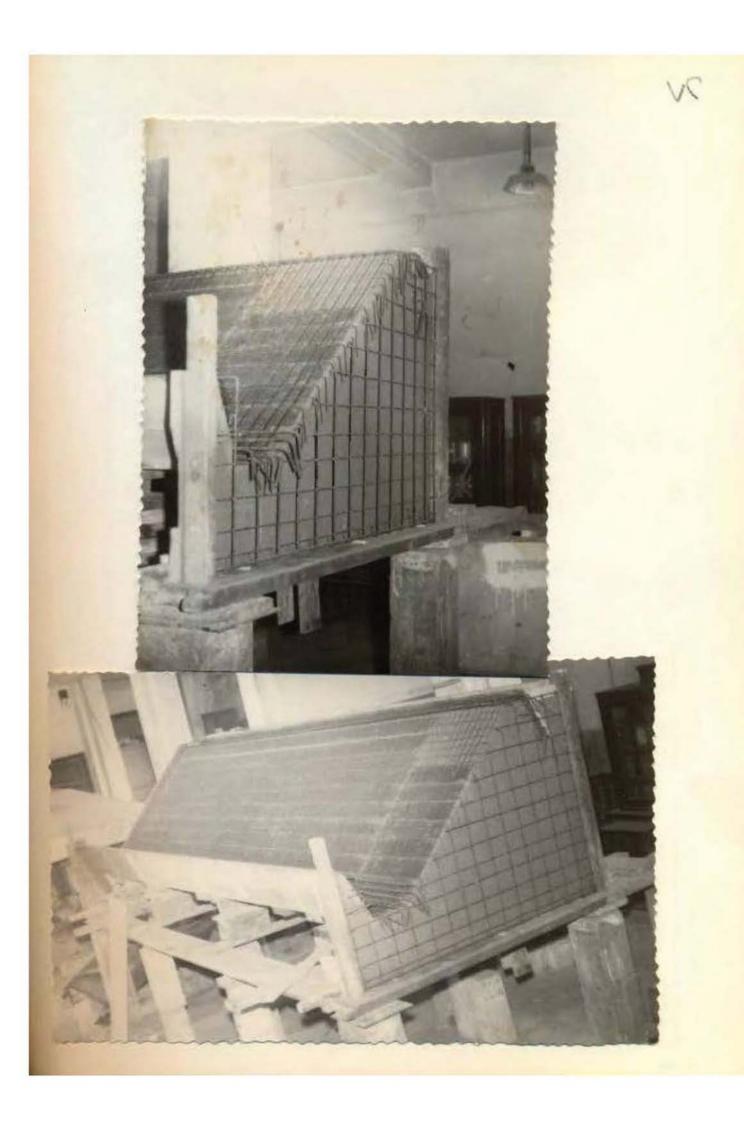


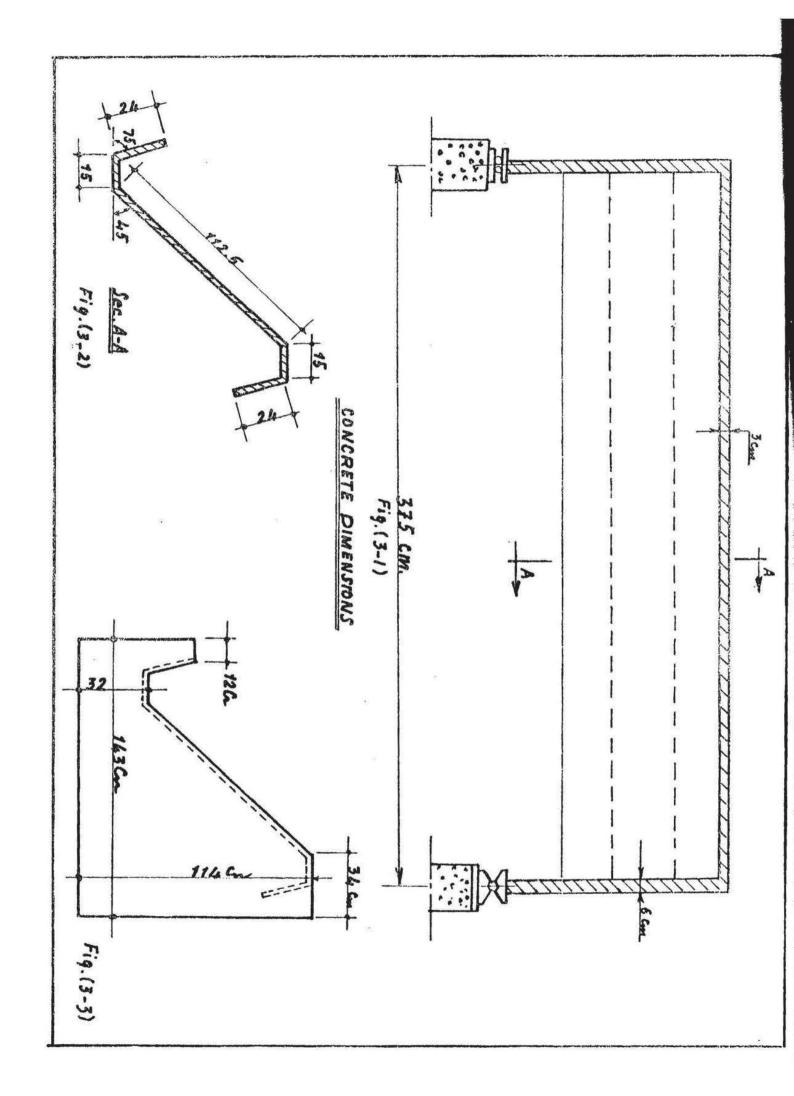


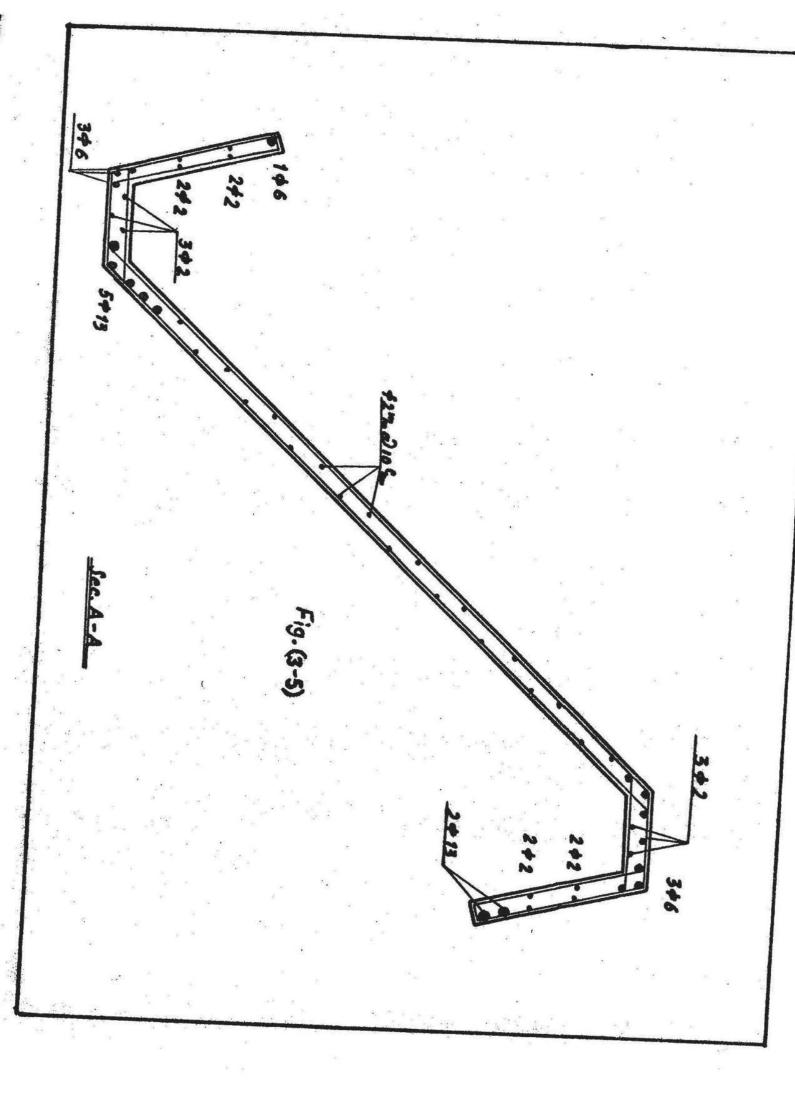


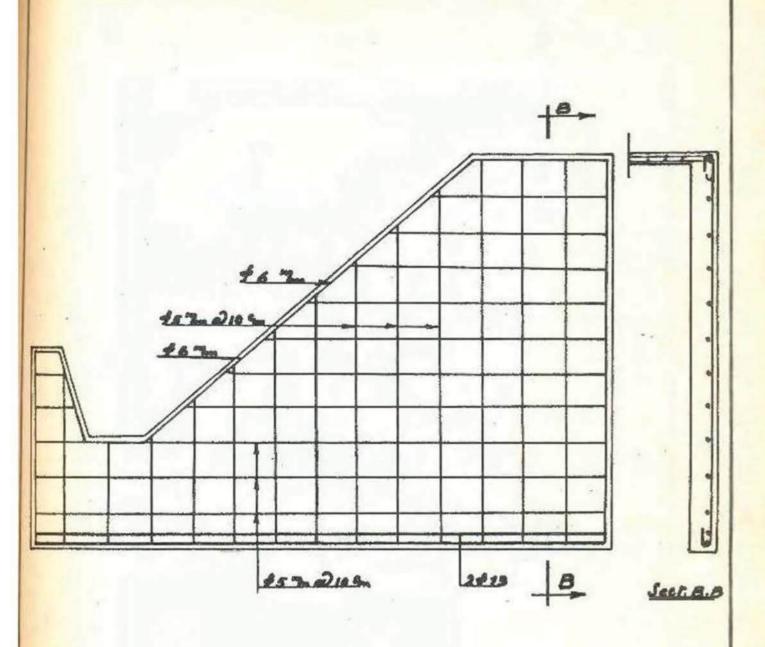








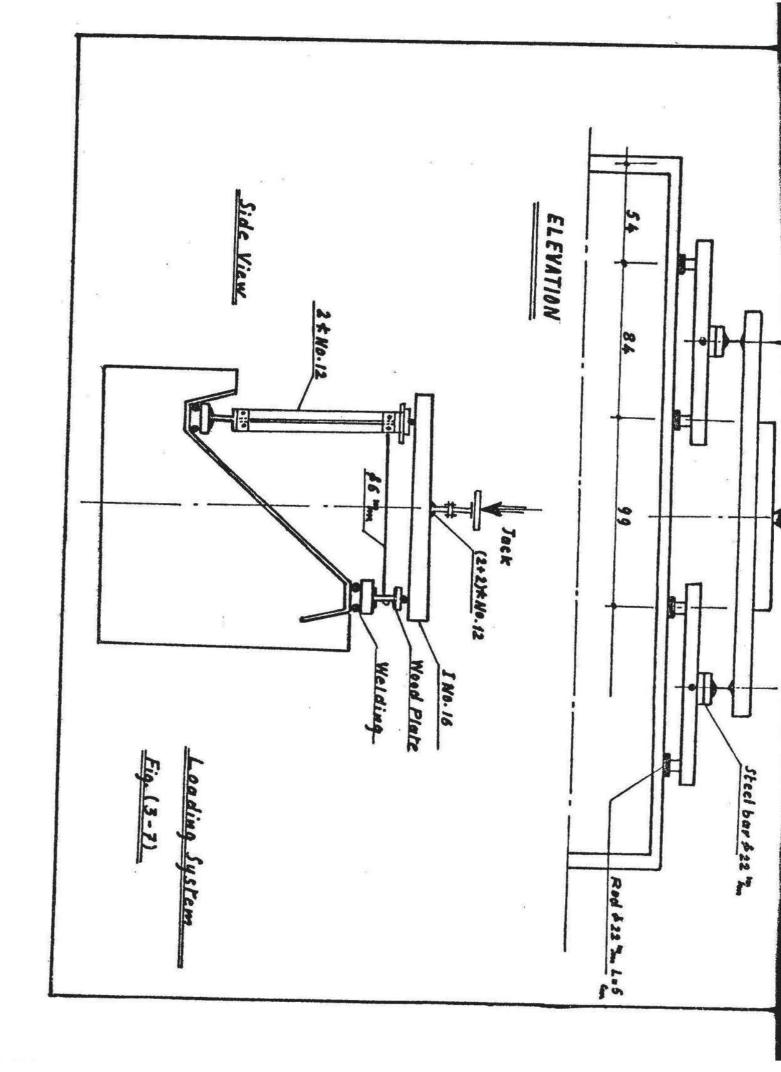


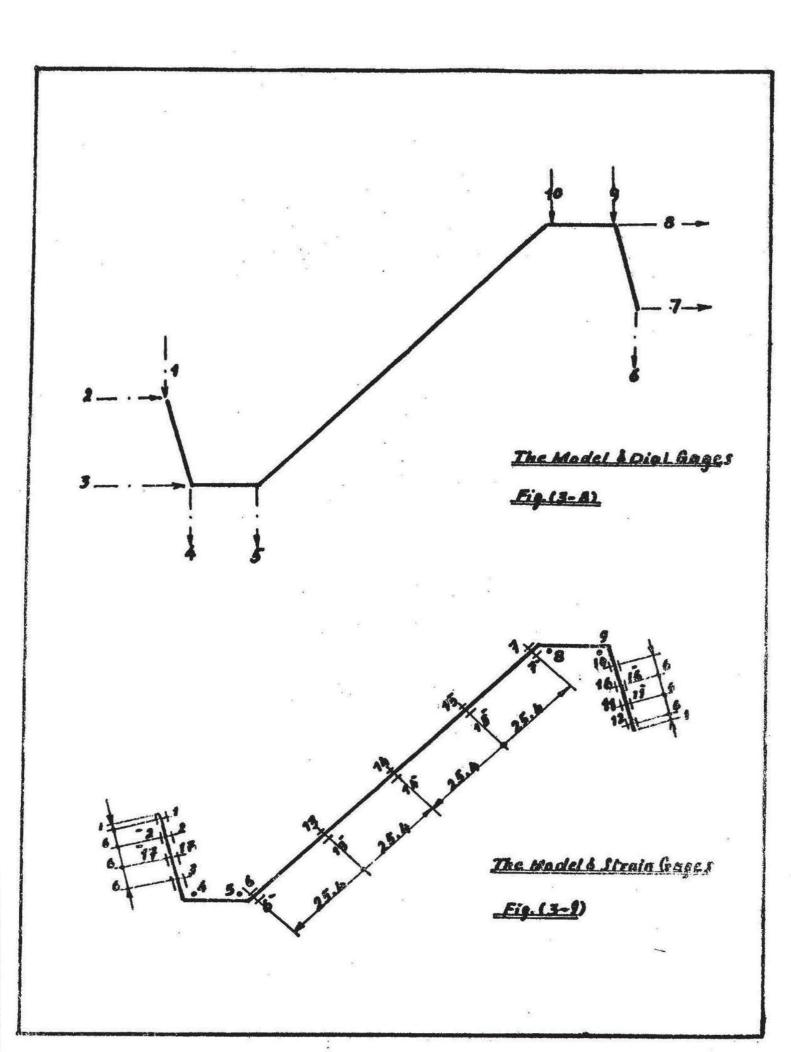


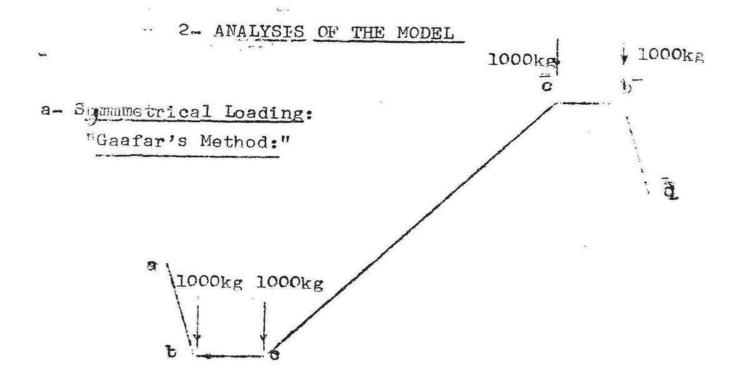
Sectional elevation
Sec. 1.10

DETAILS OF THE END DIAPHRASM

Fig. (3-6)



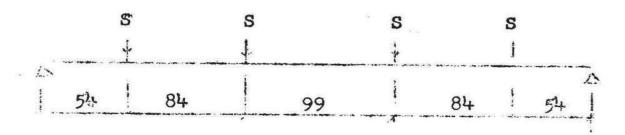




### - Proparties of the Different Plates

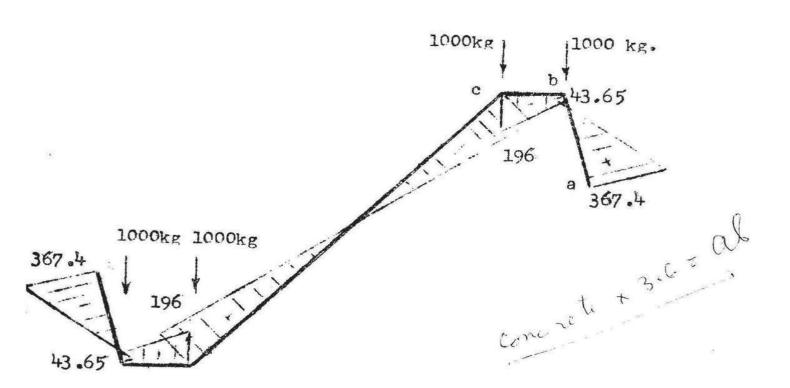
$$L = 375 \text{ cm}$$
.  $t = 3 \text{ cm}$ .

Piate	Dim. cm	Area cm2	$Z = \frac{\mathbf{t} \cdot \mathbf{t}^2}{6} - \mathbf{cm}^3$
AB, AB	24 x 3	72	288
BC,BC	15 x 3	45	112.5
cc 112.5x3		337.5	6328



#### - Stresses

As the model dimensions as well as its loading is similar to the Aluminum model; the stresses can be concluded directly using the previous calculations.



Final Stresses

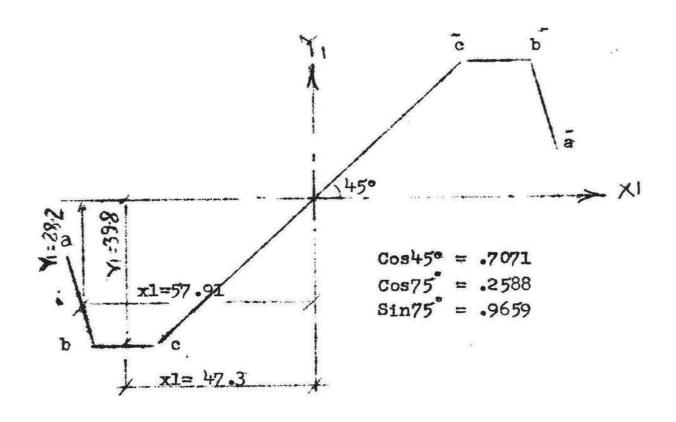
W = 16.0 tons 1000 Kg/loaded point

- Displacements 
$$= \frac{74}{2} = \frac{375}{300} = \frac{375}{300} = \frac{375}{300} = \frac{375}{300} = \frac{375}{300} = \frac{375}{300} = \frac{2}{375} = \frac{2}{37$$

0-2525

Beam Analysis t = 3 cm

							·
M	BÄ	8	8.	вс	AB	Pl.	
	2) <u>i</u> .	15	112.5	15	24	cm.	- other son sufficient, area area for
	6.22	15	79.6	15	6.22	LX.	
	23.2	0	79.6	0	23.2	( v	O CIL
	57.91 28.2	ં.	0	47.3	57.91	×	
	28,2	39.8	0	39.8	28.2	LA	- outrement of
	72	145	337.5	5	72	A	
184180	3240	0	337.5 177700	0	3240	I <sub>x=-12-</sub>	
179850	232	842	177700	842	232	$I_{y} = \frac{Ax^{2}}{12} I_{xy} = \frac{AXY}{12}$	
175968	866	0	1777700	0	866	1 xy =	
683280	240980	190660 71300	0	100660 71300	240980	AX <sub>1</sub> 2	
300 ES	57200	71300	O	71300	57200	AY1-2	
00tytot	1177500	847:00	0	84700	117500	AX <sub>1</sub> Y <sub>1</sub>	



Iy = 179850 + 683280

I<sub>xy</sub> = 175968 + 404400

= 441180 cm4

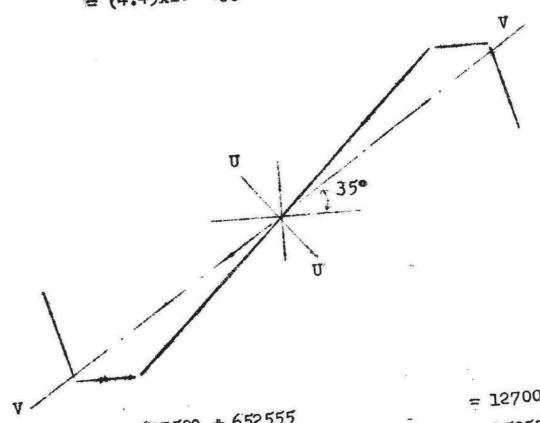
= 863130 cm4

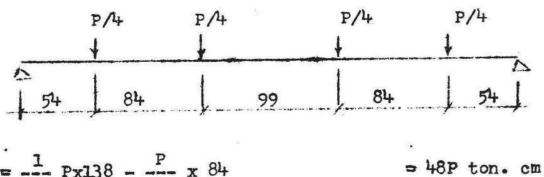
= 580368 cm4

$$tan 20 = \frac{-2Ixy}{Ix - Iy} = \frac{-2x580368}{-421950} = 2.7507$$

$$\frac{Iy - Iy}{2} = 210970$$

$$R = ((\sqrt{\frac{1}{2}} - \frac{1}{2})^{2} + 1xy) = (4.45x10^{10} + 33.68x10^{10})^{1/2} = 6.175x10$$





$$M = \frac{1}{2} Px138 - \frac{P}{4} x 84$$

$$M_u = M \sin 35^{\circ} = 48P \times .574$$

$$M_{\pi} = M \cos 35^{\circ} = 48P \times .819$$

$$S = \frac{M_u}{I_v} \cdot u + \frac{M_v}{I_u} \cdot v$$

Point	oint XI YI v=		v= a1x1+b1y1+c1 a12+b12	$u = \frac{a_1x_1 + b_1y_1 + c_1}{a_1^2 + b_1^2}$
Ã	61.02	16.6	21.75	59.40
В	54.8	39.8	0.99	673
ć	39.8	39.8	9•73	55.4

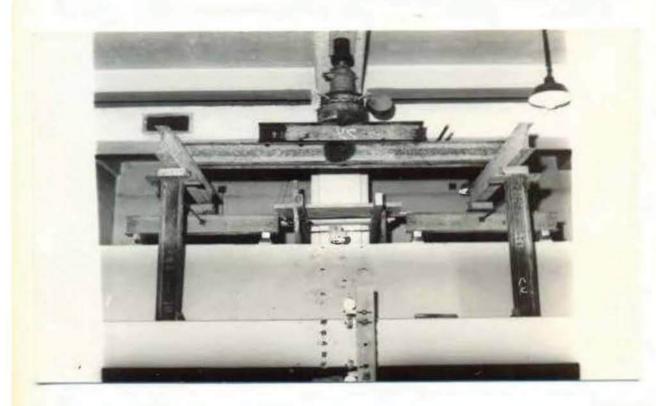
Fer P' 2 16 tens

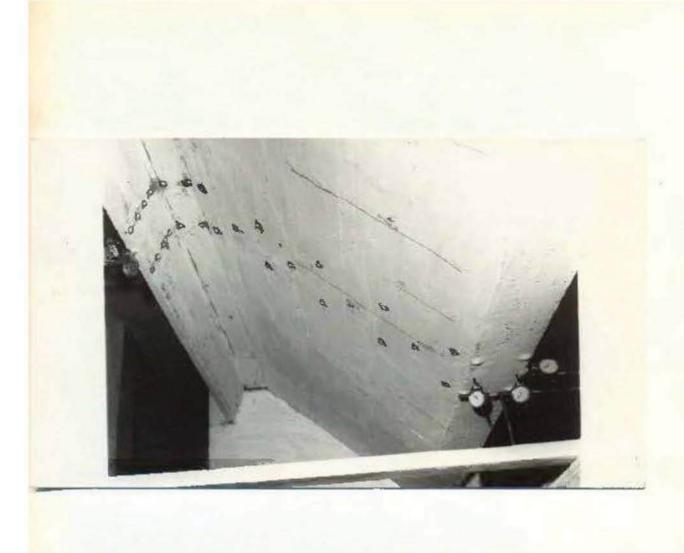
$$\sigma_{B} = \frac{39440}{35055} \times 0.994 \frac{27330}{1270055} \times 67.3 = 1.12 + 1.45 = +2.57P$$

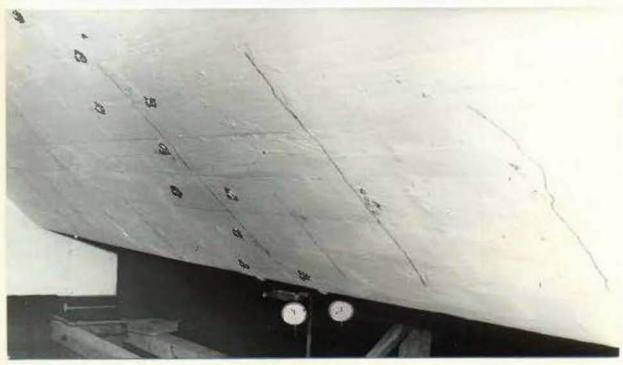
The same as that concluded using Guafar's Method

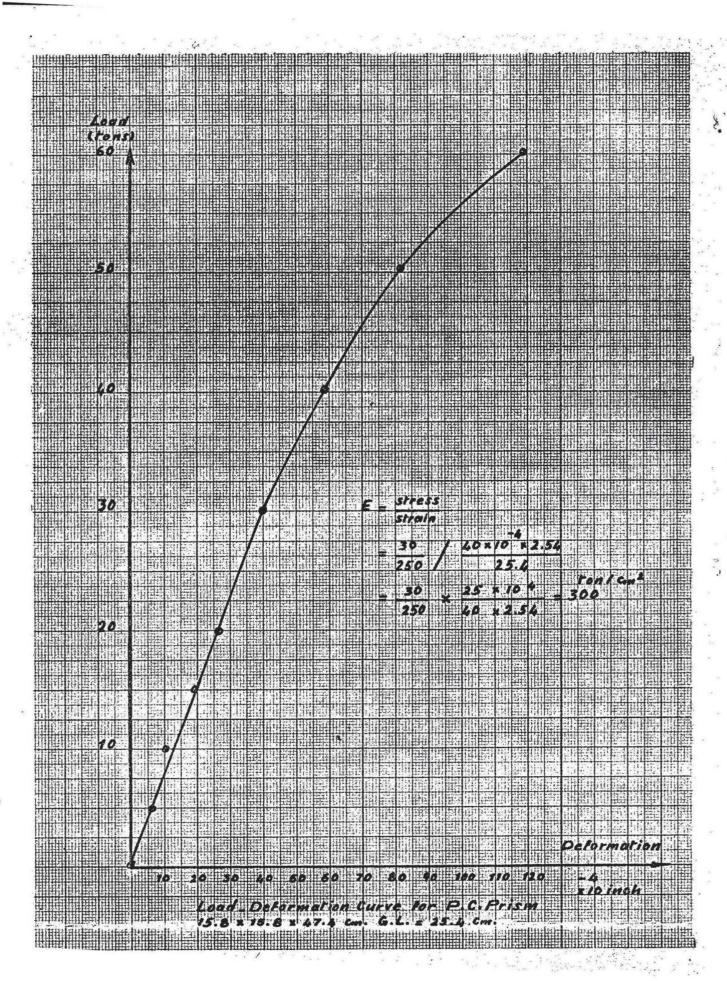
## - Experimental Results

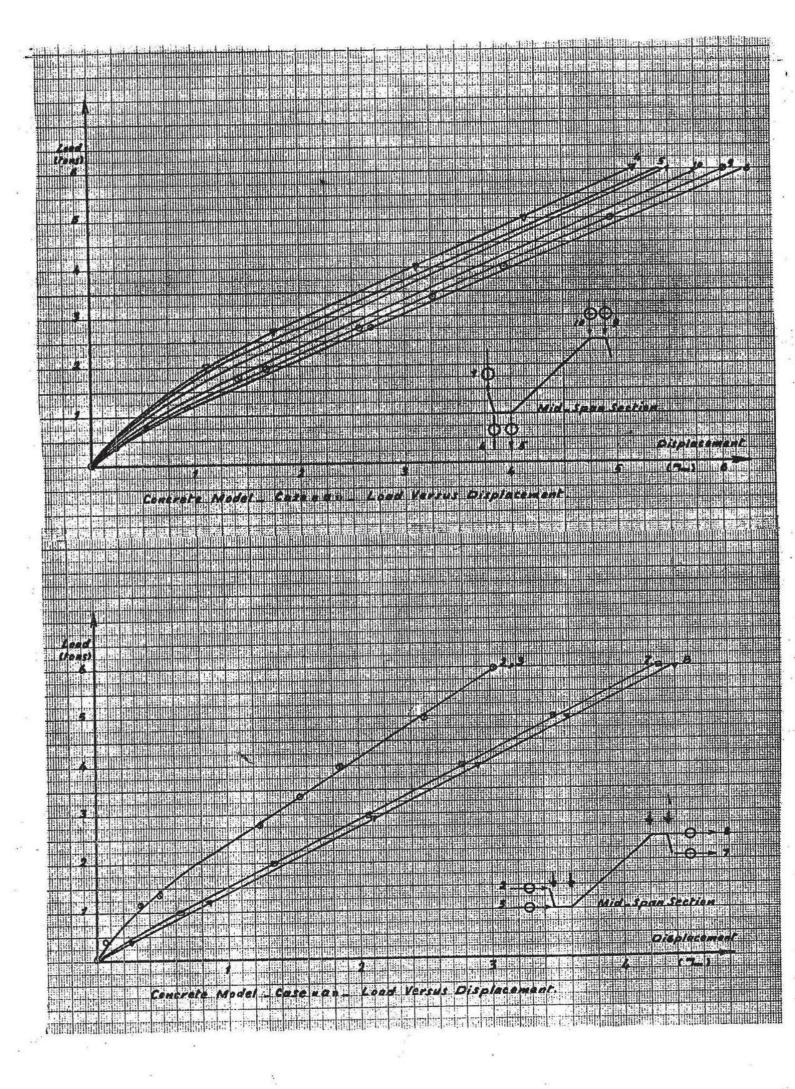












40

ad Varsus

50 00.0

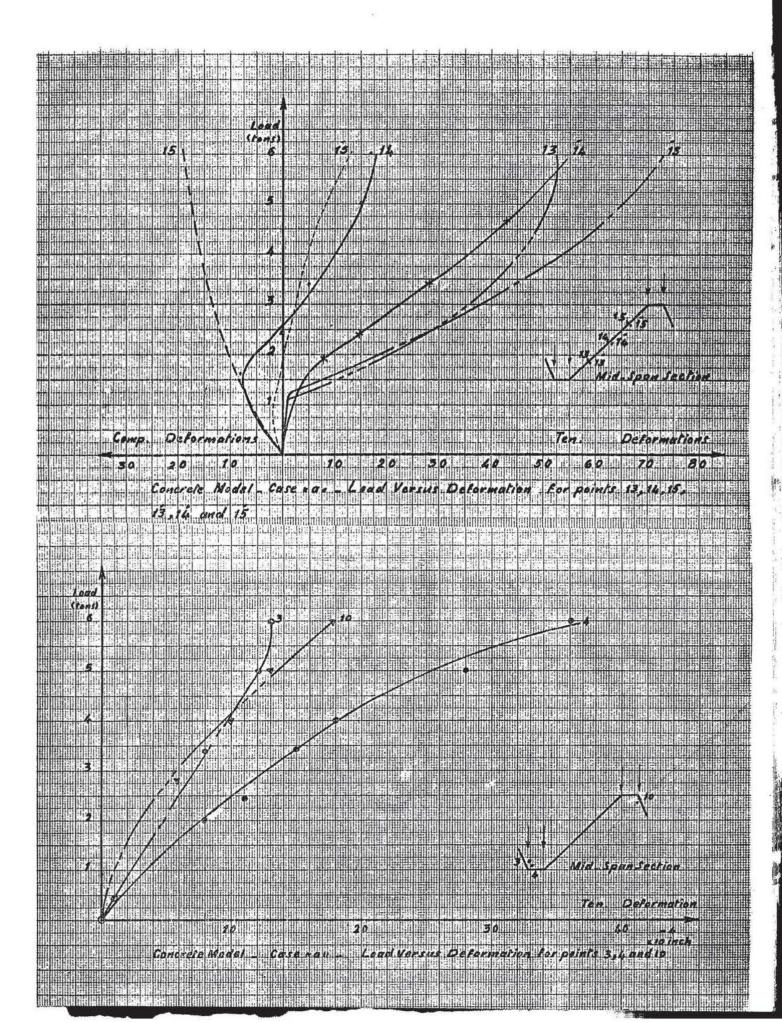
60

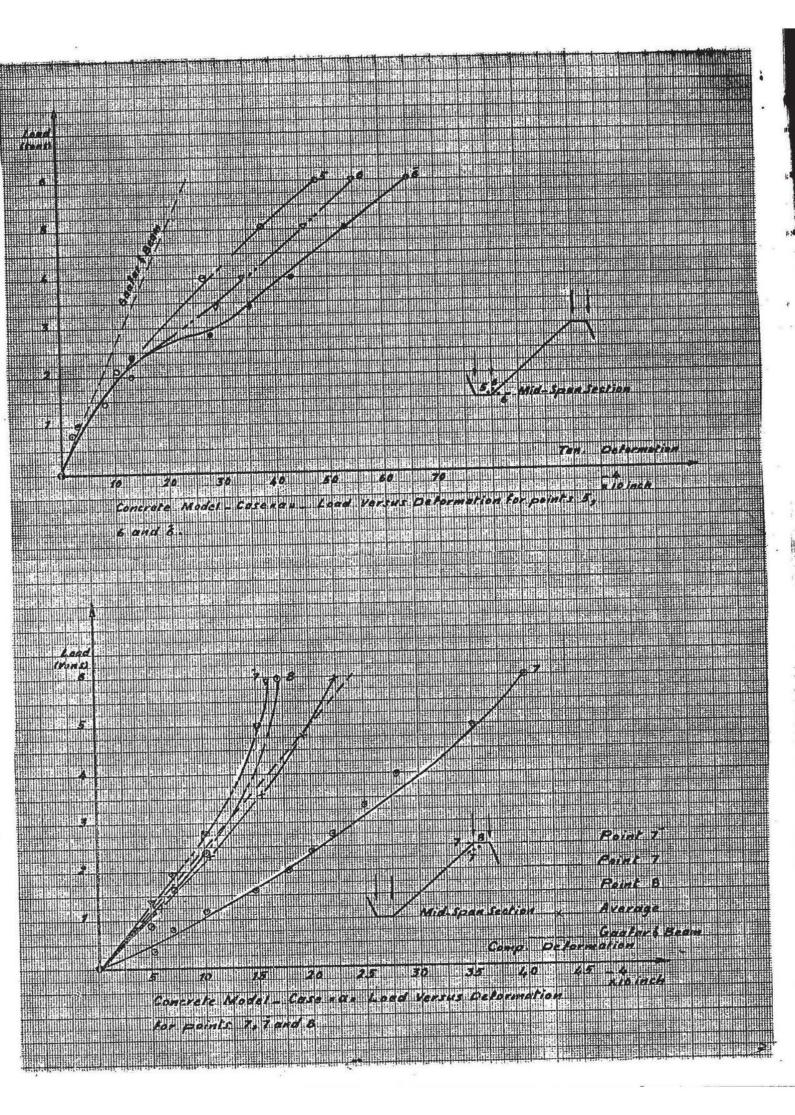
2 00

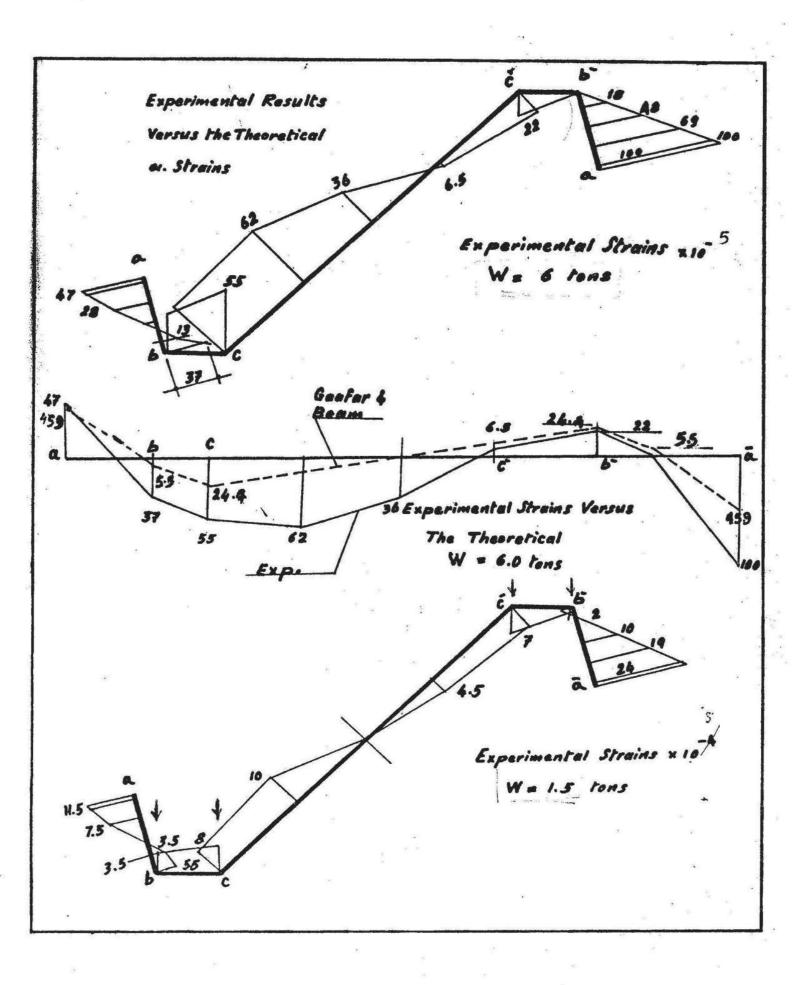
nts

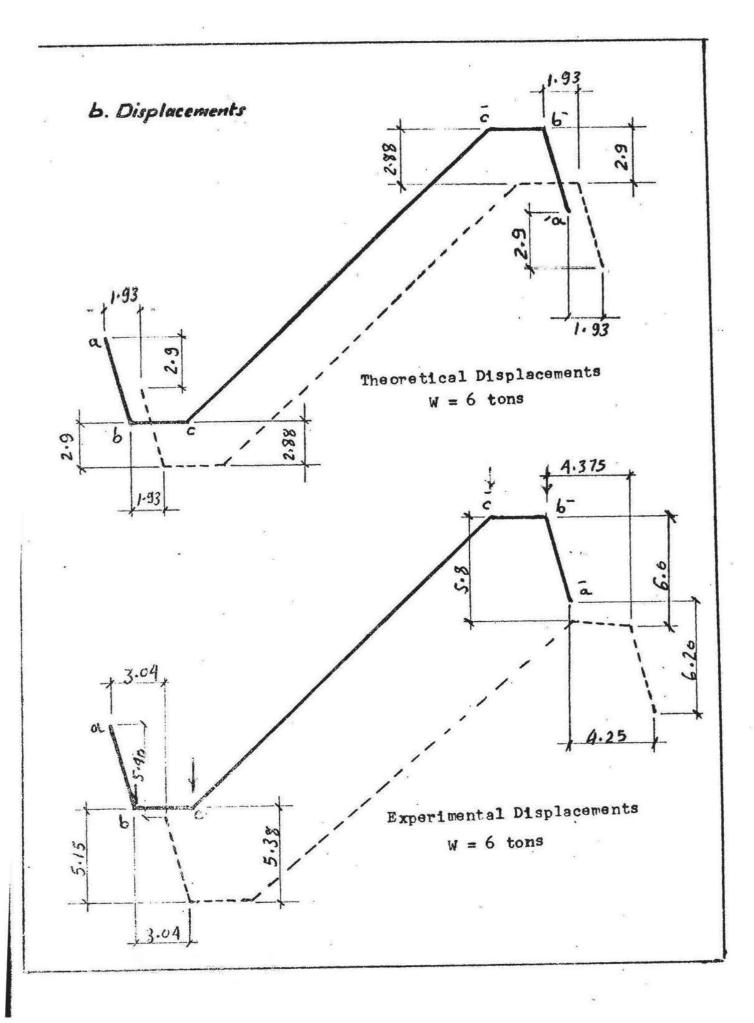
Defe

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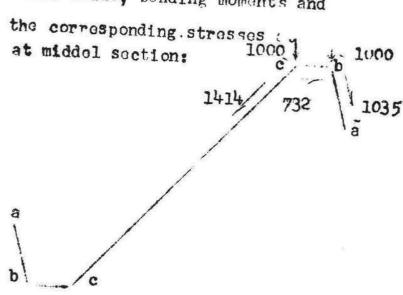


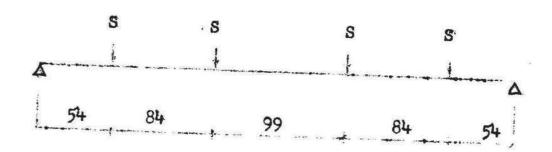


## b- Unsymmetrical Loading:

Loads on the upper edges (Gaafar's Method)

- Plate loads, bending moments and



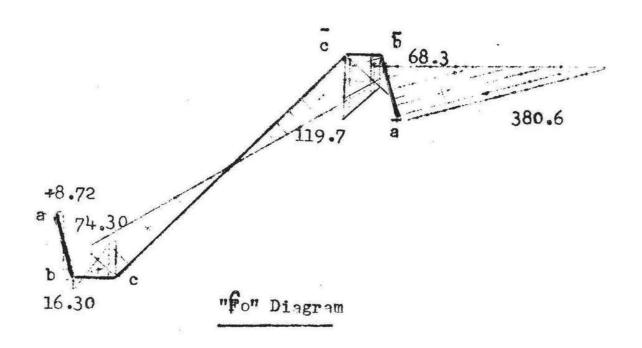


 $B.M = 2S \times 138 - S \times 84$  = 1928

lato	loads	B.M = 1925	Z	0 = M
cc	1414	271488	6328	42.9
СВ	732	140544	112.5	1250
BA 1035		198720	288	694

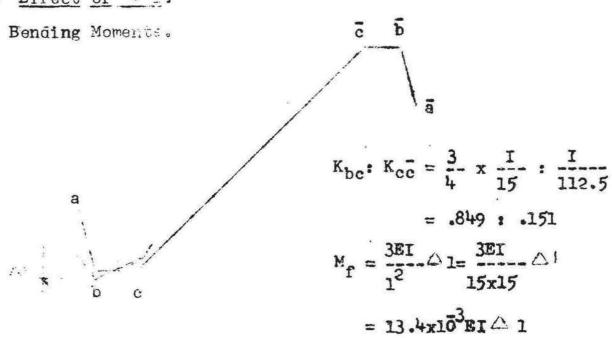
Strosses after distributions neglecting the effect of joints displacement.

plate	а	ъ	c	c	ъ	a
AB loaded	1.09	5.3	35 °5	29.3	387	540.9
BC loaded	+ 5.8	11.4	+52.5	115	+311	157.7
cc loaded	7 4.0	7.7	+34+	-34	+7.7	+4.0
Σ	8.72	16.3	74.3	119.7	68.3	+ 380.6



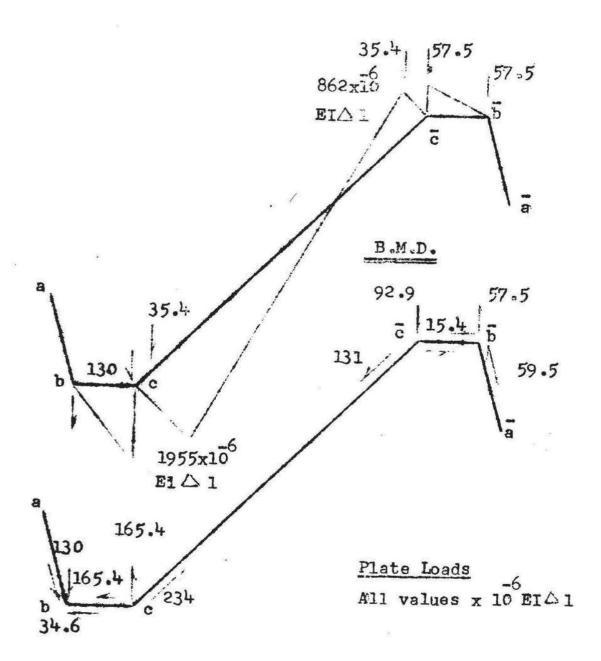
#### Joint Displacements:

- Effect of \(\triangle \).



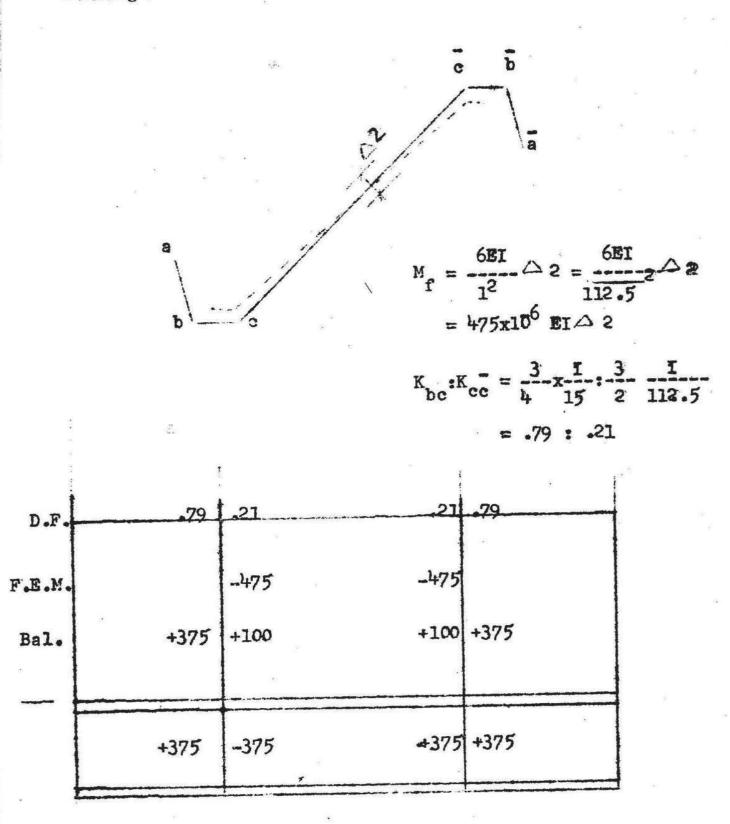
D.F	.849	.151		849
E.M.	·-13 .4	0	0	o .
Bal.	+11.38	+2.02		
c.o.	ı		+1.01	
Bal.			153	-857
C.O.	1	-0.077		
Bal.	+.065	+.012		
c.o.			+.006	
Bal.			001	005
	-1,955	+1.955	+.862	.862

## Plates Loads due to 1

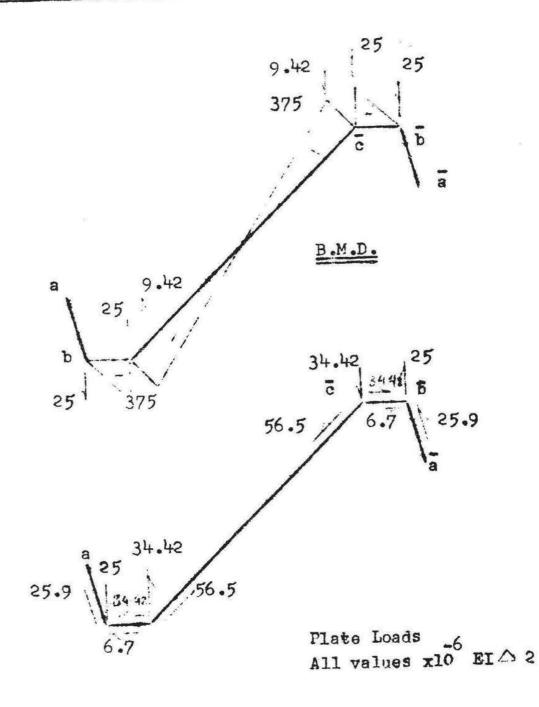


## Effect of △ 2

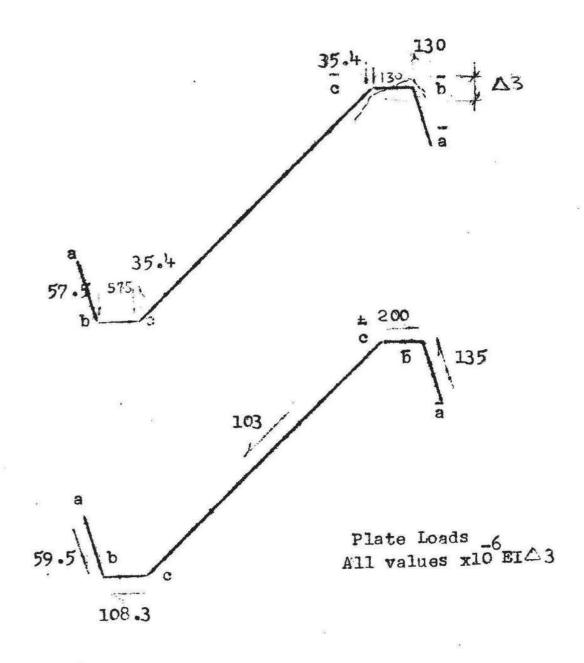
#### Bending Moments



## Plate Loads due to 2

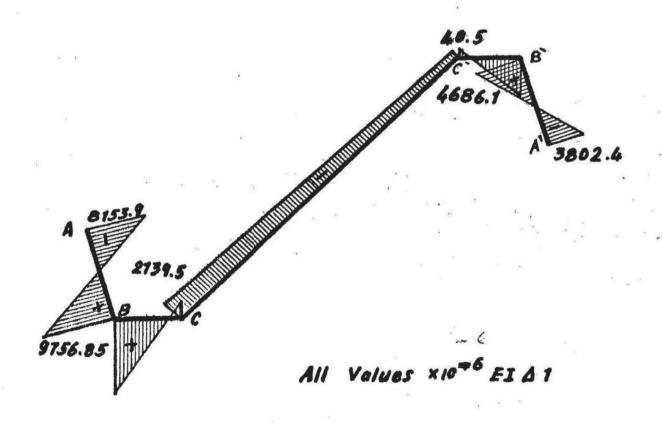


## plate Loads due to∆3



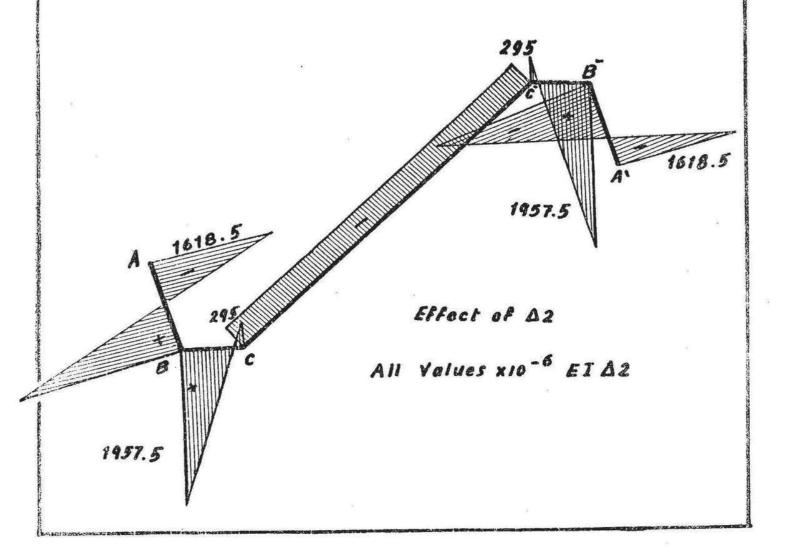
Stresse	s due	to	Δ1:
~~~~~	- 446		

PL.	LD_6	BM AC	0 -6	a	410	C	C	Bio	a"
AB	135	1.900	6600	5100	3650	265	116	25.7	10.2
ВС	200.	2.800	24900	3100	6200	2300	1020	227	116
CČ	103	1.450	228	21.4	41.2	180	180	41.2	21.4
ĊB	1083	1.520	13500	63 *	123	555	1240	3370	1700
BÀ	59.5	. 835	2900	4.5	11.35	50.5	116.5	1610	2250
٤				8153.9	A			A	



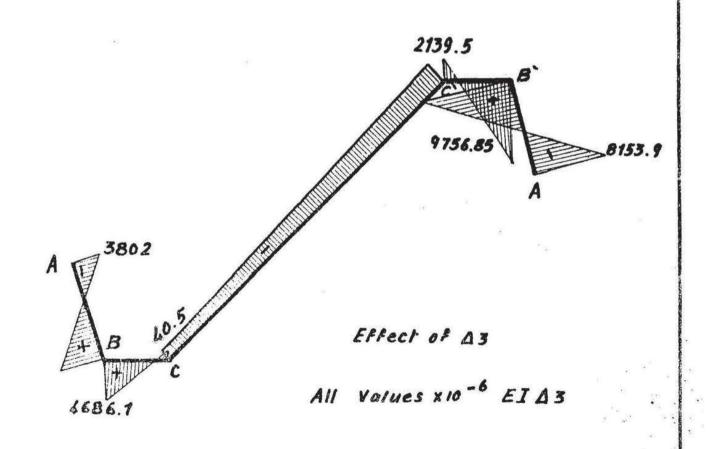
Stresses	due	ro	A 2	8

PL.	LD 6	B.M.5	0 -6	a	ь	с	ē	Б	ā
AB	25.91	.368	1280	995	<sup>+</sup> 710	51.5	22.5	5.0	2.0
BC	41.12	. 585	5200	650	1300	480	215	47.5	24.5
CE	Zero	0	0	0	0	0	0	0	٥
ĊĖ	41.12	.585	5200	24.5	47.5	215	480	1300	1650
BÀ	25.91	.368	1280	2.0	5.0	22.5	51.5	710	995
٤	-			^	1957.5		1157.5	4	0-1

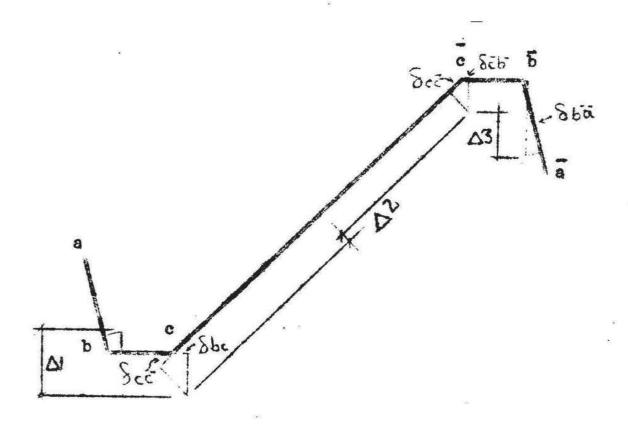


## Stresses due to 43:

PL.	LD. 6	B.M-3	0 _3 ×10	a a	P = 10	z To <sup>6</sup>	c	P	ā*10
AB	5 9.5V	83.5	2.9	2250	1610	116.5	50.5	11.35	4.5
ВС	108.3	1520	13.5	1700	3370	1240	555 555	123	63
Cč	103	1450	0.228	21.4	41.2	180	180	41.2	21.4
ĊB	200	2800	24. 9	116+	227	1020	2300	6200	3100
ΒÀ	135	1900	6.6	10.2	25.7	116	265	3650	5100
2				3802.4	4686_1	40.5	2139.5	9 756.85	8153.9



#### Geometrical Relations



 $\triangle$ I =  $\delta$ ab/cos 75 +  $\delta$ be tan 15° +  $\delta$  cc/cos 45+  $\delta$ be tan 45°

··. \$\text{\$\alpha\$} = 1.035 \$\delta\$ ab +1.268 \$\delta\$ bc +1.414 \$\delta\$ cc

 $\triangle$ 2=  $(\delta bs/cos 45 + \delta c\bar{c} tan 45)-(\delta \bar{c}b/cos 45 + \delta c\bar{c} tan 45)$ 

... △2= 1.414 Sbc - 1.414 Sbc

 $\triangle 3 = (\delta ba/cos15 - \delta bctan15) - (\delta bc tan 45+ \delta cc/cos45)$ 

 $\therefore \triangle 3 = 1.035 \& ba - 1.268 \& bc - 1.414 \& cc$  (A)

The values of 5 in terms of the relative displacements A:

The values of 3 in table 01 the 1.62271 color of the values of 3 in table 01 the 1.62271 color of the values of 3 in table 01 the 1.62271 color of table 24x9.3 
$$\times \frac{1^2}{E} - 10^6 (\frac{8153.9+9756.85}{\pi^2 x^2 4} \times \frac{1^2}{2^4 x^9 \cdot 3} + \frac{1618.6+1957.5}{77^2 x^2 4} + \frac{4686.1 \cdot 3802}{\pi^2 x^2 4} \times \frac{1^2}{\pi^2 x^2 4} + \frac{3576}{\pi^2 x^2 4} + \frac{1^2}{\pi^2 x^2 4} + \frac{3576}{\pi^2 x^2 4} + \frac{1^2}{\pi^2 x^2 4} + \frac{1^2}{\pi$$

 $+\frac{2139.5-40.5}{112.5 \times \pi^2} \triangle 3)$ 

$$... Sec = \frac{10^{14}}{112.5(9)} \frac{L^{2}}{E} + 10^{6} L^{2} (-\frac{2099}{112.5} \triangle 1 \frac{2099}{112.5} \triangle 3)$$

$$... Sec = 0.184 \frac{L^{2}}{E} + 10^{6} L^{2} (-1.88\triangle 1 + 1.88\triangle 3) I$$

$$... Sec = \frac{51.4}{15x9.3} \frac{L^{2}}{E} + 10^{6} (\frac{4726.6}{\pi^{7}x15} \triangle 1L^{2} + \frac{225.5}{\pi^{7}x15} \triangle 2L^{2} + \frac{11896.35}{\pi^{7}x15} L^{2}\triangle 3)$$

$$... Seb = 0.368 \frac{L^{2}}{E} + 10^{6} L^{2} (+31.8\triangle 1 + 15.20\triangle 2 + 80.5\triangle 3)I$$

$$... Seb = \frac{148.9}{2^{4}x9.3} \frac{L^{2}}{E} + 10^{6} (\frac{8488.1^{12}}{\pi^{2}x2^{4}} \triangle 1 + \frac{3576.0}{\pi^{2}x2^{4}} L^{2}\triangle 2$$

$$-\frac{17710.75}{\pi^{7}x2^{4}} L^{2}\triangle 3)$$

 $... \delta ba = 2.02 - \frac{L^2}{E} - 10 L^2 (35.80\triangle 1 + 15.10\triangle 2 + 75\triangle 3)I$ (B)

# Substituting the values of 8 in the Geometrical Relations:

From A & B
$$\triangle 1 = 0.117 \frac{L^2}{E} - 10 L^2 (77.9 \triangle 1 + 15.65 \triangle 2 + 37 \triangle 3) I$$

$$+ .815 \frac{L^2}{E} - 10 L^2 (101.5 \triangle 1 + 19.30 \triangle 2 + 40 \triangle 3) \times I$$

$$+ 0.26 \frac{L^2}{E} + 10 L^2 (-2.66 \triangle 1 + 0.0 \triangle + 2.66 \triangle 3) \times I$$

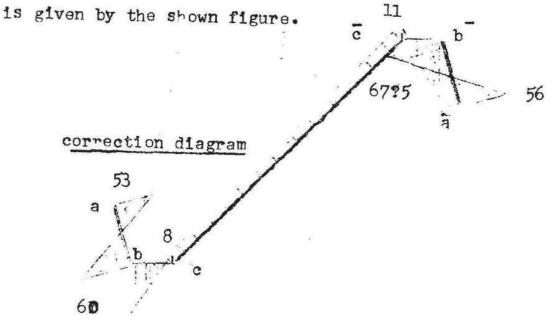
$$1.e. \triangle 1 = 1.192 \frac{L^2}{E} - 10 L^2 (182.06 \triangle 1 + 34.95 \triangle 2 + 74.34 \triangle 3) \times I$$

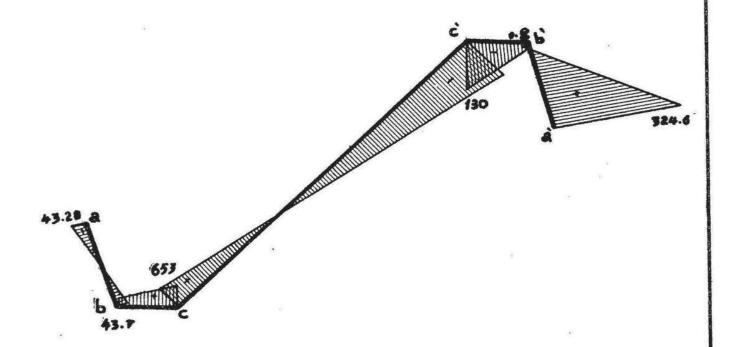
. 1.192 
$$\frac{L^2}{E}$$
 - 58.5 $\triangle$ 1 - 11.0 $\triangle$ 2 - 23.5 $\triangle$ 3=0 (1)  
 $\triangle$ 2= .917  $\frac{L^2}{E}$  - 10 $\frac{6}{L^2}$ (113.2 $\triangle$ 1+21.5 $\triangle$ 2+45 $\triangle$ 3)xI  
-.520  $\frac{L^2}{E}$  -10 $\frac{6}{L^2}$ (45 $\triangle$ 1+21.5 $\triangle$ 2 +113.2 $\triangle$ 3)xI  
1.e. $\triangle$ 2= .397  $\frac{L^2}{E}$  -10 $\frac{6}{L^2}$ (158.2 $\triangle$ 1+43.0 $\triangle$ 2+158.2 $\triangle$ 3)xI  
. 0.397  $\frac{L^2}{E}$  - 50. $\triangle$ 1 - 14.60 $\triangle$ 2 -50. $\triangle$ 3 = 0 (2)  
 $\triangle$ 3 = 2.095  $\frac{L^2}{E}$  -10 $\frac{6}{L^2}$ (37 $\triangle$ 1 +15.65 $\triangle$ 2 +77.9 $\triangle$ 3) xI  
-0.467  $\frac{L^2}{E}$  -10 $\frac{6}{L^2}$ (2.66 $\triangle$ 1+0.0 -2.66 $\triangle$ 3)xI  
1.e. $\triangle$ 3 = 1.368  $\frac{L^2}{E}$  -10 $\frac{6}{L^2}$ (74.34 $\triangle$ 1+34.95 $\triangle$ 2+182.06 $\triangle$ 3)xI

.. 1.368  $-\frac{L^2}{2}$  23.50 $\triangle$ 1 - 11.0  $\triangle$ 2 -58.5 $\triangle$  3=0

From these equations we get:

The correction diagram associated to this case of loading is given by the shown figure 11





Final Stresses

loads applied at points c.b

## Displacements

$$\delta ob = 0.113 \times \frac{375}{300000} - \frac{60 + 49}{24 \text{ Tf}^2} \times \frac{378}{300000}$$

0.171 Cm

$$\delta bc = 0.648 \times \frac{375}{300000} - \frac{60 + 8}{75 \pi^2} \times \frac{37.5}{300000}$$

.090 Cm

$$\delta c \tilde{c} = 0.184 \times \frac{375^2}{300000} + \frac{11 - 8}{112.5 \, \text{m}^2} \times \frac{375^2}{300000}$$

.0888 cm

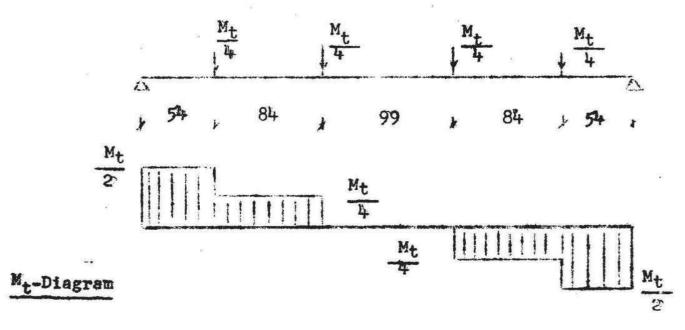
$$\delta \dot{b} \dot{c} = 0.368 \times \frac{375}{300000} + \frac{11+67.5}{15 \pi^2} = \frac{375}{300000}$$

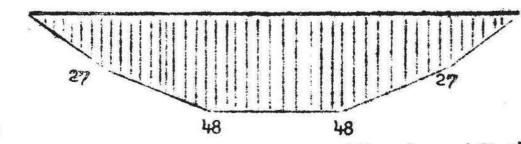
= 0.425 Cm

$$\delta \vec{b} \vec{a} = 2.02 \times \frac{375^2}{300000} - \frac{123.5}{24 \pi^2} \times \frac{375^2}{300000}$$

= 0.713 cm

## Torsion Analysis and Comments





#### Ø-Diagram

All valuesx.468x108

but we have 
$$G = \frac{E}{2(1+m)} = \frac{E}{2 \times 1.365} = 0.428E$$

$$\frac{3 \text{ Mt}}{5143.5 \times 0.428 \times 300000} = 0.463 \times 10^8 \text{ Mt}$$

$$M_t = 1000 \text{ P x } 47.3 = 47300 \text{ P kg.cm}$$

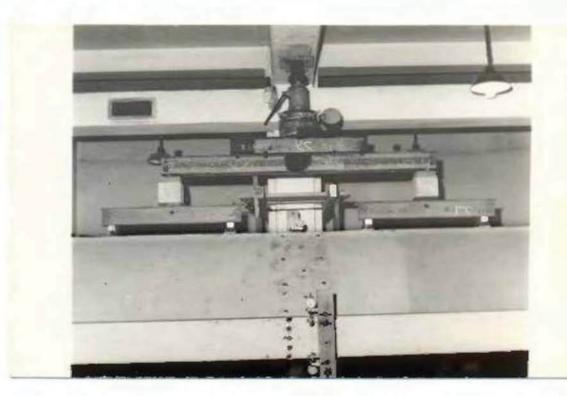
## Comments:

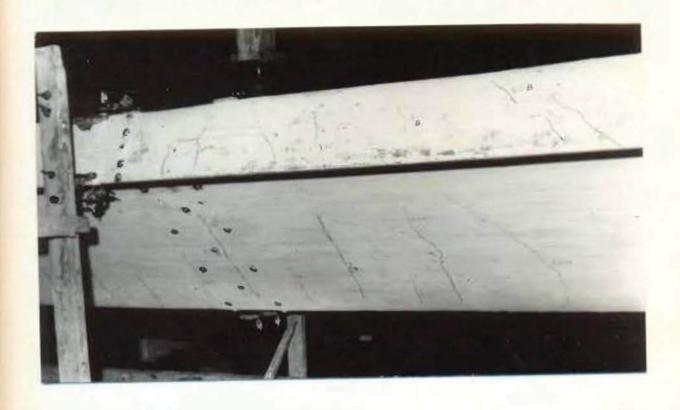
It is concluded from the next table
that the application of topsion analysis
in such cases is not correct.

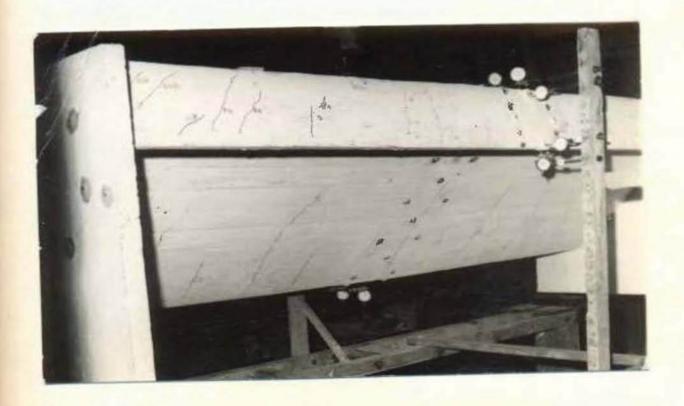
Lower egges move upwards while
tests show that it will move
downwards.

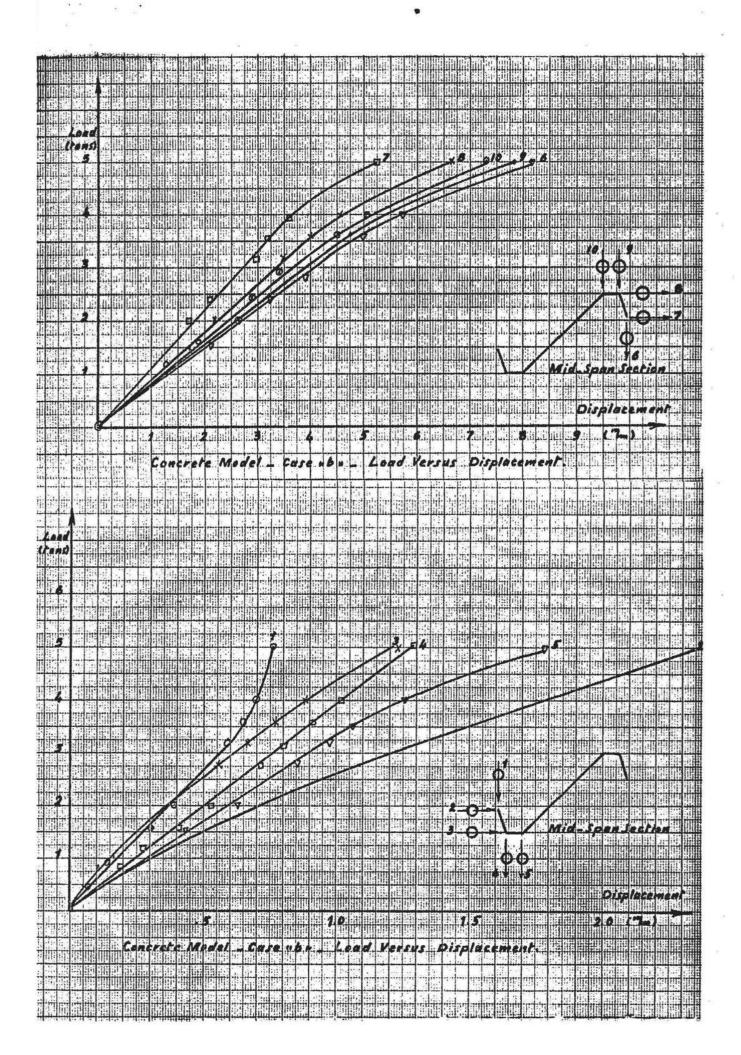
point	×	У	$R = \sqrt{x^2 + y^2}$	tan <b>g</b> = y/x	P=RD	x1 =	Y1= P X/R
A.	61.02	16.6	63.40	16.6	0.666		0.64P ↑
B	54.8	39.8	67.73	39.8 54.8	0.710	0.418P	0.575P
C	39.8	39.8	56.25	1	•59	0.417F	0.41791

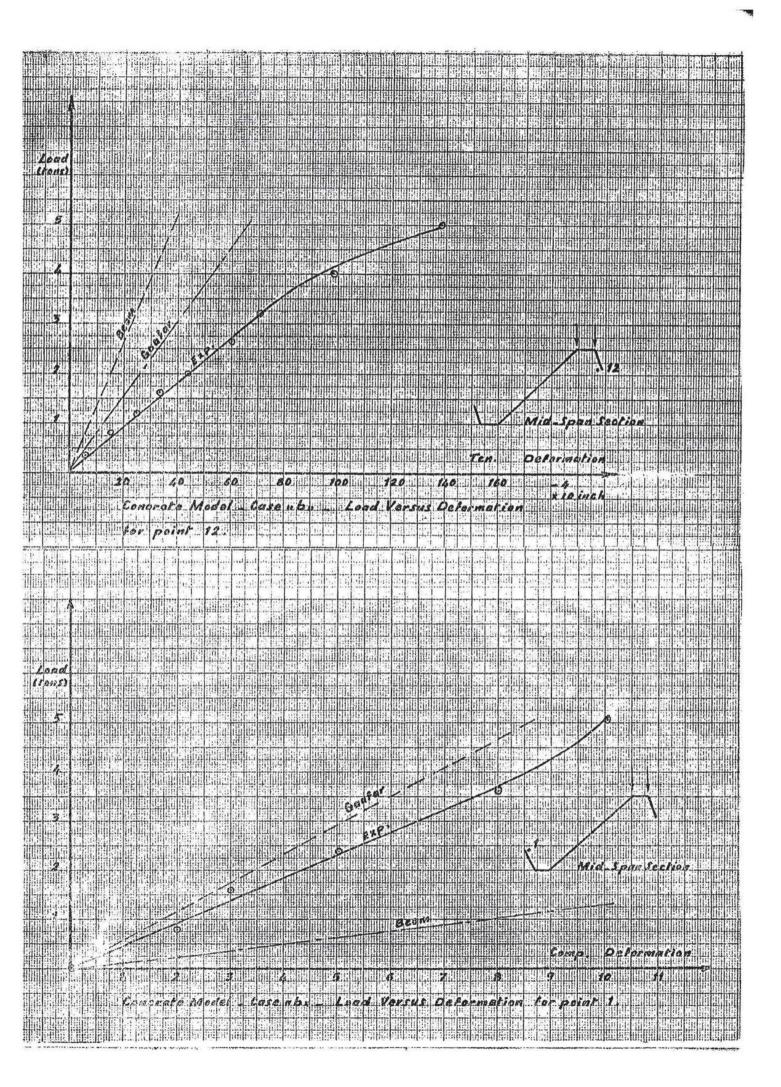
## - Experimental Results

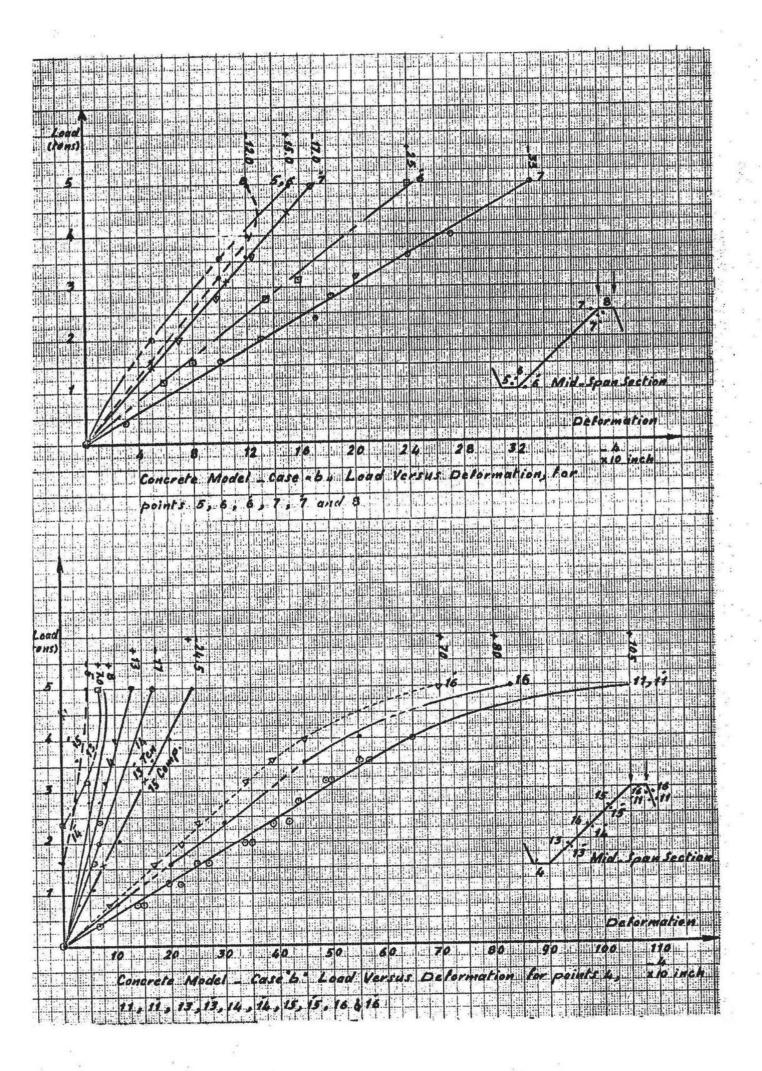


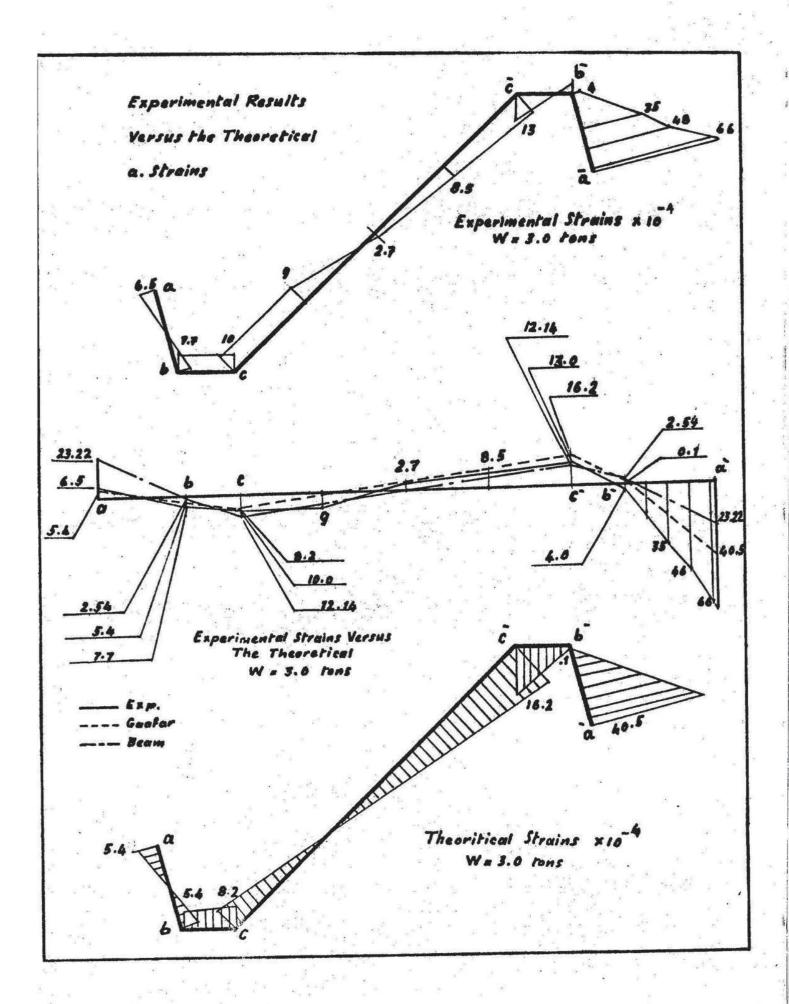


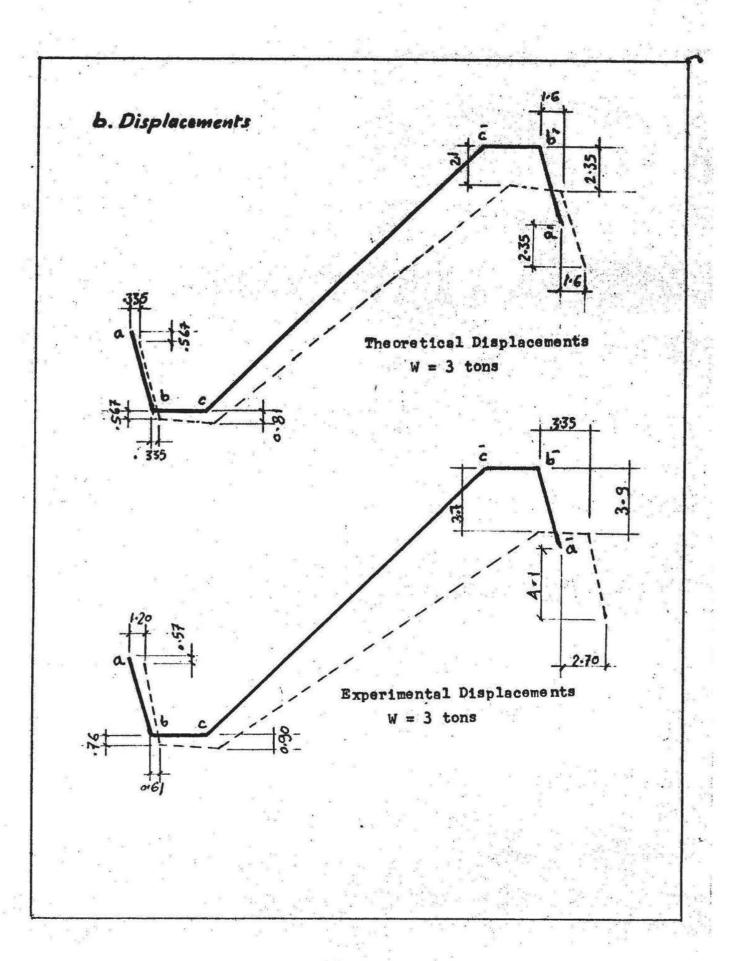






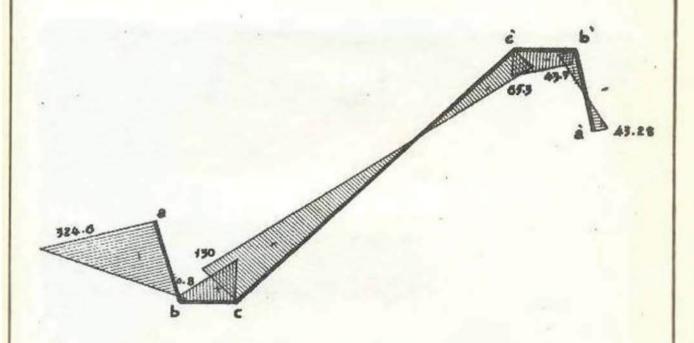






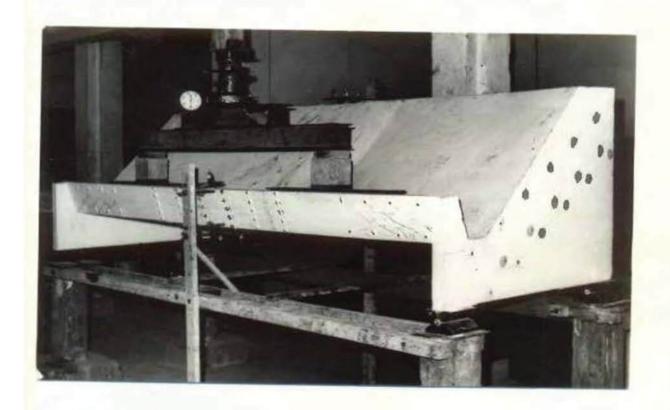
#### c. Unsymmetrical loading (case c)

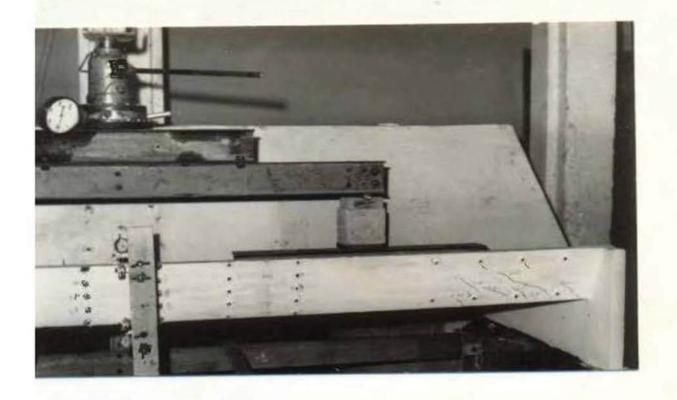
In this case stresses and displacements can be concluded directly from the previous case "b" .

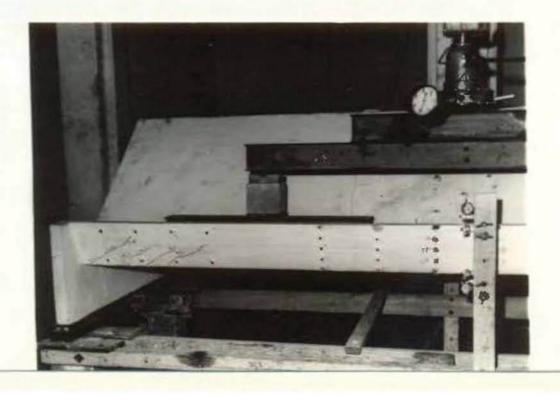


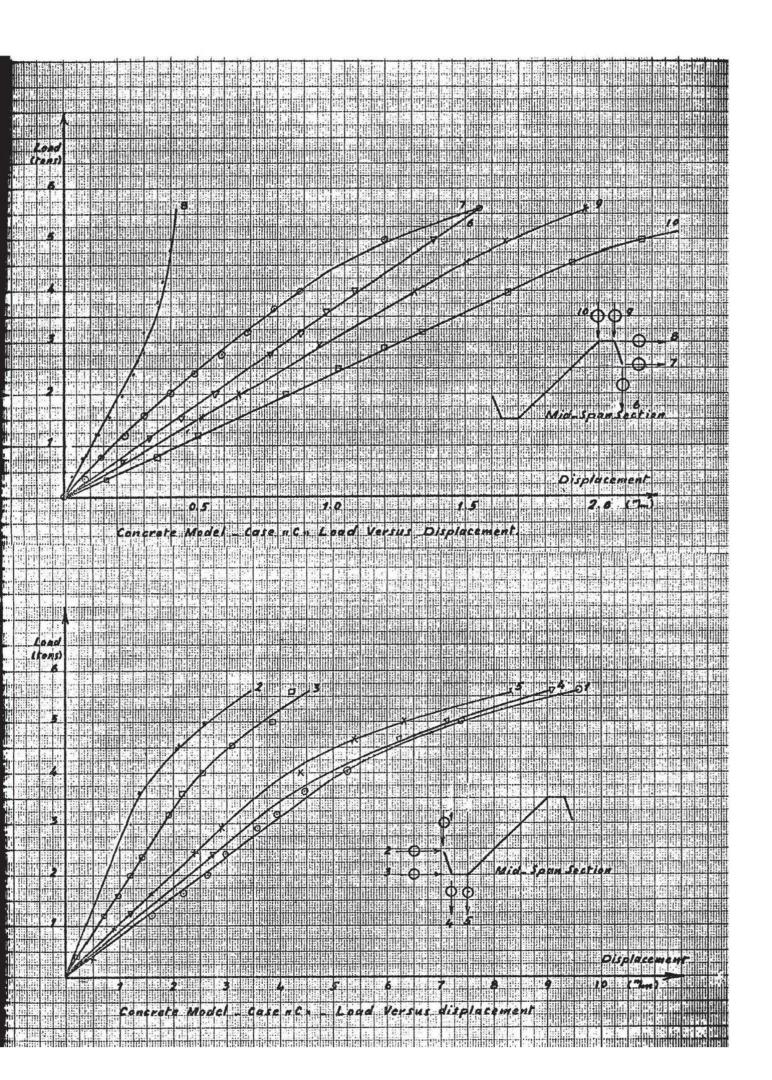
Final Stresses loads applied at points b.c

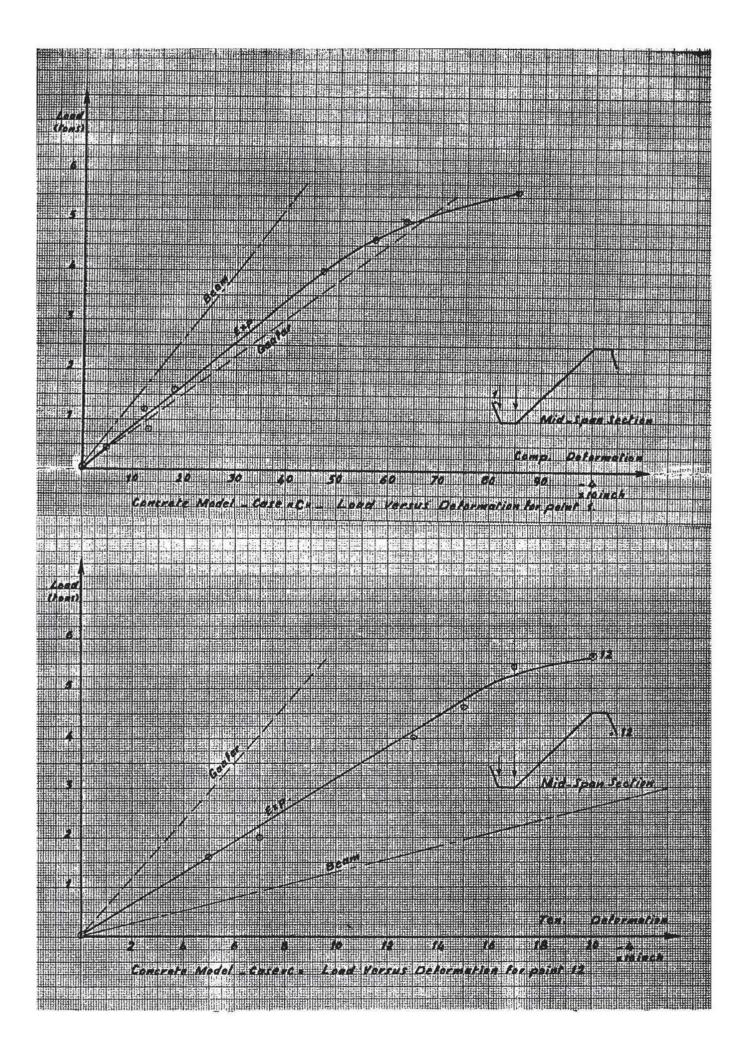
## - Experimental Results

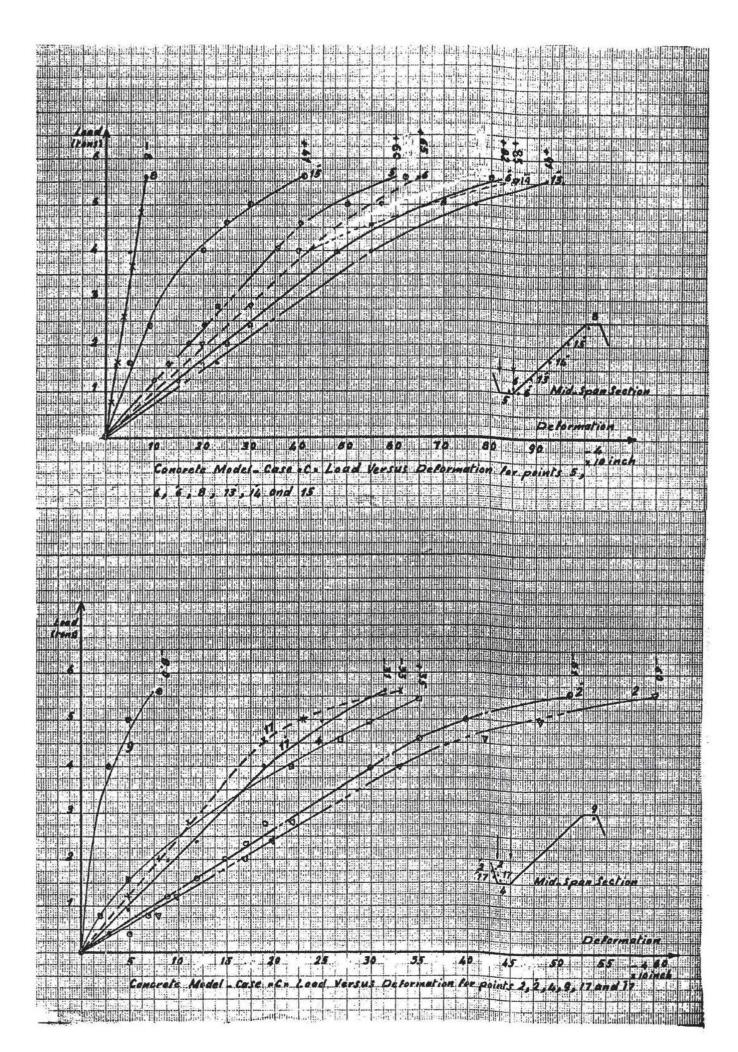


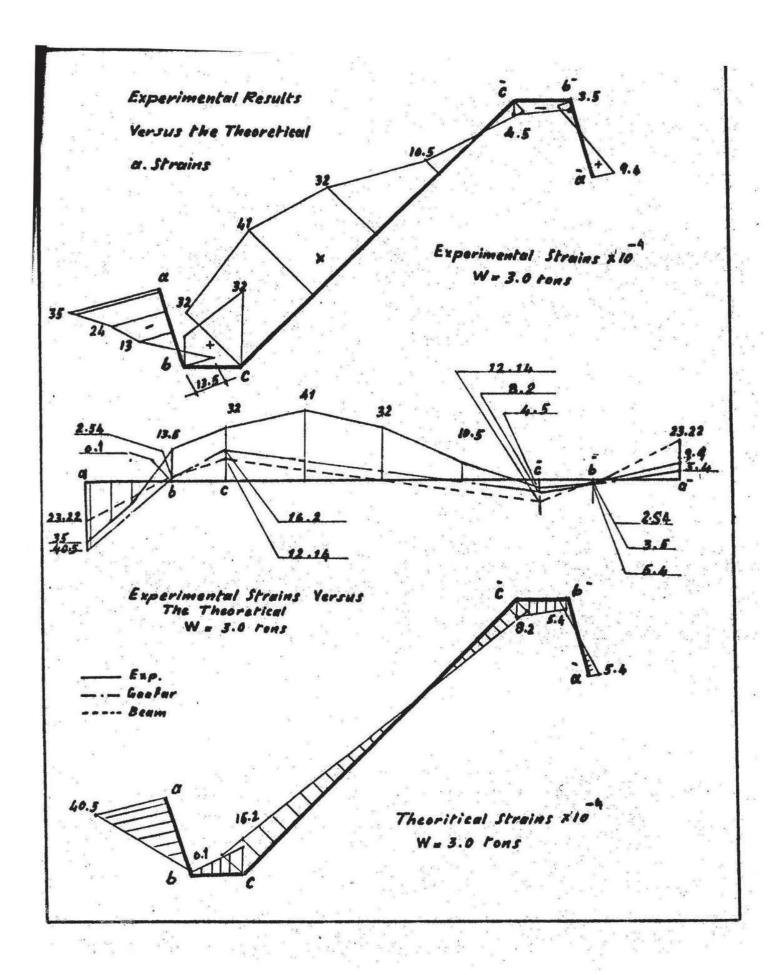


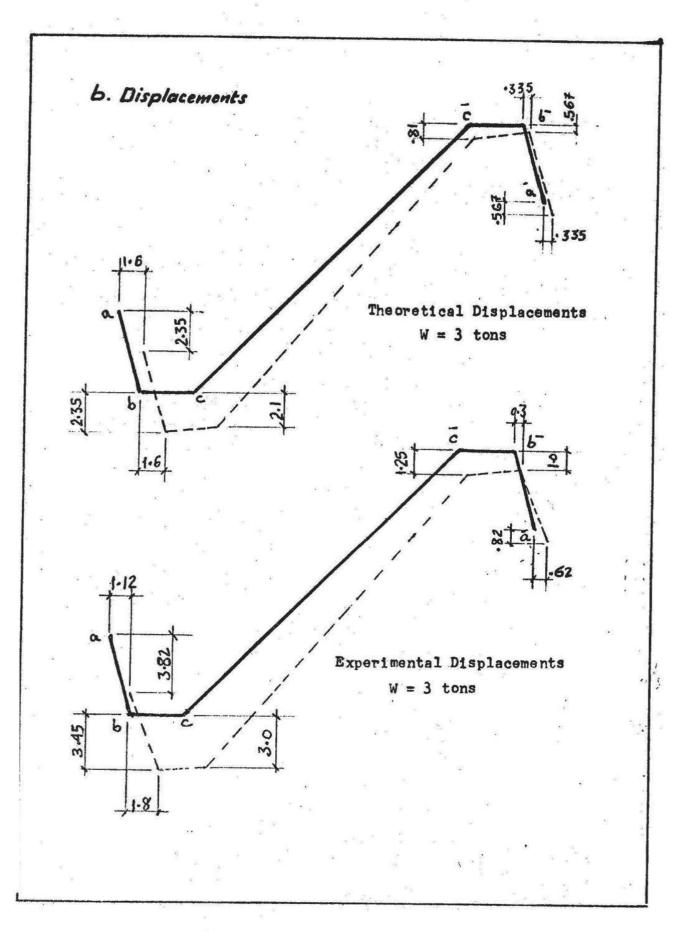












# d- General Solution based on (Gaafar's Methods)

A more general solution is introduced. This solution:
depends on correcting the stresses produced by each case
of plate loading. The advantages of this method appear
when solving a given problem for different cases of loadings

The solution is summerized in 3-steps:

#### 1. Elementary analysis:

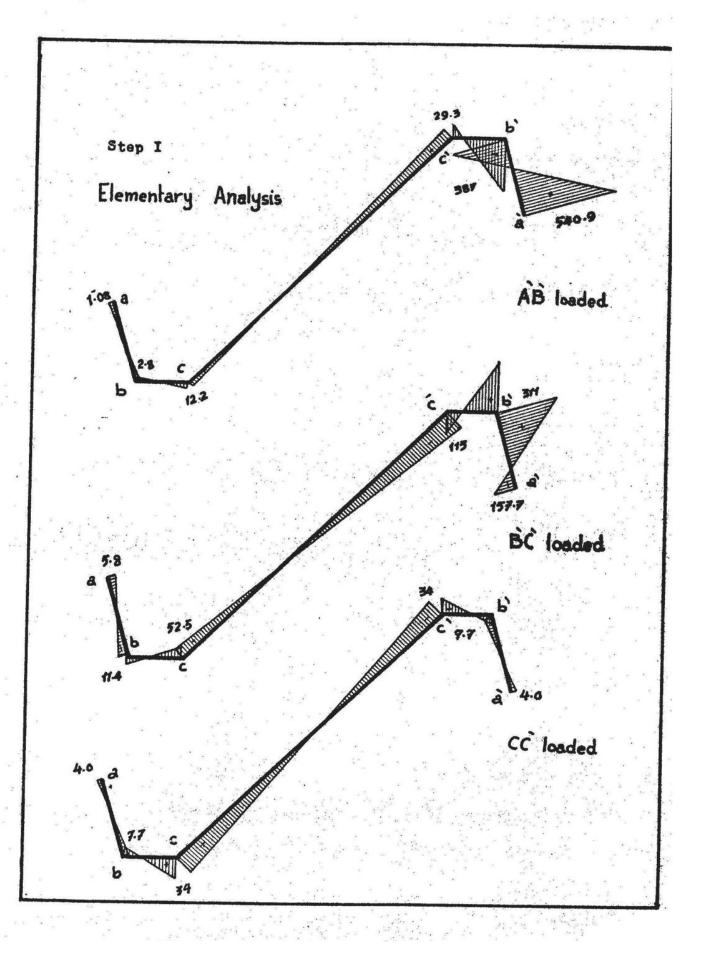
Where the stresses are calculated for any arbitrary values of each plate loading. These values were chosen to give the solution of the unsymmetrical case as a check.

### 2. Correction analysis:

In this case we make use of the pre-calculated values of stresses corresponding to  $\Delta$  1,  $\Delta$  2, and  $\Delta$  3 values .

## 3. Final stresses:

It is obtained by summing the stresses of steps 1 and 2. as in the previous solutions.



## Step 2 : CORRECTION ANALYSIS

Plate a b loaded i

 $\delta_{0} = 0.017_{T}^{L^{2}} + L^{2} \times 10^{-6} (-75 \Delta 1 + 15.1\Delta 2 + 35.8\Delta 3) \times 10^{-6}$   $\delta_{0} = .108_{T}^{L^{2}} + L^{2} \times 10^{-6} (-80.5\Delta 1 + 15.2\Delta 2 + 31.8\Delta 3) \times 10^{-6}$   $\delta_{0} = .0395_{T}^{L^{2}} + L^{2} \times 10^{-6} (-1.88\Delta 1 - 1.88\Delta 3) \times 10^{-6}$ 

 $\delta \delta \tilde{c} = 2.98 \frac{L^2}{4} + L^2 \times 10^4 (31.8 \Delta 1 - 15.2 \Delta 2 - 80.5 \Delta 3) \times I$   $\delta \delta \tilde{a} = 4.17 \frac{L^4}{4} + L^2 \times 10^4 (35.8 \Delta 1 - 15.1 \Delta 2 - 75 \Delta 3) \times I$ 

## GEOMETRICAL RELATIONS :

A1 = 1.035 Sob + 1.268 She + 1.414 Sec

AZ = 1.414 SEC = 1.414 SEC

A3 . 1.035 Sab + 1.268 Sbc + 1.414 Scc

Substituting the values of & in the geometrical relations:

A1 = .210 - 57.5 A1 + 11.0 A2 + 23.5 A3

Δ2 = 3.63 + 50 Δ1 - 13.6 Δ2 - 50 Δ3 2

Δ3 = 8.1358 + 23.5Δ1 = 11.0 Δ2 = 57.5 Δ3 3

Solving these equations we get

 $\Delta_1 = -0.0568 \frac{L^2}{E}$ 

A2 = -. 955 L

Δ3 = +.296 £

## Plate be loaded

 $\delta ab = 0.077 \frac{L^2}{8} + L^2 \times 16^4 (-75 \Delta 1 + 15.1 \Delta 2 + 35.8 \Delta 3) \times I$   $\delta bc = .458 \frac{L^4}{8} + L^2 \times 10^{-6} (-80.5 \Delta 1 + 15.2 \Delta 2 + 35.8 \Delta 3) \times I$   $\delta cc = 0.159 \frac{L^4}{8} + L^2 \times 16^6 (-1.88 \Delta 1 - 1.88 \Delta 3) \times I$   $\delta cb = 3.05 \frac{L^4}{8} + L^2 \times 16^6 (35.8 \Delta 1 - 15.2 \Delta 2 - 80.8 \Delta 3) \times I$   $\delta ba = 2.1 \frac{L^4}{8} + L^2 \times 16^6 (35.8 \Delta 1 - 15.1 \Delta 2 - 75 \Delta 3) \times I$ 

## GEOMETRICAL RELATIONS

A1 = 1.035 Sab + 1.26 Sbc + 1.414 Sce

A2 a 1.414 866 - 1.414 860

A3 = 1.035 Sat + 1.26 Sec + 1.414 Sec

Substituting the values of & in the geometrical relations:

Solving these equations we get :

$$\Delta 2 = 0.00925 \frac{L^2}{E}$$

## Plate cé louded :

 $\delta ab = 0.052 \stackrel{!}{=} . L^{2} * 10^{-6} (...75 \Delta 1 * ...35.8 \Delta 3 ) \times I$   $\delta bc = ...298 \stackrel{!}{=} . L^{2} * 10^{-6} (...80.5 \Delta 1 * ...35.8 \Delta 3 ) \times I$   $\delta c\bar{c} = 0.0645 \stackrel{!}{=} . L^{2} * 10^{-6} (....1.88 \Delta 1 * ...1.88 \Delta 3 ) \times I$   $\delta \bar{c} \bar{b} = 0.298 \stackrel{!}{=} . L^{2} * 10^{-6} (...35.8 \Delta 1 * ...80.5 \Delta 3 ) \times I$   $\delta b\bar{a} = 0.052 \stackrel{!}{=} . L^{2} * 10^{-6} (...35.8 \Delta 1 * ...75 \Delta 3 ) \times I$ 

### GEOMETRICAL RELATIONS:

A1 = 1.035 Sab + 1.268 Sbc + 1.414 Sec

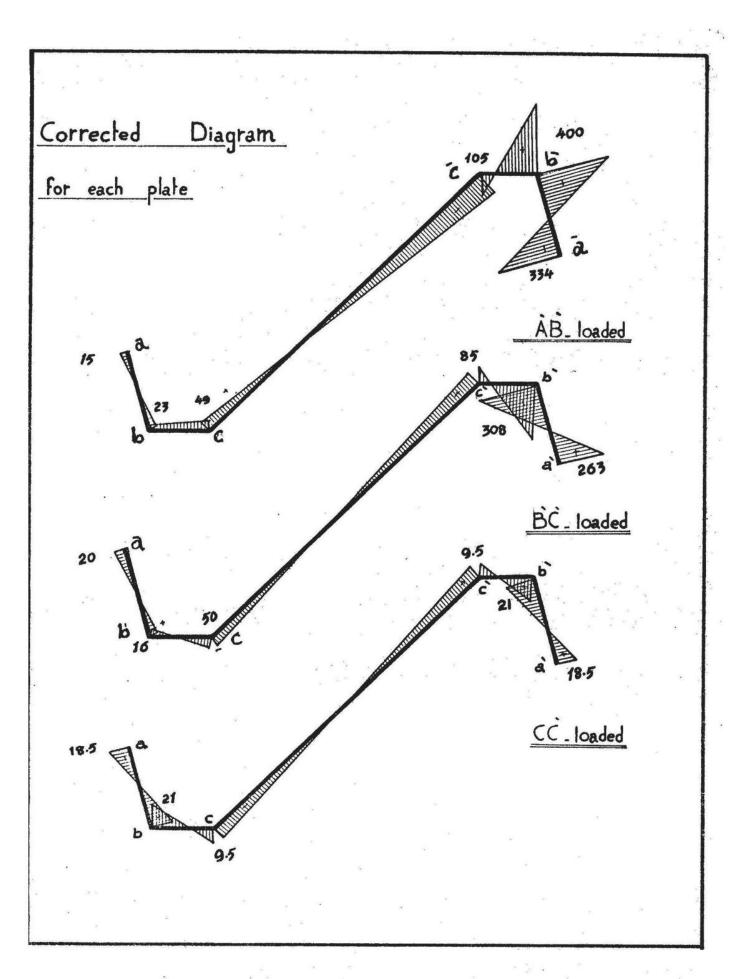
A2 = 0

43 = 1.035 Sab + 1.268 See + 1.414 See

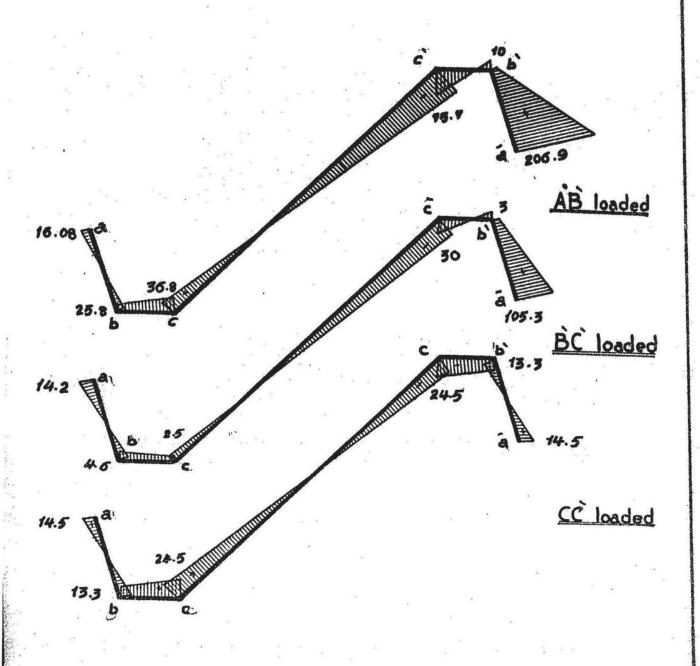
Substituting the values of & in the geometrical relations.

A = 0.523 4 - 57.5A + 23.5 A

leΔ = 0.523 4 = 34 Δ



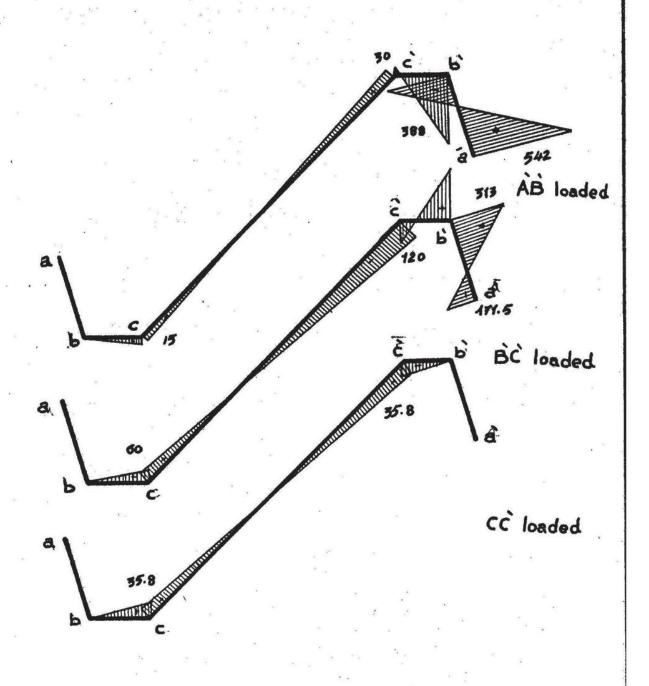
# Final Stresses

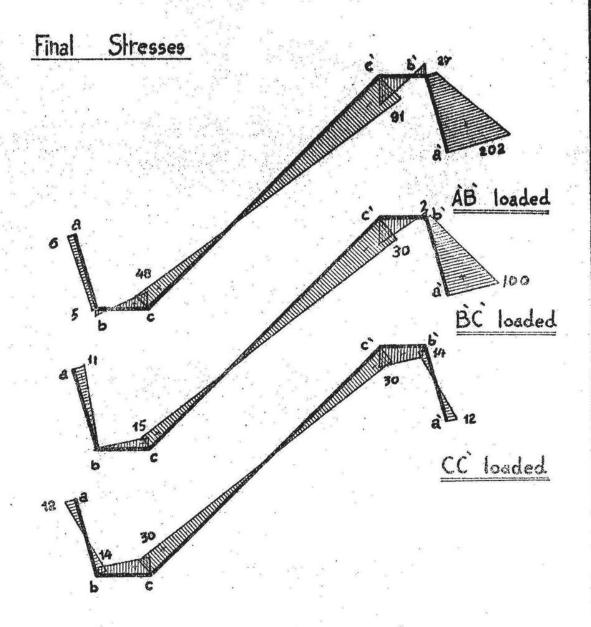


#### e - Proposed Simplifications

Engineers always tend to simplify complicated problems. Terms of small effects are neglected. Refering to the general solution we show that stresses induced in plates ab or be when plate ab is loaded are very small. So these two plates are omitted. In general, when loading any plate, not far than two plates are considered. The results obtained using this simplification are acceptable. The deviations from the usual solution do not exceed five percent as shown in the next sheets.

# Elementary Analysis





## CHAPTER IV

THE FINITE ELEMENT METHOD

2

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5

6

#### BRIEF STUDY OF THE FINITE ELEMAN METHOD

theoretical

The finite element method is also used for the theoretical investigation of stresses and deflections allowed the
Separated
domain. The structure is separated by imaginary lines into
a number of finite elements Fig.(4.1). The elements are
assumed to be interconnected at a discrete number of
nodal points situated on their boundaries. The displacements of (these nodal points are interpreted as the actual
displacements of the corresponding points on the
structure.

If the displacements at any point within an element 4
"e" can be expressed in terms of the nodal known displacements, in the form of

\langle f(x,y)\rangle = N. \langle
where \langle f\rangle, \langle \rangle: represent possible movement of a
typical point within the element and
the corresponding displacement of a
node respectively.

Then with the aid of this function, both strains and stresses at any can be determined using the following relationships:

$$\{\xi\} = [B] \cdot \{\xi\}$$
  
 $\{G'\} = [D] \cdot \{\xi\}$ 

Whore {B]is the strain matrix

[D]is the clasticity matrix

And also the stiffness properties of the assembled structure made up from the idealized elements can be put in the form

In the problem of folded plate a difficulty arises when all the elements joining at a particular node are in one plane, because in globle coordinates six equations that are singular are obtained. This is due to the fact that only five of the equations can then be independent, due to the omission of the rotation perpendicular to the plane. For such nodes, the assembly should be made in the local coordinate system.

This means that the equilibrium equations written for the nodal points are referred to two systems of axes.

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First for all nodal points ituated at the fold lines, the equilibrium equations relating forces to displacements global at every point are referred to one globle system X,Y and Z axes and the available dogroes of freedom per node are equal to six. The second for all nodal points situated at one common plate, the equilibrium equations are referred to local system X,Y & Z axes corresponding to this plate. The available dogroes of freedom per node for this case are equal to five. The transformation process adopted to relate nodal points with five dogrees of freedom with other nodal points with six degrees of freedom are as follows:

1- In the type number one where the nodel point"r"
lies on fold line and "S" lies on plane plate the
equilibrium equation is

$$\left\{F_{r}^{\prime}\right\} = \left[L\right]^{T} \left[K_{rs}\right] \left\{S_{s}^{\prime}\right\}$$

2- In type number two where nodal point "r" lies on plane plate and "s" lies on fold line the equilibrium equation is:

$$\{F_r\} = [K_{rs}] [L] \{\delta s\}$$

2

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3- In the type number three where nodal points "r" and "s" lie on common plate the equilibrium equation is.

$$\{F_r\} = [K_{rs}] \{Ss\}$$

4- In the type number four where nodal points "r" and "s" lie on the fold line the equilibrium equation is

$$\{\mathbf{F}_{\mathbf{r}}^{\mathsf{T}}\} = *[\mathbf{L}]^{\mathsf{T}} [\mathbf{K}_{\mathbf{r}\mathbf{s}}][\mathbf{L}] \{\delta s\}$$

Where L is given by

[L]= 
$$\lambda_{xx}$$
  $\lambda_{xy}$   $\lambda_{yz}$   $\lambda_{yz}$   $\lambda_{zz}$   $\lambda_{zx}$   $\lambda_{xy}$   $\lambda_{xz}$   $\lambda_{xy}$   $\lambda_{xz}$   $\lambda_{yx}$   $\lambda_{yx}$   $\lambda_{yy}$   $\lambda_{yz}$   $\lambda_{zx}$   $\lambda_{zy}$   $\lambda_{zz}$ 

In which is the direction cosine between axes.

For the right hand system of axes shown in Fig. (4-2).

direction cosines of X axes are:

$$\lambda_{xx} = 1$$

$$\lambda_{xx} = 0$$

$$\lambda_{xx} = 0$$

The direction cosines of the y axis in terms of the co-ordinates of the various model points are :

$$\lambda_{yx} = 0 
\lambda_{yy} = \frac{\bar{y}_1 - \bar{y}_1}{\sqrt{(\bar{z}_1 - \bar{z}_1)^2 + (\bar{y}_1 - \bar{y}_1)^2}}$$

$$\lambda_{yz} = \frac{\bar{z}_1 - \bar{z}_1}{\sqrt{(\bar{z}_1 - \bar{z}_1)^2 + (\bar{y}_1 - \bar{y}_1)^2}}$$

Similarly the direction cosines of the Z axis are

$$\lambda_{z\bar{x}} = 0 \qquad \dot{z}_{j} = \dot{z}_{1}$$

$$\lambda_{z\bar{y}} = -\frac{\bar{y}_{1} - \bar{y}_{1}}{\sqrt{(\bar{z}_{j} - \bar{z}_{1})^{2} + (\bar{y}_{j} - \bar{y}_{1})^{2}}}$$

$$\lambda_{z\bar{z}} = \frac{\bar{y}_{1} - \bar{y}_{1}}{\sqrt{(\bar{z}_{j} - \bar{z}_{1})^{2} + (\bar{y}_{j} - \bar{y}_{1})^{2}}}$$

The main stops required for the analysis are:

1- The space stiffness matrix of element(KE) is

domposed of number of sub-matrices. Each of these
matrices is formed from the inplane and out of plane
stiffness matrices (KP & KB).

	7	2	2	),				X
KE=	± .	· 	777	Т		KP 2×2	0	0
		5	8	7' 9	for ex. =		<b>7</b> 70	
	2.5					0	<b>KB</b> 3×3	0
				10	- Alexander	0	0	C

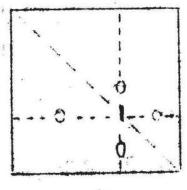
but sing the foregoing equilibrium equations, each submartix is transformed to the global axes according to its type.

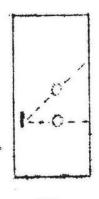
3- The space rectangular element in the global axes(K1) formed in step two can be assembled to form the globle stiffness matix (KK). This matrix is banded to have a reasonable core size in the computer:

4 - Imposing the boundary conditions:
5 - The results.

the

Clearly, without substitution of a minimum number of prescribed displacements to prevent rigid body movement of the structure, it is impossible to solve this 3 system. A unique solution will be presented by multip-4 lying raw and column corresponding to each prescribed support displacement by zero number. The diagonal coefficient of the matrix(K) for the prescribed displacement is developed by unity number. This is equivalent to reducing the number of equilibrium equations.





The boundary condition used for the folded plate shown in Fig. (4-1) are:

- For nodal points on the end diaphragm:

  Linear displacements:  $u \neq 0$ , v = 0 & w = 0Rotations:  $O_X = 0$ ,  $O_Y \neq 0 & O_Z \neq 0$
- b- For nodal points on axix of symmetry parallel to y-axis:

Linear displacements: u = 0,  $v \neq 0$  &  $w \neq 0$ . Rotations: u = 0,  $v \neq 0$  &  $v \neq 0$ 

The program posseses a special technic to form the global banded stiffness matrix.

The vertical of any node say "j"
Summing
is determined by the summation of the
degrees of freedom from joint one to
the joint number (J-1). The moint "j"
will occupy a distance in the y direction
equal to its degree of freedom.

Globał banded matrix

The abscissa of the node "21" for example in the Fig.(4-1) element 12-13-21-20 will begin by summing the degrees of freedom from joint 12 till joint 20 i.e.

(5+5+6+6+5+5+6+6+5)=49 and this node will occupy a

distance in the x direction equal to its degree of freedom. For any element the maximum abscissa is called the band width.

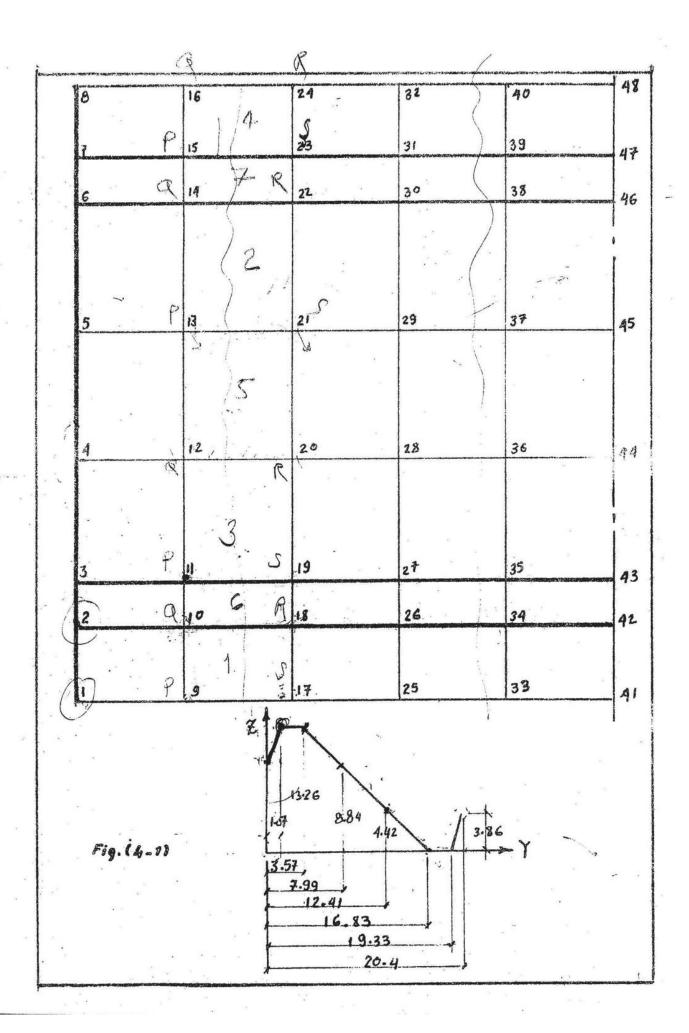
The input data containes nodal number, hodal point local and global co-ordinates, number of element types, humber of elements of the same type, nodal loads.

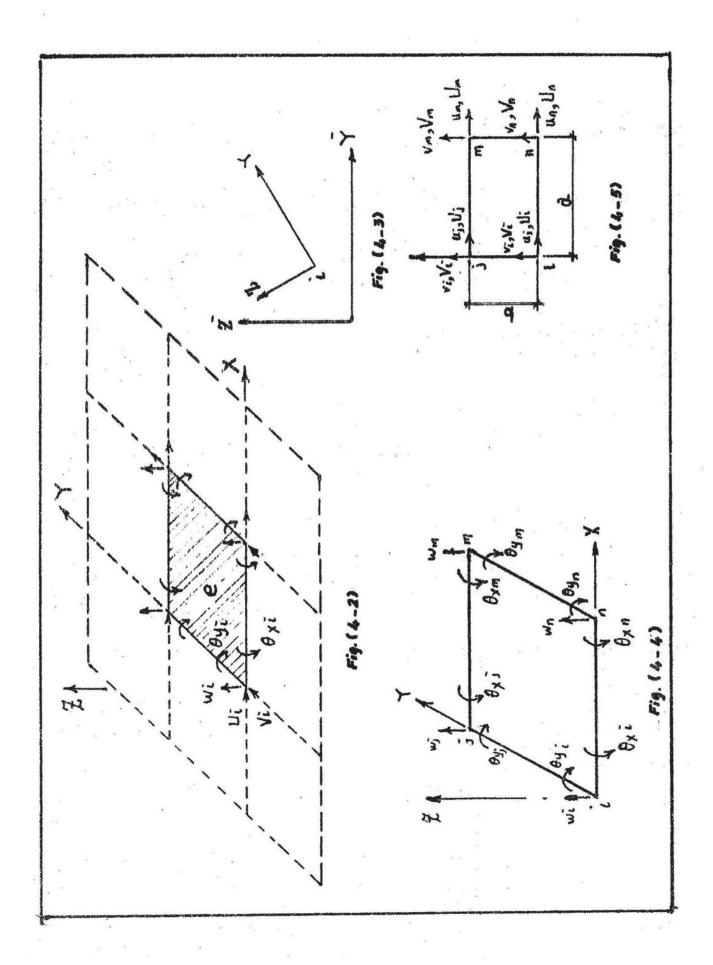
Material properties and boundary conditions:

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The output data give nodal points in-plane and out 4 of plane displacements and rotations. The displacements, of nodes on fold lines are referred to the global axis while for the other nodes, the displacements are referred to the plate local axes, displacements of nodes on fold lines are then transferred to the local axis of each of the adjoining plates. Stresses x, y & xy at different nodal points are calculated. Also the moments Mx, My & Mxy are calculated at all nodal points.





1026	025	12	025	170	020	019	018	017	016	015	014	015	012	011	010	009	800	007				9					066	2005	004	003	062	007	
9													1								3												
) -/t	·	2		, 20	0		30	2,	7,	<b>.</b>		<b>20</b>			••	-	20	3							.XX		\ <u>\</u>	<b>–</b> (	2 1		<b>D</b>		
FAIN		A1(12	EAD(1,2)		UN/AR	IMENSION	0	6) A1	_	ENSION	TEGERRA	AL KT	*	VIEGERSO!	WIEGER R	NTEGER H.	EAL MUA,	ASTER TE	•		7						0	ACE C	-	NTUT INCRO	RUGRAMCAP	SEND TO	
			MUA.E.TH	WBJ . WBT	7	\$1(56,1),	,6), JL(48),	(6,6),81(6	1H (01) 13	NBL(7).	W.COL.BANDS		W.R.S.Z.TP1	OH,	SOR D, S	TS, HH,	CK, KP, KB.	371					**			+			Pu/160		MU)	SENICOMP	
,				NBJEX, LP	·	1,82156,1	SZ(48.6) . RL (30	16) K1(1	(16), HR(30), COL(96), RA	1(8,7),1			P1, [P2, TH3, TP	, R	D. SUNT . SUNZ . SUN3	1	KE, LS		!													ILED. ZZZ	
		9		'TP1.TP2	D DIS/SS		.RL (30,6	U.6.6),D	196) RAW	£(8,7),1			141 SE		SUN3, SUN4					V.												2)	3.7
				, TB3, TP4	(265,1),	6,11,84(56	,6),08(265,	1610,6,6	(94) KPC	3(8,7),1			4		N4 SUMA							***	¥ 4 3					*					
	*		13		25(265,1	1),85	3	0	10.2.216	4(8,7).X		•			SUM2, SUM	* *		: c				1	<u> </u>			4 *	4						100
			•			19156	5,5)	(10,6,6),	2.2) (KB(10.3.3) (KE(10	(16) V(16) X1					UM2, SUM3, SUM4, SUM5, SUM6		1		T.	٠	9			(3)						v. 7			¥?
						(56,1)			KECTO	6) X1 (16			. 3	fix e	UM5 . SUM6						,					50							
						4				1	Б  							4															-

he uses .41 3 FURNAL (2010) READ(1,1)((HH(S),(RL(S,1),1=1,6)),S=1,NBJFX) 1) 62 DU 6 T=1, NBJ 660 113" 6 RAW(1)=5 RAW(2) . RAW(3) . RAW(6) . HAW(7) = 6 051 DU 7003 I=8,40,8 136 7005 RAW (3+1) , RAW (7+1) , RAW (2+1) , RAW (6+1)=6 135 2/2 50R=0 134 DU 12 1=1, NBJ 935 12 SOR=SUN+RAW(1) 1136 1)57 BANU6=56 DU 14 1=1,50R 1138 F(I,1) +ZS(I,1)=0.0 039 040 DO 13 J=1. BANDO 1)41 KK(T, J)=0.0 042 CUNTINUE 045 VAL=U.0 1)44 TOUR DU ZUUU Z=1.NBJ 045 p=11(1,2) 046 G=15(1.5) 047 R=15(1,Z) 048 5=14(1.Z) REAU(1,4)X(P),Y(P),X(4),Y(4),X(R),Y(g),X(S),Y(S) 049 050 4 FORMAI (BFU. 0) 051 READ(1,2)X1(P),Y1(P),41(P),X1(Q),Y1(Q),Z1(Q),X1(R),X1(R),Y1(K),Z1(R), 1 X 1 (\$) , Y 1 (\$) , Z 1 (\$) 952 BB=Y(4)-Y(P) 055 AA=X(5)-X(P) 154 D 055 B=BB/AA 056 BP=AA/BB (((S\*\*AUM)-1)\*, \$7)/(HI\*#3)1) 057 AB=(6\*(TH\*\*3))/(12.\*(1.\*(MUA\* Ab=6/LAA\*B\*(1.\*(MUA\*\*2.))) OSH 050 DX . DY 4 ((E+(TH++3))) (14. +(1. -(MUA++2)))) 060 DXY=((1.-MUA)/4.)\*DX ~ 061 DUT=MUA+DX 002 IF (VAL. EQ. 1.0) GO TU 0000 065 6000 CONTINUE 064 KP(1,1,1),KP(5,1,1),KP(8,1,1),KP(10,1,1)=4.\*B\*A+((2./B)\*(1.-MUA)) 065 066 KP(1,1,2),KP(1,2,1),KP(7,1,2),KP(7,2,1),KP(8,1,2),KP(8,2,1)= 067 1(3./2.) \*(1.+MUA) \*A 068 KP(1,4,2),KP(5,2,2),KP(8,4,2),KP(10,2,2)=((4,/8)+(2,\*8\* 069 1(1.-MUA)))\*A 070 KP(2,1,1),KP(9,1,1)=((2,\*B)-((2,/B)+(1-MUA)))\*A 071 . KP(2,1,2),KP(4,2,1),KP(6,1,2),KP(9,1,2)=(-5,/2,)+(1,-3,+MUA)+A 076 KREZ. (1) , KP(4,1,2) , KP(6,2,1) , KP(9,2,1) = (3./2.) \* (1.-3. \*MUA) \*A . 175 074 KP(2,2,2),KP(9,2,2)=((-4./8)+(1.-MUA)\*B)\*A 0/5 XP(3,1,1), KP(/,1,1)=(-(2,+8)-((1,-MUA)/8))+A KP(3,1,2),KP(3,2,1),KP(5,1,2),KP(5,2,1),KP(10,1,2),KP(10,2,1) 076 1=((-3./2.)\*(1.+MUA))\*A 077 KP(3,4,2),KP(1,2,2)=(-(2,/8)-((1,-MUA)+8))+A 0/8 KP(4,1,1),KP(6,1,1)=(-(4.\*8)+((1.-MUA)/B))\*A 079 XP(4,2,2), KP(6,2,2)=((2,/8)-(2,\*B\*(1,-MUA)))\*A 080 KB(1,1,1), KB(5,1,1), KB(8,1,1), KB(10,4,1)= (54, (AB+(4.\*(B\*+2+(1./(B\*\*237)+(1./5.3\*(4.-(4.\*MUA)))) 081 082 KB(1,4,1),KB(1,1,2),KB(10,4,1),KB(10,1,2)= 085 1((2./(B\*\*2))+(7./5.)\*(1.+(4.\*MUA)))\*AB\*BB 084 KB(1,3,1), KB(1,1,3), KB(5,3,1), KB(5,1,3)= 085 (-AA) \*AB\*((2.\*(B\*\*2))\*(1./5.)\*(1.\*4.\*MUA)) 086 KB(1,4,2).KB(>,2,2),KB(8,4,4),KB(10,2,2)= 087 1(BB\*\*2)\*AB\*((4.0/3.)\*(1./(8\*\*2))+(4./15.)\*(1.-MUA)) 880 KB(1,3,2),KB(1,2,3),KB(8,3,2),KB(8,2,3)=((-MUA)\*AB\*AA\*BB) 089 KB(1,3,5),KB(3,3,3),KB(8,5,5),KB(10,3,3)=(AA++2)+AB+((4./3,)+ 09.1

```
1 (6*#2)+(4./15.)*(1.-MUA))
                      KB(2,1,1),KB(Y,1,1)=Ad+(2.*((B*+2)-4./(B**2))~
1146
                     1(1./5.)*(14.-(4.*MUA)))
095
                      KB(2,1,2),KB(9,2,1)=(-BB)*AB*((2,/(B+2))+
044
                     1(1./5.)*(1.-MUA))
045
                      KB(2,1,3),KB(2,3,1)=AA+AB+(-(B++2)+
114.5
097
                     1(1./5.)*(1.+(4.*MUA)))
                      x8(2,4,1), KB(Y,1,2)=BB*((4,7(B**2))*(1,/5,)*
NYX
                     1 (1 . -MUA)) *AB
090
                      (8(2,2,2), KB(Y,2,2)=((BB**2)*AB*((2./3.)*(1./(B**2))*(1./15.)*
100
                     1(1.-MUA)))
1 41
                      KB(2,2,3),KB(2,3,2),KB(3,5,2),KB(5,2,3),KB(4,2,3).
102
                     1KB(4,5,2), KB(6,2,3), KB(0,5,2), KB(7,2,3), KB(7,5,2),
103
                     2KB(Y, 4,3), KB(Y,3,2)=0.0
104
                      KB(5,2,1),KB(5,1,2),KB(8,2,1),KB(8,1,2)=(~AB)=#8*
145
                     ]((2./(B**2))+(]./5.)+(].+(4.*MUA)))
KB(2,5,3),KB(Y,3,3)=AB*(AA**2)*((2./3,)*(B**2)*(4,/15.)
106
147
                     1 . (1 - MUA))
1 48
                      KB(5,5,2),KB(3,2,3),KB(10,5,2),KB(10,2,3)=MUA*AA+BB*AB
149
                      KB(5,1,1),KB(/,1,1)=AB+(-(2,+((B++2)+(1,/(B++2))))+
110
                     1(1:/5.)*(14.-(4.*MUA)))
111
                      KB(5,1,2),KB(/,2,1)=BB+AB+((-1./(B++2))+(1./5.)+(1.-MUA))
112
                      KB(5,1,3),KB(/11,5)=AA+AB*((B**2)-(1,/5,)*(1,-NUA))
113
                      KB(0,1,1),KB(4,1,1)=AB+(-(2,)+(2,*(B++2)-(1,/(B++2)))
114
                     1-(1./5.)*(14.-(4.*MUA)))
115
                      KB(0,1,2),KB(6,2,1)=BB*AB*(-()./(B**2))+((1./5.)*(1.+4.*MUA)))
116
                      KB(0,1,5), KB(4,1,5)=AA+AB+(2,+(B++2)+(1,/5,)+(1,-MUA))
117
                      KB(5,2,1), KB(/,1,2)=BB*AB*(1./(B**2)*(1./5.)*(1.-MUA))
118
                      KB(5,4,2), KB(7,2,2)=(BB++4)+AB+((1,/(3,+(B++2)))
119
140
                     1+(1,/15,)*(1,-MUA))
                      KB(6,2,2),KB(4,2,2)=(BB*+2)*AB*((2,/3.)*(1./(B*+2))*
121
                     1 (4./13.)*(1.-MUA))
142
                       KB(5,3,1),KB(7,3,1)=AA+AB+(-(B++2)+(4./5.)+(1.*MUA))
165
                      KB(3,5,3),KB(/,3,5)=(AA**2)*AB*(((B**2)/3.)+
1124
                     1(1./15.)*(1.-MUA))
145
                      KB(6,3,1),KB(4,3,1)=("AA)*AB*(2,*(B**2)+(1./5.)*(1,=MUA))
126
                      KB(0,5,3),KB(4,3,5)=(AA**2)*AB*((2,/3,)*(B**2)*(1,/15,)*(1,-MUA))
KB(8,5,1),KB(8,1,5),KB(10,1,5),KB(10,5,1)=AA*AB*(2,*(
147
1 48
149
                      18**2)+(1./5.)*(1.+(4.*MUA)))
                      KK(4,1,2),KB(4,2,1)=BH+AB+((1./(B++2)-(1./5.)+(1.+4.+MUA)))
130
                      KB(9,1,3),KB(9,3,1)=4A+AB+((B++2)+(1,/5,)+(1,+(4,+MUA)))
:131
                      IF (VAL. EQ. U) GU TO GOUT
132
                      00 19 1=1.10
153
                 6001
                      DO 18 J=1.6
154
                      00 17 L=1.6
155
                      KE(1,J,L),K1(1,J,L)=0.0
156
                   17
                   18 CUNITINUE
151
158
                      CONTINUE
                      DU 20 1=1.10
159
140
                      KE(1,1,1)=KP(1,1,1)
                       KE(1,1,2)=KP(1,1,2)
147
142
                       KE(1,4,1)=KP(1,2,1)
                       KE(1,4,2)=KP(1,2,2)
143
                       KE(1,3,3)=KB(1,1,1)
144
                       KE(1,3,4)=KB(1,1,2)
165
146
                       KE(1,3,5)=KB(1,1,3)
                       KE(1,4,3)=KB(1,2,1)
147
                       KE(1,4,4)=KB(1,2,2)
148
                       KE(1,4,5)=KB(1,2,3)
149
                      KE(1,5,3)=KB(1,3,1)
150
                       KE(1,3,4)=KB(1,3,2)
131
                      KE(1,3,5)=KB(1,3,3)
152
                 1010 FORMAI(6(5X,F15,7))
135
.154
                  20 CUNTINUE
155
                 MOGE CONTINUE
1.56
                      00=50KT((Y1(Q)=Y1(P))**2+44
```

1141

```
1157
                                     (q) (Y=Y1(U) FY=YY
  158
                                     ZZ=Z1(Q)=Z1(P)
                                     50 44 141.6
00 43 481.6
                                23 47 (1,11,81(1,1)=0,0
  1106
                                24 CHAIINUE
                                  4 CUNTINUE
AT(1,1),A1(4,4),B1(1,1),B1(4,4)B1(0
B1(2,4),B1(5,3),A1(2,4),A1(5,5)WYY/PB
B1(2,3),B1(5,6),A1(3,4),A1(6,5)B2Z/PB
B1(3,4),B1(6,6),A1(3,5),A1(5,6)BP(ZZ/PD)
B1(3,4),B1(6,6),A1(3,5),A1(6,6)BP(ZY/PD)
IF(Z,LB,TPZ) GU TO AZ
  1964
  1967
 168
                               IF(Z.LE.TP4) 90 TO 500
80 TO 5000
25 DO 28 NK=1,6
BU 27 ME1,5
 1471
 19/2
 1974
                                    V.ViEV.O
                                    0966167.6
                               V=V+A1 (NK, L) *KE(2, L, M)
26 V1=V1+A1 (NK, L) *KE(3, L, M)
 1979
                                    KICKINKIMINV
 :480
                               27 KILSINKIMINVI
                               28 CONTINUE
 1161
 1982
                                    DU 31 NKH1 16
                                    DOAUMAT . S
 1945
                              POSUMATIO

RICIRITERO . O

DO 29 LOT. O

RICORITE AT (NK. L) DKG (S. L. M)

RICORITE AT (NK. L) DKG (S. L. M)
 1946
 145
1766
 1147
 1881
                             29 RZHRZ+AT (NKIL) HKE (BILIM)
                                    D3(5,NK,M)=K16
 769
 1140
                                    01 (6, NK, M) #R1
                              30 DE (B, NK, M) BRZ
                               31 CONTINUE
 1193
                               0034 NK#1,5
 144
                                    DU 35 Ma1 . 6
1195
                                    W.WI HU.U
1196
                              DU 32 L=1.6
                                   W=W+K=(7,NK,L)+B1(L,M)
1197
1148
                               32 W1=W1+KE(Y,NK,L) +B1(L,M)
:199
                               KI (Y, NK, M) MW
                              33 K1 (9, NK, M) #W1
34 CONTINUE

0U57 NK=1, 6

0U 56 M=1, 6

R14, R1, R2=U, 0

0U 35 L=1, 6
,200
1201
202
1203
1204
1205
                              R12=R12+03(5,NK,L)+B1(L,M)
R1=R1+D1(0,NK,L)+B1(L,M)
35 R2=R2+02(8,NK,L)+B1(L,M)
1506
1247
1208
1209
                                K1 (5, NK, M) = R12
1210
                                   K1 (6, NK, M) #R1
1211
                             36 K1 (8, NK, M) # H2
                              37 CONTINUE
DO 39 I=1.5
DO 38J=1.5
212
:215
.214
                                  x1(1,1,J)=KE(1,1,J)
1215
1216
                                 K1(4,1,J)=KE(4,1,J)
                             38 K1(10,1,3)=KE(10,1,3)
39 CONTINUE
218
                             42 00 45 NK=1,6
513
3 41)
1241
245
                                 0. u=[V,V.
```

```
263
                       V=V+AT(NK,L)+Kt(7,L,M)
264
                   45 V.=V1+A1(hK,L)*KE(Y,L+M)
225
                       KILL HK MINA
226
                   44 K1 (9, NK, M) = V1
227
                   45 CONTINUE
818
                       0046NN#1.6
220
                       004/ M=1.6
250
                       R12.R1.R2=U.U
231
232
                       6400 641.0
                       RIZERIZEAT (NK. L) *KE(T. L. M)
235
                       R:=K1+A1(NK,L)*KE(4,L.M)
254
                    40 RZ=RZ+A1 (NK, L) + KE(10, L.M)
235
                       DS(1,NK,M) #R12
233
                       D1 (4, NX, M) = R1
237
238
                    47 D2(10, NK, M) = RZ.
                   48 CUNTINUE
239
                       C. PERNICOG
240
                       000 1 M=7 ,6
241
                       W. W1 = U. 0
246
                       DU 49 6=1.6
243
                       W=H+KE(2, NK, L) *89(L, M)
244
                    49 W1=W1+KE(5,NK,L) +81(L+M)
245
                       KI (Z . NK . M) FW
260
                    50 K1 (5, NK, M) = W1
51 CONTINUES
247
                       DU 54 NK=1.6
249
                       DU 55 M=1.6
250
                       R12, R1, R2=0.0
251
                       00 52 4=1.6
252
                       R12=R12+D3(1.,NK,L) 481(L,M)
                       R1=R1+D1(4,NK,L) *B1(L,M)
254
                    52 RZ=RZ+DZ(10,NK,L)#87(L,M)
255
                       KI(1, NX, M) # R12
256
                       K1 (4, NK, M) = R1
257
                    53 K1 (10 . NK . M) = RC
258
                    54 CONTINUE
259
                       DU 56 1=1.5
 200
                       00 55 401.5
201
                       K1(5,1,J)=KE(5,1,J)
 262
                        K1(8,1,J)=KE(8,1,J)
263
                    55 K1(0.1.J)=KE(0:1.J)
264
                    56 CONTINUE
 265
                    SY FURMAL (5X, 'STIF, MAT. TYPEZ')
 206
 207
                       60 TO 1700
                    58 DD 61 L=1,10
 268
 269
                       DO 60 1=1.5
                       00 59 1=1.5
 270
                    59 K1 (L, 1, J) = KE(L, 1, J)
 271
                    .60 CUNTINUE
 2/2
                    61 CUNTINUE
 2/3
                        GU TO 1700
 274
                   300 DUSU4L#1.10
 215
                        DUSUSNA1.6
 216
                        DO 306121.0
 277
                        R1450. U
 2/8
                        00301181,0
 279
                   501 R12=R12+A1(N,J)+KE(L,J,1)
502 01(L,N,1)=R12
 280
 281
 282
                   303 CONTINUE
 203
                   304 CUNTINUE
                        DO308L=1,10
 284
 285
                        DU 30/N=1.6
                        no 3001=1.6
 206
                        R12=0.0
 287
                        nu 3021=1.6
 588
```

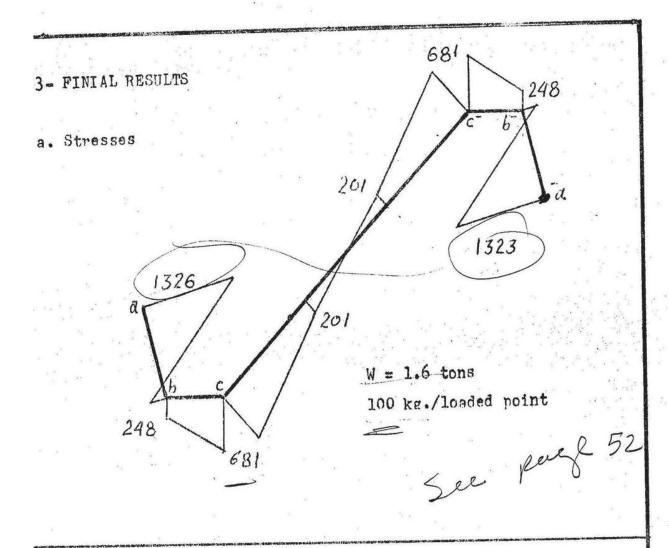
```
505 RIZERIZEDILL, N. J) *** (4.1)
 104
                                     306 K7 (L.N.I)=R12
 240
                                      307 CONTINUE
  291
                                      308 CUNTINUE
  242
                                    1700 CUNTINUE
243
                                               NU 1900 D=1, NBL(Z)
  244
                                               (Sed) riaq
  295
                                               0=12(0,2)
  546
                                               R=13(U.Z)
  247
                                                8314(Vo2)
  246
                                               BUNT FUNZ . BUNS . BUNA . BUMS , FUMS , BUMS . BUNS . BUNS
  多科色
                                                1 F LP . EN , 1 / 40 TO 1446
  1411
                                               60 69 181 1844
  当日本
                                        65 SUNTERAL(1)+SUNT
  144
                                    TOUR CONTINUE
  星山岩
                                                00 66 181,441
  $44
                                         SUNERKI)WAHESUNE
. 345
                                                DU 67 1811 1841
: 300
                                         67 SUNSERAW(1)+SUN3
: 347
                                                DU 68 1 11 1 5 m1
.308
                                         68 SUN4=HAW(1)+SUN4
304
                                                DU 69 1=1 . G=P
  310
                                         69 SUM1=KAW(1+P-1)+SUM1
  311
                                                DO 70 I=1.K-P
.312
                                         70 SUNZERAW. 1+P-17+SUM2
 313
                                                DO 71 121,5-P
.314
                                          71 SUM3=HAW(1+P-1)+SUM3
  315
                                                00 12 181 . K-Q
 315
                                          72 SUM4= NAH (1+Q-1)+SUM4
.317
                                                DU 75 1=1.5-Q
 318
                                          73 SUM5=KAW(I+Q-1)+SUM5
 1810
                                                DO 74 1=1 . R-S
 1340
                                          74 SUMCERAW(1+5-1)+SUM6
 . 341
                                                 1F(VAL.EG.1.0) GO TO 1600
1F(Z.LE.TP1) GO TO 76
 1342
 .323
                                                 1 F (Z. LE. TPZ) GO TO 85
 364
                                                 18 (Z. LE. TP3) GU TO 95
   325
                                                1 F (Z. LE. TP4) NU1 46
 1360
                                                 GU TO 700
  327
                                          76 NUT=5
 .. 3 68
                                                 DO 80 1=1.NU1
   420
                                                 DU 76 J=1.NU1
 330
                                                 KK(SUN1+1,SUM5+1-1+1)=KK(SUN1+1,SUM5+1-1+1)+K9(4,1,1)
   331
                                                 1+1-L=0x
  332
                                                 1F(KO) 78, /8,7/
  . 333
                                          77 KK(SUN1+1,KU)=KK(SUN1+1,KU)+K1(1,1,1,1)
  . 334
                                                 KK(SUN4+1,KU)=KK(SUN4+1,KU)+K1(10.1.J)
 . 335
                                           78 CUNTINUE
  1335
                                                 00 79 327.6
  .357
                                                 KK(SUN1+1,SUM1+J-1+1)=KK(SUN1+1,SUM1+J-1+1)+K1(2,J,I)
  3338
                                                 KK(SUN1+1,SUM2+J-1+1)=KK(SUN1+1,SUM2+J-1+1)+K1(3,J,1)
 J339
                                          79 KK(SUN4+1.SUM5+J-1+1)=KK(SUN4+1,SUM6.J=1+12+K119,1,J)
  :340
                                         80 CONTINUE
  1341 .
                                                 DO 84 1=1.0
  342
                                                 90 81 J=1.NU1
                                           81 KK(SUNZ+1,SUMS+3-1+1)=KK(SUNZ+1,SUM5+3-1+1)+K1(7,3,1)
    343
  344
                                                  DU 83 Ja1 . 0
  . 3 - 5
                                                  KU=J-101
  340
                                                 1F(K0)83,85,82
  -347
                                           82 KK(SUHZ+1,KU)=KK(SUNZ+1,KU)+K1(5,1,1)
  348
                                                  KK(SUN3+1, KU) = KK(SUN5+1, KU)+K1(8, 1, 1)
                                           83 KK(SUNZ+1,SUM4+J-1+1)=KK(SUNZ+1,SUM4+J=1+1)+K1(6,J,1)
  364
  . 350
                                            SA CONTINUE
  . 351
                                                  60 TO 100
  .332
                                            85 NOT=5
: 353
                                                   DU 89-141.841
   .354
```

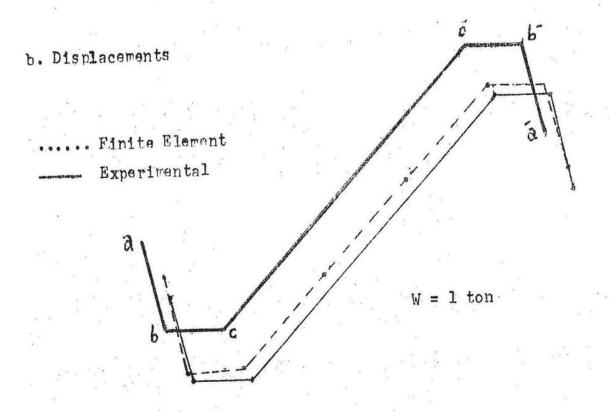
```
DO 87 J=1.NO1
355
.356
                       KU=J-1+1
:,357
                       IF(KU)87,61,86
                    86 KK(SUNZ+1, KU) = KK(SUNZ+1, KU) + K1(5, 1, J)
.358
                       KK(SUN3+1,KU)=KK(SUN3+1,KU)+K1(8,1,J)
359
                       KK(SUNZ+1,BUM4+J-1+1)*KK(SUNZ+1,SUM4+J-1+1)+K1(6,J,1)
:360
361
                       DO 88 J=1.6
                   88 KK(SUNZ+1,5UM5+J-I+1) = KK(SUNZ+1,8UM5+J-I+1)+K1(7,J,I)
352
                   88
U363
0364
                       00 93 141 .6
                       DO 90 J=1.NU1
365
                       KK(SUN1+1,SUM1+1=1+1)*KK(SUN1+1,SUM1+1=1+1)+K1(2,1,1)
366
                       KK(SUN1+1,SUM2+J-1+1)=KK(SUN1+1,SUM2+J-1+1)+K1(3,J,1)
:367
                    90 KK(SUNA+1,8UMO+J-1+1) = KK(SUN4+1,SUMO+J-1+1)+K1(9,1,J)
. 368
                       KO#J-1+1
1309
1:370
                       1 F (KO) 92, 92, 91
                   91 KK(SUN1+1, KU) = KK(SUN1+1, KU)+K1(1,1,1)
.372
                       KK(SUN4+1,KU)=KK(SUN4+1,KUJ+K1(10,1,1)
v3/3
                    92 KK(SUN1+1.%UM5+J-1+1) #KK(BUN1+1,8UM5+J-1+1)+K1(4,J.1)
11374
                       CUNTINUE
                       60 TO 100
376
                   45
                       NUTHS
 371
                      CUNTINUE
                   740
 8 PW
                       00 99 Im1.NU1
4380
                       DO VE JETINUT
                       KUBJalet
山苏西年
                       1 F (KO) Y7, YY, Y6
USUS
                      KK(8UN1+1, KU) #KK(8UN1+1, KU)+K1(1,1,1)
U363
                       KK (BUNK+1+KU) MKK (BUNZ+1+KUJ+K1 (B+1+J)
11384
                       KK(8UN3+1,KU)=KK(8UN3+1,KU)+K1(8,1,4)
                       KK(8UNA+1,KU) = KK(8UNA+1,KU) + K7(10,1)
4366
                       KK(8UN1+1,8UM1+j=1+1)=KK(8UN1+1,8UM1+J=1+1)+K1(2,J;1)
1347
                       KKEBUNT+I, BUM2+J-I+TJ#KKEBUNT+I, BUM24J-I+TJ > KT (3, J, I)
BRED
                       KK(3UNK+1,8UM4+Jm1+1)BKK(BUNZ+1,8UM4+Jm1+1)+K1(C,J,1)
                       KKIBUNK+1, BUMbel = [+1) xkK(BUNKet, BUNbanamin1) + Ki (P, d, 1)
1, 39 B
                       KKEBUN1+1.6UM2+1.141) = KK(BUN1+1.6UM1+1.4K1 + K1 + k1.1+1.1
1369
                      KKIBUNA-1.BUME#Jm147)#KKEBUNA+1.BUME#Jm1473+K4 19.1.13
1392
11343
                    99
                       CONTINUE
                       1 F ( VAL, EQ, U, O) 60 TU 1900
11374
                   140
                      DO 1844 L#1, LP
                  1600
0396
                       00 111 141 ib
                       DS((P-1)+b+1,L),D8((Q-1)+b+1,L),D8((g-1)+b+1,L),D8((8-1)+5+1,L)+0
11397
U398
                       1 F (Z. LE. TP1) GU TO 101
                       IF(Z.LE. TPZ) 60 TO 104
11399
                       IF(Z, LE, TP3) GU TO 107
6400
                       IF(Z.LE.TP4) GU TO 380
0401
0402
                       CETUN
                       DO 103 1=1,NO1
1.603
                       DS((P-1)+>+1,L)=28(SUN1+1,L)
4404
                       DS((S-1) +>+1, L)=25(SUN4+1, L)
0405
                       C. U=U. U
1:400
                       00 104 347.0
U407
                       C#C+81(I,J)*ZS(SUNZ*J+L)
11408
                   102 0=U+B1(1,J)+ZS(SUNS+J+L)
11409
                       DS((Q-1)*5+1, L)=C
: 410
                   105 DS((R-7)+5+1,1)=0
411
                       GO TO 110
6412
                   104 00 100 147.5
1413
                       DS((Q-1)+5+1, L) = 25(SUH2+1:L)
6474
                       DS((R-1)+5+1, L)=2S(SUN3+1+4)
,415
                       C. U=0. U
1410
11417
                       DO 105 JE116
                       C=C+B1(1,J)*ZS(SUN1+J,L)
1418
                   105 000+87(1,J) + Z5(SUN4+J,L)
1419
                       85442-1742+1-F18C
```

```
146 DS1(S-1) *>*1.Lj=0
44.1
                       60 TO 110
0422
                   107 NUTED
463
                       00 100 IFT NOT
1424
                       08((P-7)*5+1, L)=ZS(SUN1+1:L)
625
                       DS((Q-1)+5+1,L1=2S(SUN2+1,L)
.425
                       DS((R-1)*5+1, L) = 25(5UN3+1+L)
467
                   1 U8 DS((S-1) +5+1, L) = 25(SUN4+1, L)
 428
                       60 70 110
 420
                   380 00 400 1=1,5
430
                       C.U.CU.00=U.0
.451
                       DU 390 J=1.6
432
. 433
                       C#C+8:(1,J) #ZS(SUN7+J.L)
                       0 = U+B ( ( 1 , J ) + Z S ( S UN 2 + J , L )
.434
                       CC=CC+81(1,1)*LS(SUN3+1,L)
1435
                   390 00=00+81(1,J) #ZS(SUN4+J,L)
436
                       ns((p=1) + 5+1, L) = C
437
                       05((Q-1) w5+1, L) #0
 438
                       DS((R-1)+5+1, L/=CC
 439
                   400 DS((S-1)*3*1,L)=00
                   110 CONTINUE
.. 641
                       $1(P,L),S2(P,L),S3(P,L),S4(P,L),$5(P,L),S6(P,L),
                      151(Q,L), 32(Q,L), 53(Q,L), 54(Q,L), 55(Q,L), 56(Q,L), 51(R,L), 82(R,L), 53(
                      2(R.L), $4(R.L), $5(R.L), $6(K.L), $1($,L), $2(8,L), $3($,L),
                      3$4($,L),$>($,L),$6($,L)*0.
1445
                       p1=85(1p-11+5+1,L)
: 446
 447
                       p2=DS((p-1) +5+4, L)
                       03=05((Q-1)+5+1,1)
 448
:440
                       PS=DS((R-1)+5+1,L)
                       P6#US ((R-1) #5+2, L)
 651
                       p7=DS((S-1)+5+1,L)
                       p8=US((S-1)=5+2, L)
1633
                       BE1=05((P-1)+5+3,L)
 454
                       BE4#DS ( [P=7 ] + > + 4 + 4 }
 425
                        BEJ=DS((P#1) #9+5, L)
A 96
                        ##4#D$ ( (Q=1 ) +3+3 , L)
. 437
. 438
                        BE2=05((G=1)+9+6, b)
                        BE0#08((Q=1) #3*5, L)
4459
                        RE/=DS((R=1)*9+3,L)
1:460
                        BEUSPS ( (R=1) *5+6, L)
469
                        BEY#08((R=1) #5+5, L)
462
0663
                        8811 aD8 ((8=1) *5+4, L)
 1404
                        88720UB((Sm1) +3+5,6)
 465
 406
                        825=##3
467
: 468
                        8 6 3 8 - 4 6 5
4669
                        BENAMUEH
                        DE11 ## BE11
1672
                        BE12="8E14
                        91(P, L)=(-8B+P1-MUA*AA*P2+MUA*AA*P4+BB*P7)*AS+81(P, L)
.474
                        52(P, L) = (-MUA+88+P1-AA+P2+AA+P4+MUA+88+P7) +A5+82(P, L)
                        $3(P,L)=(-(1.-MUA)+(AA/2.)+P1-(1.-MUA)+(BB/2.)+P2+(1.-MUA)
 476
                        +(AA/2.)+PS+(1.=MUA)*(BB/2.)+P8)*AS+83(P.L)
 477
                        $1 (Q, L) = (- (MUA+AA+P2) - (BS+P3) + (MUA+AA+P4) + (BB+P5) ) +AS+31 (Q, L)
478
                        $2 (Q, L)=(-AA+P2-MUA+88+P3+AA+P4+MUA+88+P5)+A5+$2(Q, L)
 479
                        $3(Q,L)=(-(1,-MUA)+(AA/2,)*P1+(1,-MUA)+(AL/2,)+P3-(1,-MUA)*
 480
                       1 (88/2.) +P4+(1. -MUA) + (88/2.) +P8) +A5+53(Q.L)
:481
                        $1 (R, L) = (-88+P5+B8+P5+MUA+AA+P6-MUA+AA+P8) *AS+$1 (R, L)
1.682
                        $2(R, L)=(-MUA+88+P5+MUA+88*P5+AA+P6-AA+P8) 445+82(R, L)
. 483
                        $3(R, L)=(-(1, -MUA) +P4*(88/d.)+(1, -MUA) +P5*(AA/d.)+(1, -MUA)
 454
                       1 + P5+ (88/2.) - (1. - MUA) + P(+ (AA/2.)) * AS+83(R, L)
 905
                        . 486
```

```
52(5, L) m (-MUA+68+P1+AA+P6+MUA+68+P7-AA+P8)+A5+82($, L)
                       $5($,L)=(=(1,=MUA)+P2=(BB/2.)+(1,=MUA)+P5+(AA/2.)=(1,+MUA)
1+P/+(AA/2.)+(1,=MUA)+P8+(B8/2.))+A8+g3($,L)
.488
1489
                        $4(P,L)=((0.+B*DX+(0.*DU1/B))*BE1=4.*AA*DO1*BEZ+
.490
4491
                       14. *B6*UX+BE3-(0, *DU1/B) *BE4-2. *AA*DU4*BE5-
                       26. *U*DX*BE10+2. *88*DX*BE12)*(1./AA+88)+$4(P.L)
.492
                        $5 (P, L) = ( (6. *DY/B) +6. *B*DU1) *BE1 ~4. *AA*DY*BE2+
0493
                       14. *88 * DU1 * BE3 - (6. * DY + BE4/ b) -2. *AA * DY + BE5
0494
0495
                       2+0. +5+001+8E10+2. +88+001+8E12)+(1./AA+88)+$5(P,L)
                        So(P,L)=(-2.+DXY+BE1+4.+88+DXY+BE2-2.+AA+DXY+
1.490
                       1865+2.*DXY*864+2.*AA*PXY*860-2.*DXY*8E7
2+2.*DXY*8E1U-2.*RB*DXY*8E11)*(1/AA*BB)*S6(P.L)
11497
U498
                        $4(Q,L)#(=(6.*001*BE1/B)+d,*AA*DQ1*BE2+(6.*B*DX+(6.
11499
                       1+001/82)+BE4+4.+AA+001+BE3+4,+BB+PX+BE6+6.
4500
                       2+6+DX+BE7+2.+BB+DX+BEY)W(1./AA+BB)+54(Q,L)
U501
0502
                        SH(Q.L)=(-(0,+UY/B)+BH1+2.*AA*DY*BE2+((6,*QY/B)+
                       0503
                       2-6. *B*D01*HE7+2. *BH*DU1*BEY3*(1./AA*BB)*$5(Q,L)
86(Q,L)*(-2.*PAY*BE1-2.*AA*DXY*BE3+2.*PXY*BE4*
0504
0509
                       12.*88*UXY*HE5+d.*AA*OXY*B&0*2.*0XY*867
2-2.*88*DXY*BE8+2.*UXY*BE1U3*C1./AA*88)+86(8,L)
U546
1507
                       $4(R,L)=(~0.*8*DX*864~2.*8#*DX*866*0.*(8*DX*(801
1/8))*867*4.*AA*DO1*868~4.*88*PX*869~(6.*001
0508
4500
                       2/b)+8t10+2. +AA+DO1+BE11)+(1./AA+BB)+44(R,L)
$5(R,L)=(-6.+B+D01+BE4-2.+BB+D01+BE0+6.+((DY/
J5111
1511
                       18/+8*UU1)*BE7+4. *AA*DY*BE8*4. *BB*D01*869
.512
                       2-0.+(UY/8)+BE1U+2.+AA*DY*BE11)+(1,/AA*BB)+85(R.L)
50(R.L)=(-2.+DXY+BE1+4.+DXY*BE4+2.+B8*DXY*BE5
1-2.*DXY*BE/-2.*BB*DXY*BE8+4.*AA*DXY*BE9+2.
313
3514
1515
                       0516
317
:518
.519
1520
. 321
                       644+DY+BE11=4. +BB+DU1+BE123+(1./AA+BB)+$5(8.4)
1362
                        86($,L)=(-2,*0XY*8E1+4,*DXY*88*8E2+2,*DXY*8E4-2,*DY
.1523
                       1+BE7+4.*DXY*AA*BE9+2.*DXY*BE10~2.*BB*DXY*BE11*2.
0564
                       2+AA+DXY+BE12)+(1,/AA+BB)+S6($,L)
1.545
u526
                        WRITE(4,109)P.Z.D.$1(P.L).$2(P.L).$3(P.L).$4(P.L).$5(P.L).$6(P.L)
                        WRITE(6,104)Q. &. D. $1 (4, L) . $2 (4, L) . $5 (Q, L) . $4 (Q, L) . $5 (Q, L) . $6 (Q, L)
1527
                        WRITE(4,1UY)R,Z,D,S1(K,L),S2(R,L),83(R,L),84(R,L),85(R,L),86(R,L)
0528
                        WRITE(2,104)S,Z,D,S1(S,L),S2(8,L),S3(S,L),S4(8,L),S5(8,L),S6(S,L)
0529
0530
                    109 FORMAI (3(3X,12),6(3X,F13./))
                        po 6007 1=1.6
531
1532
                        Y5, Y2, Y3, Y4=0.
                        00 6008 J=1.6
0533
11534
                        Y5=Y5+(K1(1,1,1)+Z5(SUN1+1,L)+K1(Z,1,1)+Z5(SUN2+1,L)+K1(3,1,1)+Z5
                       1 (SUN3+4, L)+K1 (4, J, 1) + 25 (SUN4+J, L))
U535
                        Y2=Y2+(K1(2,1,1)+ZS(SUN1+J,L)+K1(5,1,1)+ZS(SUN2+J,L)
11536
                       1+K1(6,J,1)+ZS(5UN3+J,L)+K1(7,J,1)+Z5(5UN4+J,L))
0537
                         Y3=Y5+(K1(5,1,1)+ZS(SUN1+J,L)+K1(6,1,1)+ZS(SUN2+J,L)
.. 538
                       1+K1(8,1,1)*ZS(SUN3+J,L)+K1(9,J,I)*ZS(SUN4+J,L))
.539
                        Y4=Y4+(+K1(4, I, J)+ZS(SUN1+J, L)+X1(7.1, J)+
1540
                       125(SUNZ+J,L)+K1(9,1,J)+Z5(SUN3+J,L)+K1(10,1,J)*Z8(BUNA+J,L))
1.541
542
                   6008 CONTINUE
                         F(SUN1+1, L)=Y5+F(SUN1+1, L)
 545
                         F(SUNZ+1, L) = YZ+F(SUNZ+1, L)
:544
                         F($UN3+1,L)=Y3+F(8UN3+1,L)
2545
1.546
                   6007 F(SUN4+I,L)=Y4+F(SUN4+I,L)
0547
                   1800 CONTINUE
U548
                   1900 CONTINUE
549
                   2000 CONTINUE
.550
                         TF(VAL.EQ.U.O) GO TO 150
U551
                         15UN#U
```

```
18(1, Lu. 1) CO TO 15
                      18UH=KAW(1-1)+15UN
57 -
                   15 WRITE(2,10) 1 . (ZS(1SUN*J.1).UNT. RAW(1))
555
                   16 FORMAI(5X,12,0(3X,112,/))
556
                      1507=7
.537
                      Lawerst Es no
35%
                      1811,68,1) no in al
524
                      ISUNARAW(I-1) +15UN
500
                   21 WELTE (2,16)1, (+(1506+4,1), 0#1:RAW(1))
501
                      go to buny
.502
                  130 CUNTINUE
303
                       02 131 S=1.N6JFX
504
                       (C) HK=H
 505
                       1F(H. EQ. 1) GO TO 152
 504
                      SON=0
00 153 1=1,H=1
 501
.568
                  133 SUH=SUH+RAW(I)
509
                  132 DU 130 I=1. RAW(H)
. 571.
                       IFCRL(S, 1)) 134, 134, 134, 135
 3/1
                  134 ZESOH+1
.512
                       DU 150 Ja1 BANDE
 375
                       KR(2,31=0.0
 574
                       1+(3,61.2) 60 10 155
 515
                       KK (2-3+1, 3)=0. U
 510
                   . 15 CUNTINUE
 517
                       KK12.11=1.0
 5/3
                  136 CONTINUE
137 CONTINUE
 579
.580
                       DO 140 5=1. NBJ+X
.581
                       H#HH(S)
. 382
                   140 WRITE(2,159)H, (RL(S,1),127, RAW(H))
                   159 FURMA: (12x,12,9x,2(11,4x),11,5x,11,6x,11,6x,11)
 583
584
                       00 143 JaliLP
. 385
                       DU 144 1=1.50H
:566
                   142 55(1,1)=0.0
 55/
                   145 CUNTINUE
.588
0589
                       00 14/ K=1, LP
                       READ(1,1)NLJ
. 590
                       REAU(1,5)(JL(1),1=1,NLJ)
. 591
                       READ(1,2)((82(1,1),0=1.6),1=1.8:5)
 1592
                       DU 140 1=1.NLJ
 543
                       STEUL(I)
 .576
                        SULZU
 5,45
                        DU 144 L=1.57-1
 .546
                   144 SUL=SUL+RAW(L)
 597
                        DO 140 J=1, HAW(ST)
  619
                   145 SS(SUL+J, K) = SZ(1, J)
 347
                   145 CONTINUE
 640
                   147 CONTINUE
 601
                   148 FORMAL (141, 10 (4X, F/. 5))
..602
                        WRITE(2,148) ((SS(1,J),1=1,SOR),J=1,Lp)
 $115
                        CALL BAND MAT (BANDO, SUR, LP)
                                                                             BAND
 644
                                                                      15
                        VAL=1.0
. 5115
                        60 TO 1000
 000
                  SOOO CONTINUE
.507
                        STUP
 1608
                        ENU
 .609
```





	Test of the second		2 2		
1 -224.0205550	-/6,0007/46	35.1964711	-0, 7810742	-0.0803647	-2.4206055
1 32.4498818	10.0166104	182.2091432	0.9252/12	0.0603647	*4.1303386 *3.8910138
1 -209.4507498	-15,4813022	149.0846142	P140,1090107	-0.5154683	149.6223042
2 112.032/02/	91.880/412	-106,5105954	~5.1092738 ~6.7643372	-0.4263432 8.2117251	"2.7814267 41.8769178
2 145.0377358	189.0935381	322.2116248	-16.9539375	-8,2378353	+1.9910382
3 6987.9131181	-/3,0115735	301.3393015 -231.4824934	-253,2770029 -15,7507235	0.8767480	42.1124960
3 192.8767465	205,0589456	*252.1402288	-25.0453284	-8.1022397	45.2807092
3 191.3530130	-171.0205037	359,3762565	*12,5249089 *310,6194285	-0.8732519	361,8308953
4 -1.119.2167640	~268.3810645	-325.7344863	-13.4370542	-1.0493761	*2.8376173
4 222.4101145	282,71005476	-511,5868360 434,6551361	-17.9804997 -22.1776478	9.0223697 +6.2899793	0.1649426
5 -1195.6994720	-197,8309182 -256,1679243	449,3074839 -402,8071674	*308.6343541 *19.9663261	0.9857496	*0.8725929
5 261.3131313	287.8594640	*410.8647809	+33,4363438	-6,2147486	43,3061902
9 248.8037358	-2/3,0961335	402.6574376	+15.2449136 +351.8085885	9.3369330 -0.9898163	471 9806119
442.5407571	-14,1759/49	~28.8672283	-0.9918699	WG. 3512896	0.8993922
1 0104.3/34085	16, 3281/00	49,573/938 450,466/071	2,9814399	-2,2938918	10.7099783
2 -85.46/3573	43,3942660	\$3,7601417 5,0657202	36,2030344	18,9389435	4.3922708
2 -348.2931053	-04,4643010	5.0217909	30,8873538	-1.5549089	2,1911204
2 -347.3018746	*62,/106066 84,/1/808/	#81.0472923 #81.8033630	801,4259285	7,7834122	11398, 9366699
\$ -148.110/602	3.7502851	13.6299746	103.3165107	7.2898595	1.8938011
3 -532.8978341	+124,4965446	-61.0674860	138,9055607	72,4125000 44,5531952	14.0629777
3 120.90/8826	80,4030000	*88.7962666	1079.5501219	37.8963794	-1900,7383203
4 -141.7105684	72,0886.58	50.793/027	07,2916902	4,6360921	4.2077280
6 *637.9203901	44,7590>80	-103.8461900 -04.4703429	1204,7824302	76.3743930 13.0634336	-2238 9239374
5 -191-9/00019	37.9071094	80.1701622	122.7304028	19.0811364	. 2.4771833
5 -699./410121	-151,4808581 -76,1546646	49.5730909	110,6021541	76.3389917 *3.5372858	10.4643349
5 -173,1343540	93.8333009	-85,1339348	1340,9164396	41,4263716	-2390 7872199
1 123.3431360	14.1199731	~9.373/933 ~28.8572280	0.0189321	0.6623793	66.8993964
1 104.3732080	-43,5442045 -16,3281/07	-99.7601A11 -36,660/063	34.3112074	46.9588648	-735 4989970
2 348.2931036	64.9643805	5,8217915	50.0733836	+3.5569043	- 62.4941207
2 89.46/3585	-22.4641149	-81.8037208	106.2572795	7,5695735	*0.7571357
2 347.9018768	02./100099	-81.0472916	840,0566221	71.7854219	-1373.4999886
3 168.110/622	124.4905440	\$6.3383358 \$8,6299754	105,6489276	72.4125997	•14.0629796 •1.8938025
3 120,56/884/	41.84.58.2YD	-88,7962632 -61.067888	134,2212604	37,8965831 -4,5531897	*2.0766406 *1893.7890894
6 628.6105261	5440042	50.7937038	86.8410313	-4.8366868	218768099
6 18117108709	- 72,0488652 -44-/89056	-04,4783445	154,7912103	38.0383318	- 0,8077896
637.928393	110.8786714	-103.846186V -08.7310478	1304,4894868	76.3744055	-10.493247B
5 699.7410162	"37 . 9071 U92	80.1701835	192,7792427	15.0811386	.2.4771847
5 681.208471	76,1366629	~88.1339547 ~64.5730904	128,3935277	-3,5372720	*0.3179106 *2318.1918797
4 432,4498822	010.4166106	46, 6610218	0.1810943	0.0603847	4.1305388
269.8205535	15,4813968	169.0846142	0.3629734	"0,0603647 "0.5156683	2,4206037
2 -52.8236897	-/1, 44ABB53	182,2091450	#414,4614301 #4,7719547	4.3008501 4.2117251	145.5424577
2 675.4746470	1/0.824113	-100,5105980	-7,2880232	WO.4263A33	2.7614267
2 -145.9399381	73,9115156 -189,0939568	361,3393029	-15,3439350 -225,9860494	0.8003437	271,2757655
3 -192,8761479	~203, V389860 188, >5/1011	-232,1402308	+16.5471494	78,1022386	5.2807087 2.1124966
5 980.9342475	191 6209037	359.5762338	**2,9086921	-0.8732517	1.6085783
4 47.255.44	-201.V(55238 -212.1603460	358,9185180 -341,5868331	~276,3011568 ~11,9965369	9.0223678	0.4303043
4 1219,2101904	268.3810612	~350. PSA4859	#20.2169208	-1.0493758	2,8376170
4 1199.6994595 4 -245.9208051	147.83091/3	449.3074798 444.4551298	-19.7405356	0.9857494	427.7545685
5 -265.9208051 5 -261.3121317 5 1310,9106026	-207.8594607 256.16/9466	-410.864/576 -402.8071637	-41.9520050 -30.3535994	0.9105186	3,3061903
5 1523.2149582	273.0961506	410.7155256	-15,8598338	.0.9808160	·0.5486926
5 -268.003/382	14.112567	402.6579319	*0.9938699	9.350355	01.9851490
7 -62.546/677	** **14.1135769	-03,254445	0.0189521	-0.3312896	1.9851468
42,340/6/4	14,1155/52	~121.304V592 ~124.304V301	412.8506613	12,3319849	1,9851468
2 105.1312862	\$5.0457467 -85.0457470	-60,4790972 -60,4790973	94.6673770	12,9625948	9.6845596
2	-35.0457272	-130.1220702	109.0708823	12,9623035	5.6845599
2 105.1372861 3 68.0379266	35.0457663	-130.1220701 -9.668/325	772.0928508 108.1301180	16.0352707	+1380.5285222
5 "168.0379189	-20.0132208	+9.688/326	908.4445234	15,7216797	-2.0038823
3 168.0377190	-30.0152912 30.0152901	-120.9984311 -120.9984311	131.1601091	28.7131186	-1877.9375849
6 504 . 1854 064	69.1215000	27.9613853	131.5853562	28.8548768	04.8256902
4 -209.18490AF	-64.7575000	27.9613857	131.7496500	28.8548670 24.1113569	4.8256885
4 204.1854035	09.12/5009	-110.6014407	1249.1709610	26.1113577	-2205.8291297
5 -229.7507710	-76.5795149	64.0450843	135,7663742	24.1290583	3.1321572
	- 24 - 344-6 9 411		ANE TROPERRY	SOATTAGE	
5 229./38/741	76.2/95149 76.2/95140	-88.1339548 -48.1339347	135,3025404	32.2734065	-3.1321579 -2306 4753468

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CHAPTER V
CONCLUSIONS

The results presented in this thesis illustrate the importance of careful study of the experimental tests before developing an analytical method of solution. The experimental studies provided the following conclusions for the Saw-Tooth folded plate structures:

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- 1. Both Aluminum as well as reinforced concrete models are required for studying the structural action of hipped plate roofs. Aluminum is caracterized by straight line stress-strain relationship while the reinforced concrete model indicates the behaviour of the real structures.
- 2. Due to the presence of wide plates in the Saw-Tooth folded plate structure, the behaviour is not far from that of the beam analysis.
- 3. The effect of the transverse deformations should be taken into consideration specially when the resultant of loads are far from the shear center of the cross section.
- 4. The theoretical strains based on Gaafar's analysis are very close to the experimental results for both aluminum and reinforced concrete models all over the domain. The tensile strains which occur

In the reinforced concrete models are buch bigger than the theoretical .

- 5. The deflected corss sections under loads are similar to that concluded by Gaafar's method. The aluminum model gives closer results.
- 6. Such roofs are vary sensitive to the cross sectional dimensions, small change in this section causes great changes in the analytical results.
- 7. For thin walled sections, Gaafar's Mathod provides a good tool to determine the stresses due to both symmetrical or unsymmetrical loadings.
- 8. The finite element method offers a coloured method for researchers. Designers may benefit the results after checking it using any pre-tested model to have an idea how much the results deviate from the true values.

## Suggested Further Studies on Saw-Tooth Folded Plate Structures:

1. Similar shapes with hollow blocks intermediate plate.

- 2. Prestressed models
- 3. Similar Shapes with variable intermediate plate length .
- 4. Other shapesof the Saw-Tooth folded plates .

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- 5. Different span/ width ratios
- 6. Continuous structures .

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